



Optimization of a spool-geometry for jet force compensation in a pressure control valve



C. Hugel (M. Eng.) | Hilite Germany GmbH | 28.06.2016

Motoranwendungen

Getriebeanwendungen

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What is Hilite International GmbH?

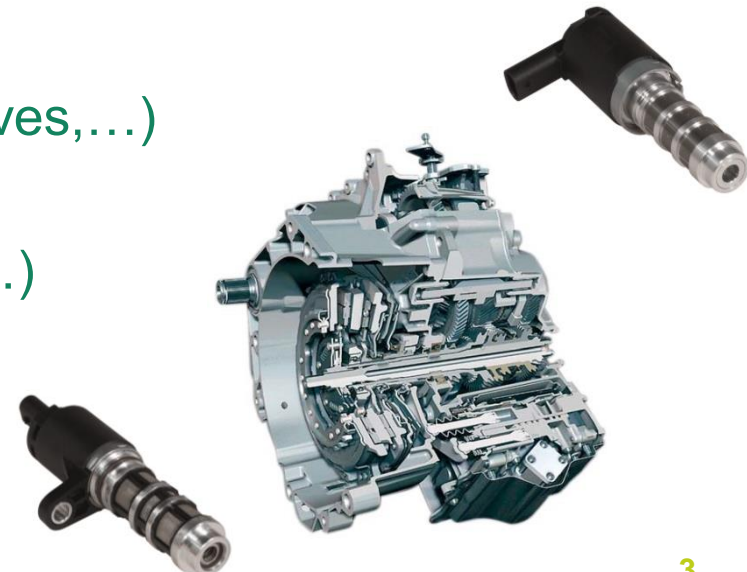
Everything important about us

Information

Hilite International GmbH

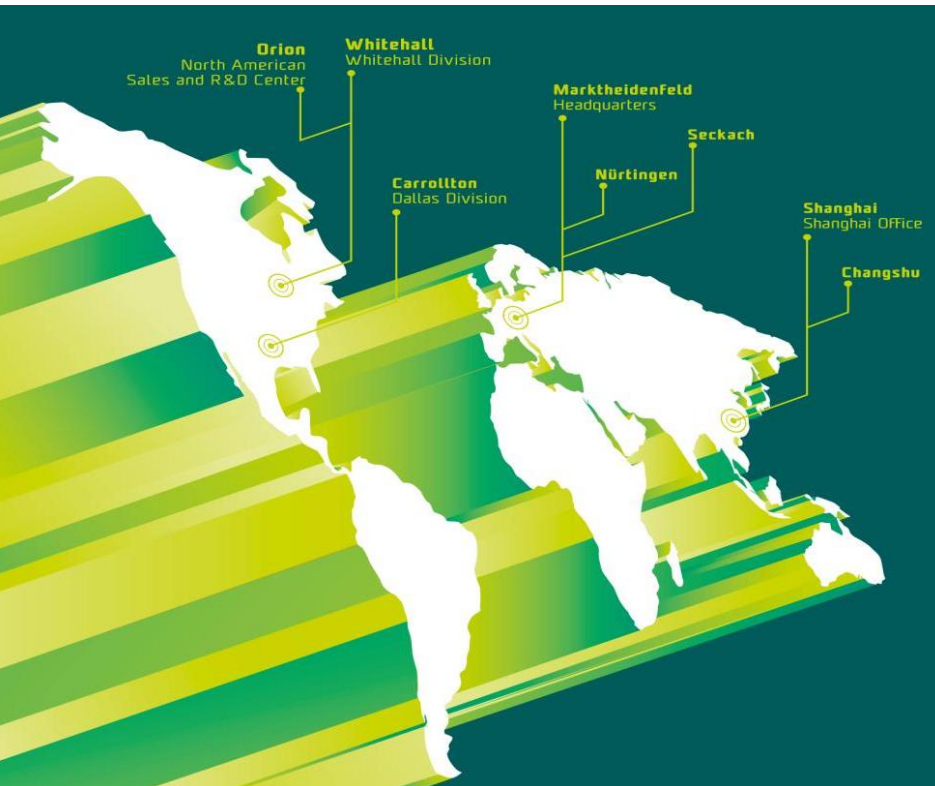
Key facts about Hilite

- Foundation of Hilite (Heller Hydraulik) in 1930
- Products for the automotive industry divided into two fields:
 - Transmission (AT valves, DCT valves,...)
 - Unit Engine (VVT, engine valves,...)
- Approx. 1500 employees

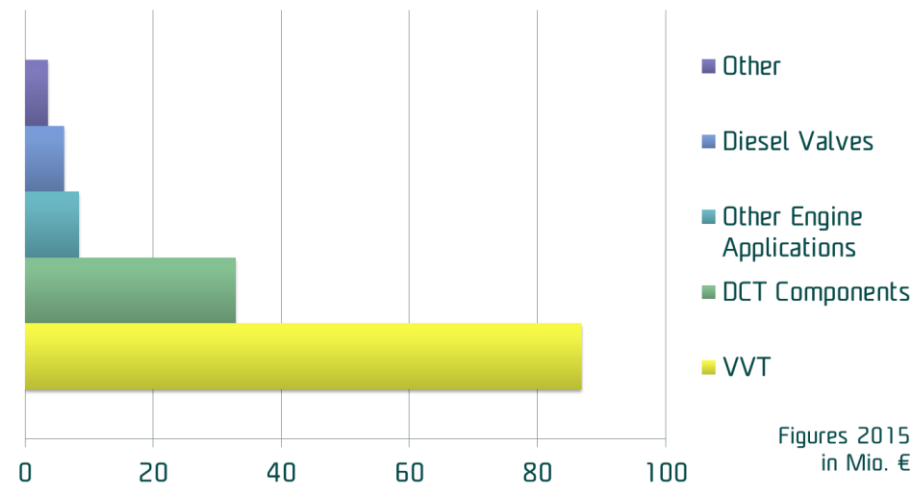
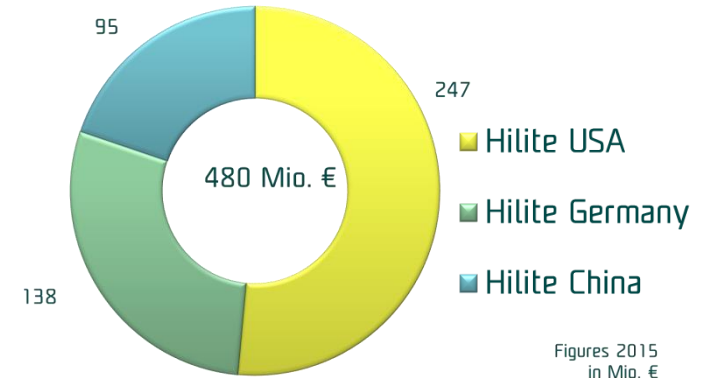


Information

Hilite International GmbH

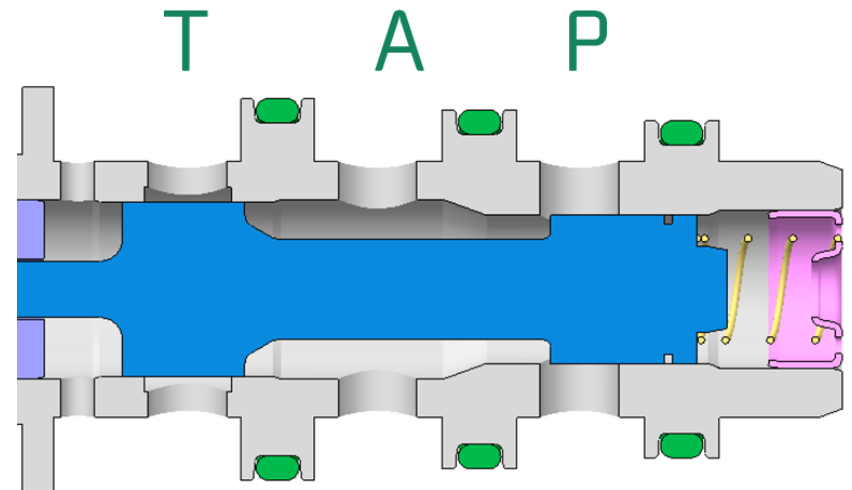


- Headquarters: Marktheidenfeld
- 8 Locations on 3 continents



What had to be solved?

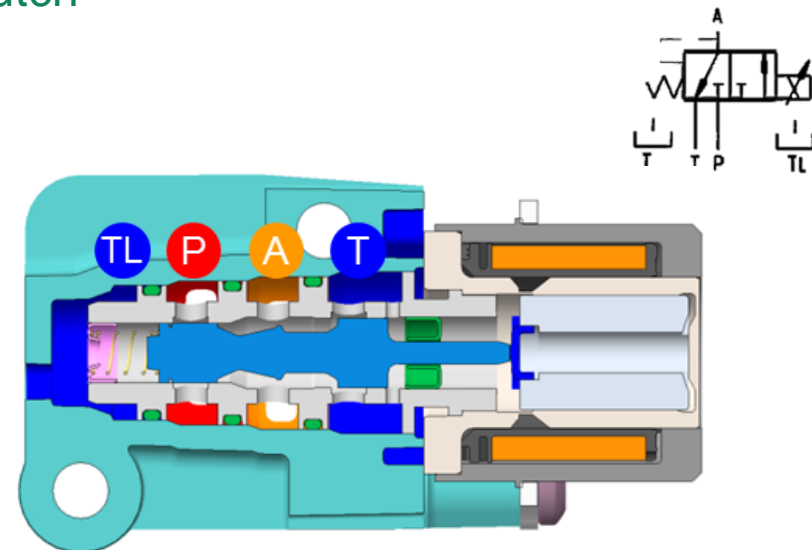
Introduction to the product and its problem



Introduction – response time

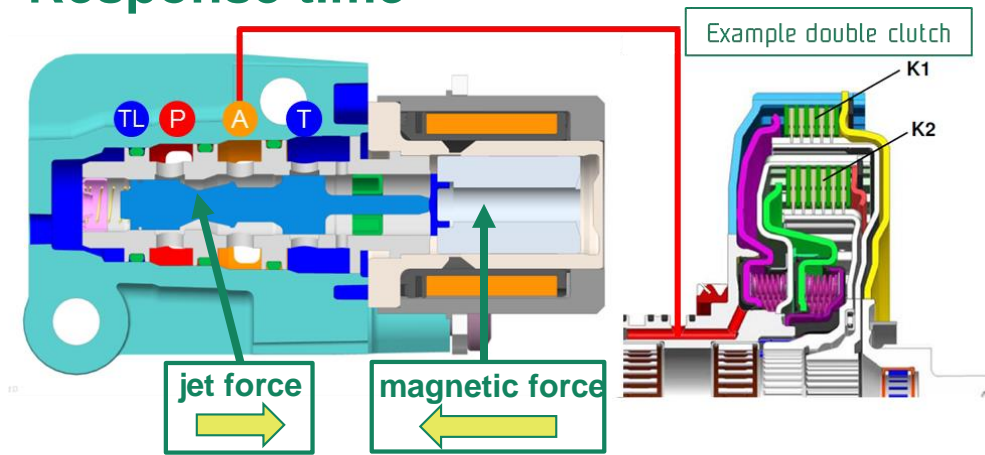
Pressure control valve for DCT

- 3/2 proportional valve
- Regulation of the inflation of the clutch
- Low hysteresis and leakage
- Control pressure: 0 – 15bar
- Supply pressure up to 20bar

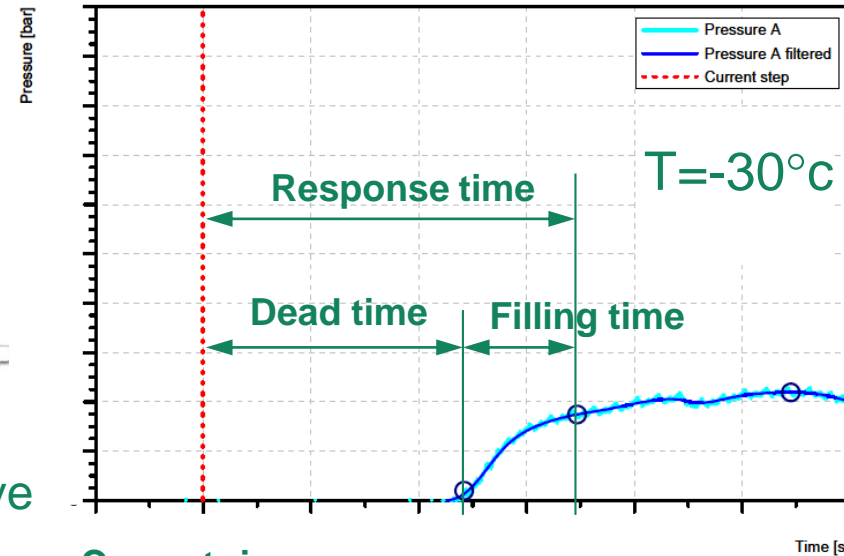


Introduction

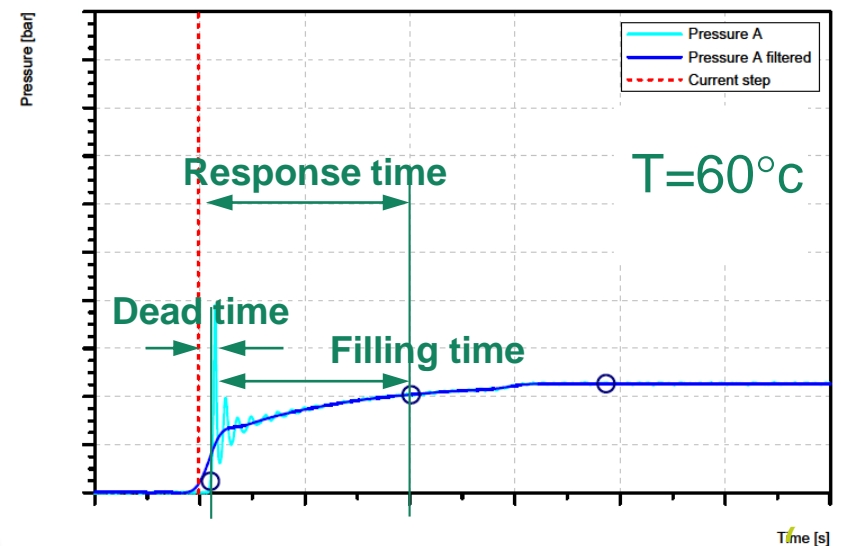
Response time



Current rise



Current rise



Actuation of the clutch with pressure control valve

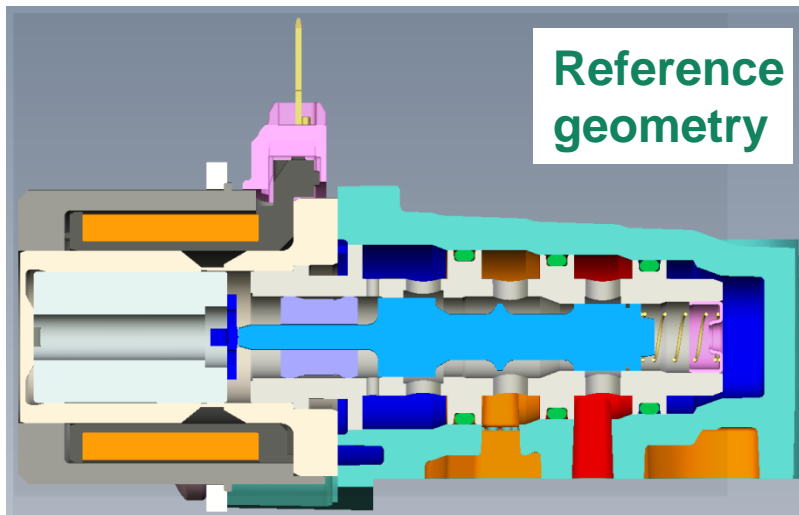
- **Response time:** Time between current rise (actuation) and achieving the target pressure (till 90% of target pressure)
- **Dead time:** Time between current rise (actuation) and pressure rise (<10%)
- **Filling time:** Time between pressure rise and achieving 90% of the target pressure

Introduction

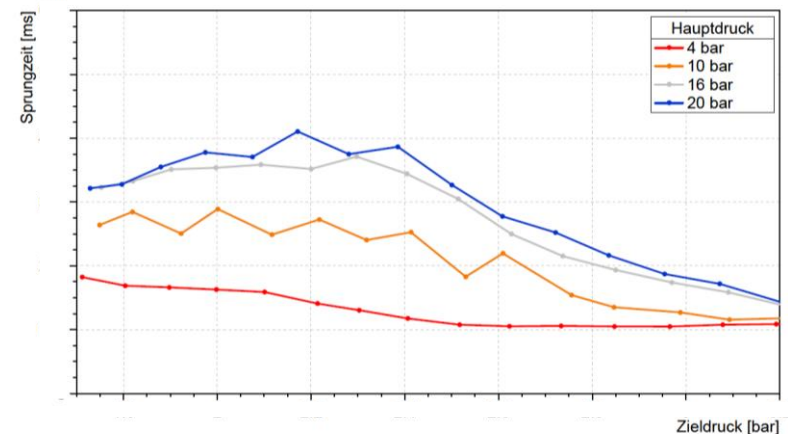
Results reference system

- max. step response at 20bar supply pressure of 420ms
- pressure dependence within a band of 250-300ms

→ Slow response times lead to cogging during gear change



Reference step response



→ Reduction of the step response by optimizing jet force

Solving the problem!

Optimization with OptiSLang and ANSYS CFX

Solving the problem

Methods and tools

- **CFD: Computational Fluid Dynamics (simplified)**

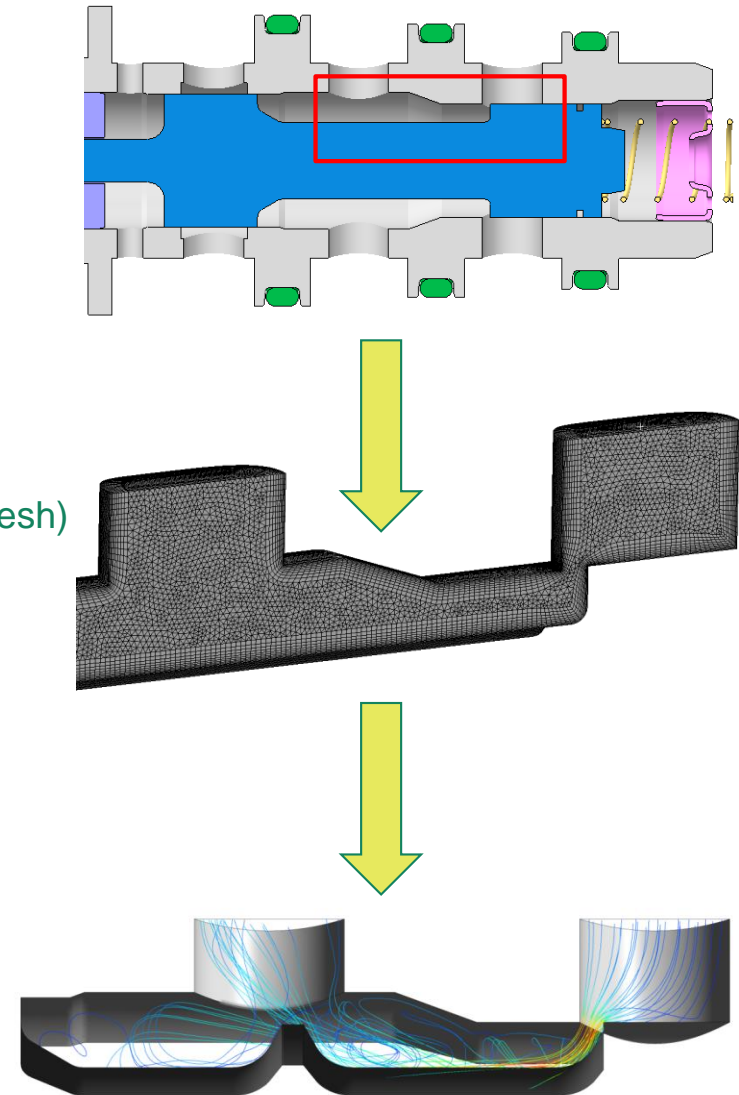
- Numerical approximation of fluid dynamic problem (Mainly used Navier-Stokes or Euler-equations)

- Method: Finite Volume

- Separate the area into discrete elements (mesh)

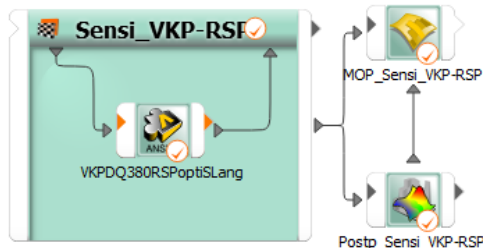
- Solve the model equations for every element over all surfaces

- **Simulation of the oil flow through the valve with ANSYS CFX**



Solving the problem

Methods and tools



Parametric System

- Basic System to build sensitivity and optimization
- Connection with Workbench and ANSYS CFX

Sensitivity analysis

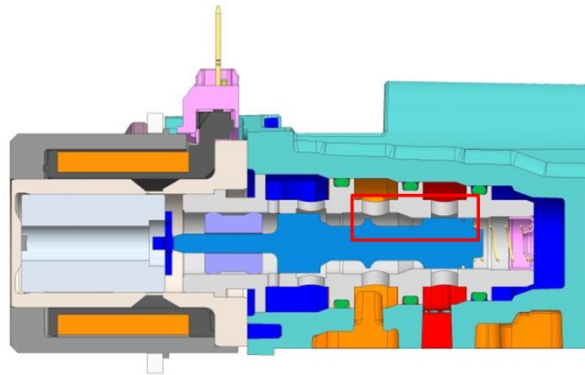
- Advanced Latin Hypercube Sampling (DoE) with 200 designs
- Analysis of the important parameters
- Start designs for optimization

Optimization

- Multidisciplinary optimization with two objectives
- Evolutionary algorithm with pareto ranking (start size 20, archive size 20, number of parents 10)

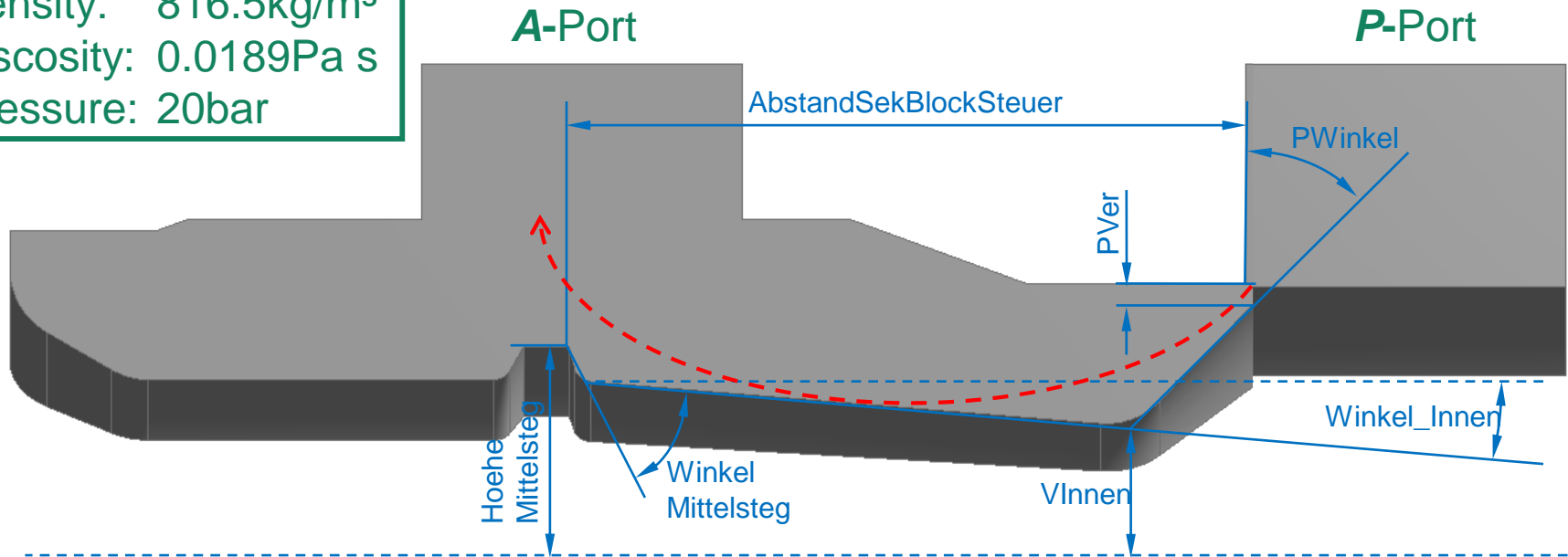
Solving the problem

Model information



Density: 816.5 kg/m^3
Viscosity: 0.0189 Pa s
Pressure: 20 bar









- Edited CFD-Model to fit **jet force** calculation with optiSLang
- **7 adjustable spool geometry parameters** (see figure)
- **Different spool positions** (moving the lower part of the model to the right, with A- and P-Port staying in place; figure orifice 0.1 mm)



Solving the problem

Parameters, constraints and objectives

- Variation of all important **geometry parameters** within a defined range
- Input pressure 20bar** at P-Port
- Different constraints to ensure correct geometries (not listed)
- Calculated spool positions (orifice at P-Port):
0.025mm, 0.05mm, 0.15mm, 0.40mm
- Objectives (optimization targets):
 - Reduction** of the spools **jet force** (Obj_ForceMIN)
 - Increase** of the valves **flow rate** (Obj_FlowMAX)

Parameter	Start designs	Criteria	Initialization	Selection	Crossover	Mutation	Other	Result designs
Name	Parameter type	Reference value	Constant	Value type	Resolution	Range	Range plot	
1 VInnen	Optimization	2.25	<input type="checkbox"/>	REAL	Continuous	1.5 3		
2 Winkel_innen	Optimization	0.1	<input type="checkbox"/>	REAL	Continuous	0.01 15		
3 Winkel_Mittelsteg	Optimization	60	<input type="checkbox"/>	REAL	Continuous	45 80		
4 AbstandSekBlockSteuer	Optimization	7	<input type="checkbox"/>	REAL	Continuous	5.5 7.5		
5 HoeheMittelsteg	Optimization	3.25	<input type="checkbox"/>	REAL	Continuous	2.5 3.85		
6 PressureP	Optimization	2e+06	<input checked="" type="checkbox"/>	REAL	Continuous	450000 2e+06		
7 PWinkel	Optimization	30	<input type="checkbox"/>	REAL	Continuous	25 85		
8 PVer	Optimization	0.3	<input type="checkbox"/>	REAL	Continuous	0.1 0.5		

Name	Criterion	Expression	
Obj_ForceMIN	MIN	(-ForceSpoul0025)+(-ForceSpoul005)+(-ForceSpoul015)	3.52129
Obj_FlowMAX	MAX	(-FlowrateA0025*8e5)+(-FlowrateA005*3e5)+(-FlowrateA015*1.7e5)+(-FlowrateA04*1e5)	64.6387
new			



Improve of the response time to ensure, that the clutch is filled with an acceptable speed.

Results - Sensitivity

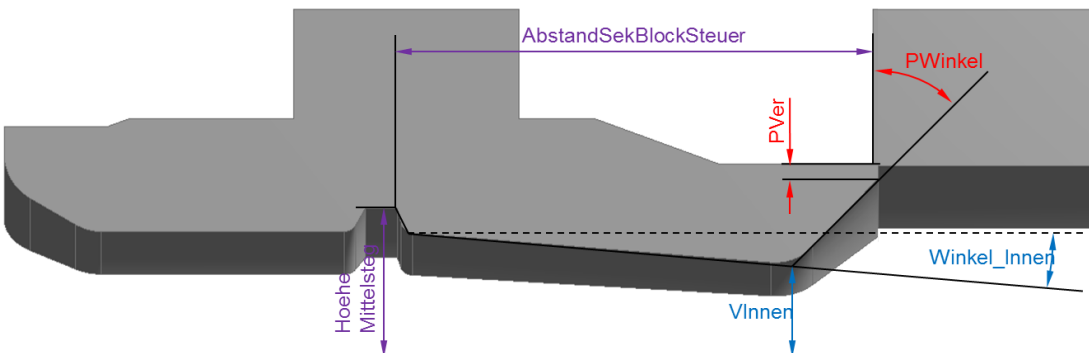
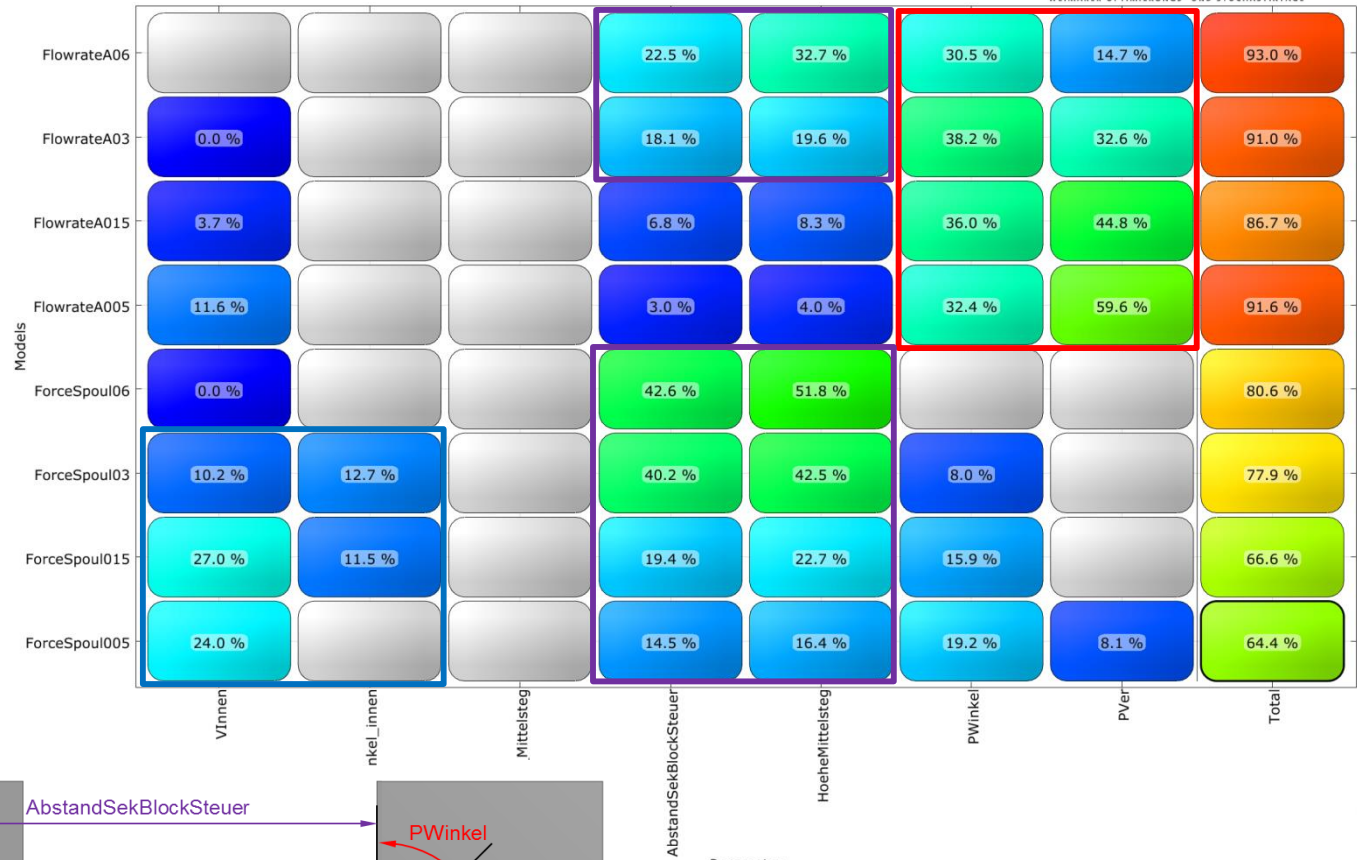
CoP

Important for jet force and flow rate:

- AbstandSBS
- HoeheMittelsteg

Important for jet force

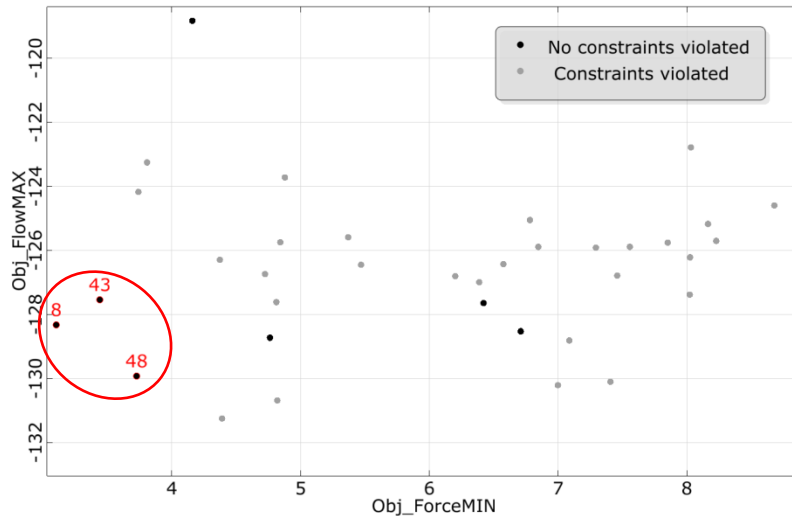
- VInnen
- Winkel_Innen



Important for flow rate

- PVer
- PWinkel

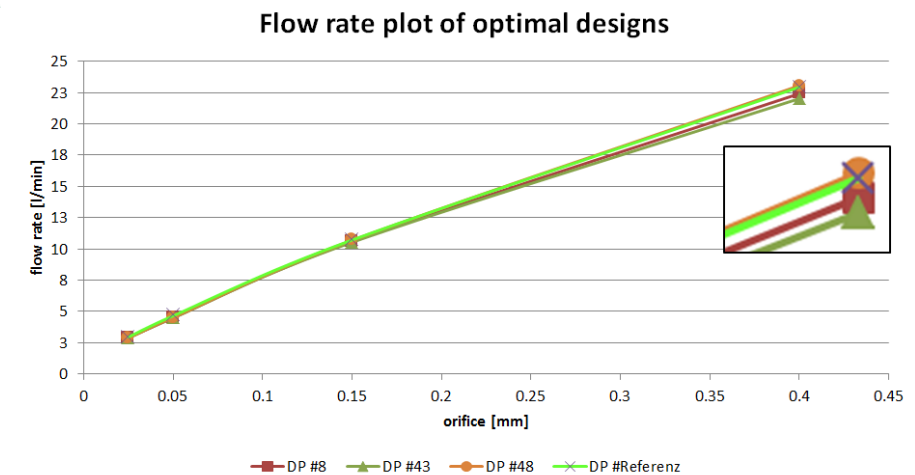
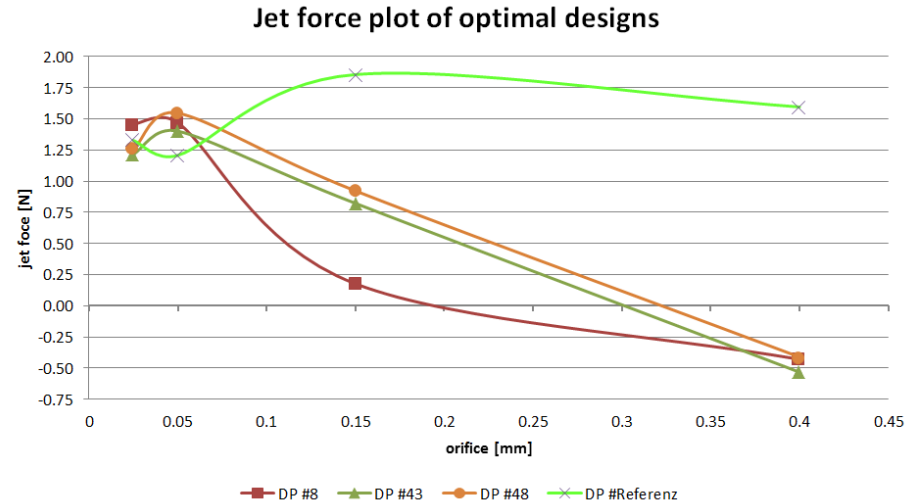
Results – Optimization Calculation



Three designs with major reduction of jet force: #8, #43 and #48

- #08 lowest jet force
- #48 highest flow rate
- #43 lowest jet force at small orifice; good characteristics of the curve

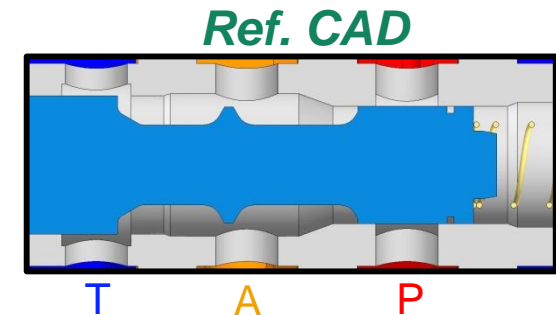
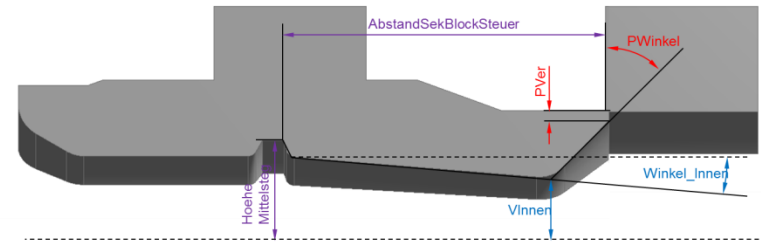
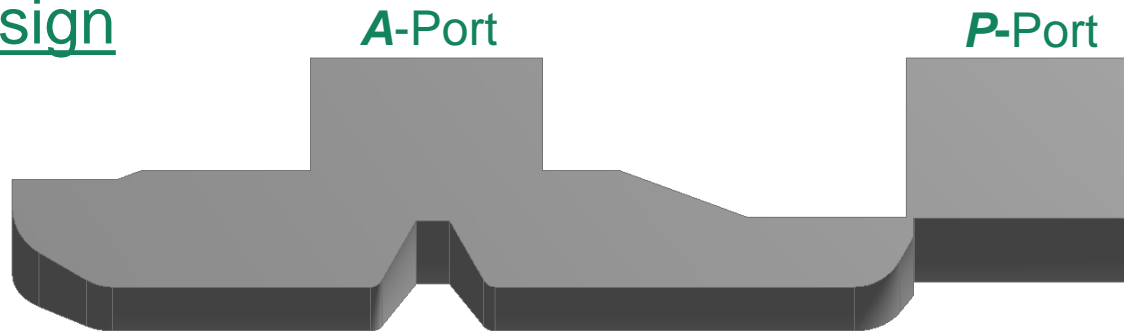
Optimal design for the given requirements is #43



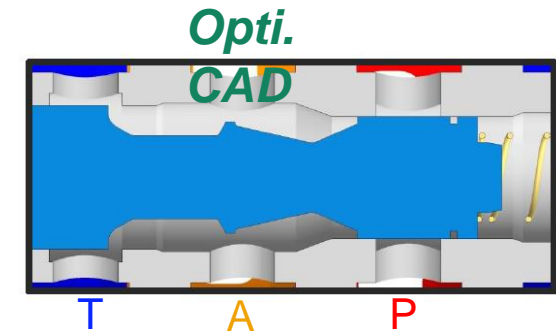
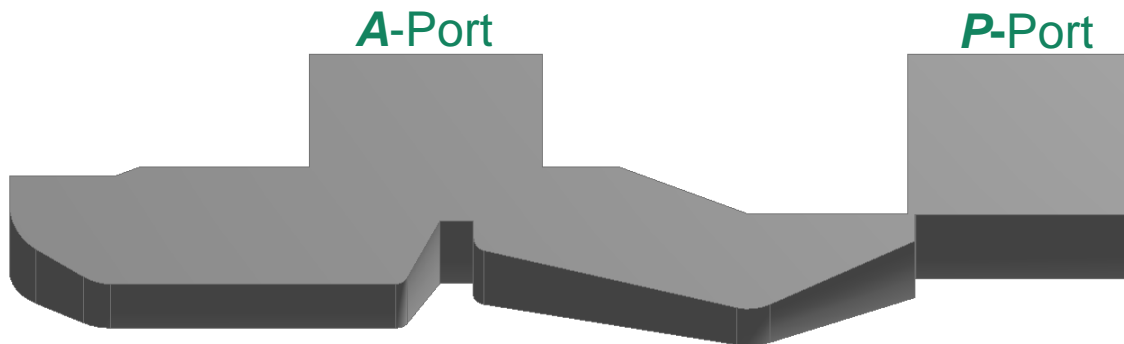
Results - Optimization

Geometry

Reference Design

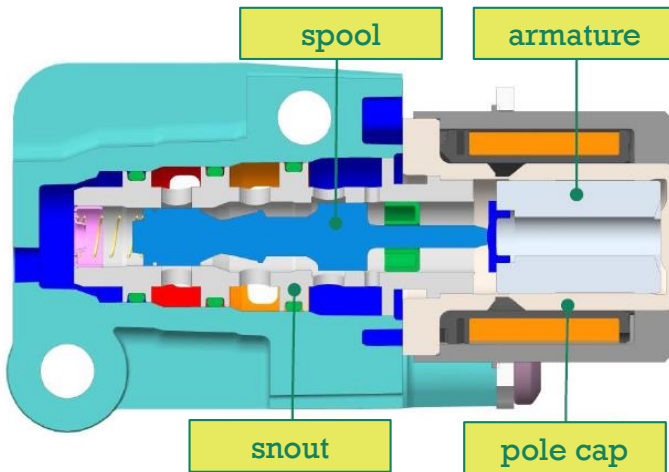


Optimized Design

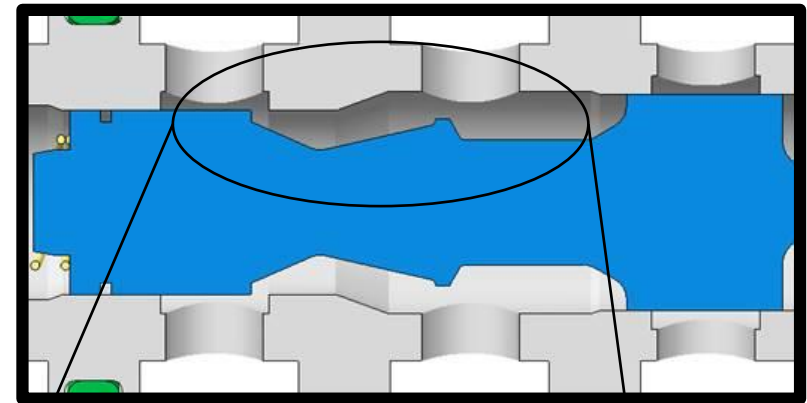


Results - Optimization

Changes of the spool



jet force compensation at



P A T

Jet Force reduction up to **50%** &
flow rate decrease of only **7%**

Density: 816.5kg/m³;
Viscosity: 0.0189Pa s;
Pressure: 20bar;
gap: 0.2mm
spool P-Port control edge

ledge at P-Port

snout P-Port control edge

A-Port

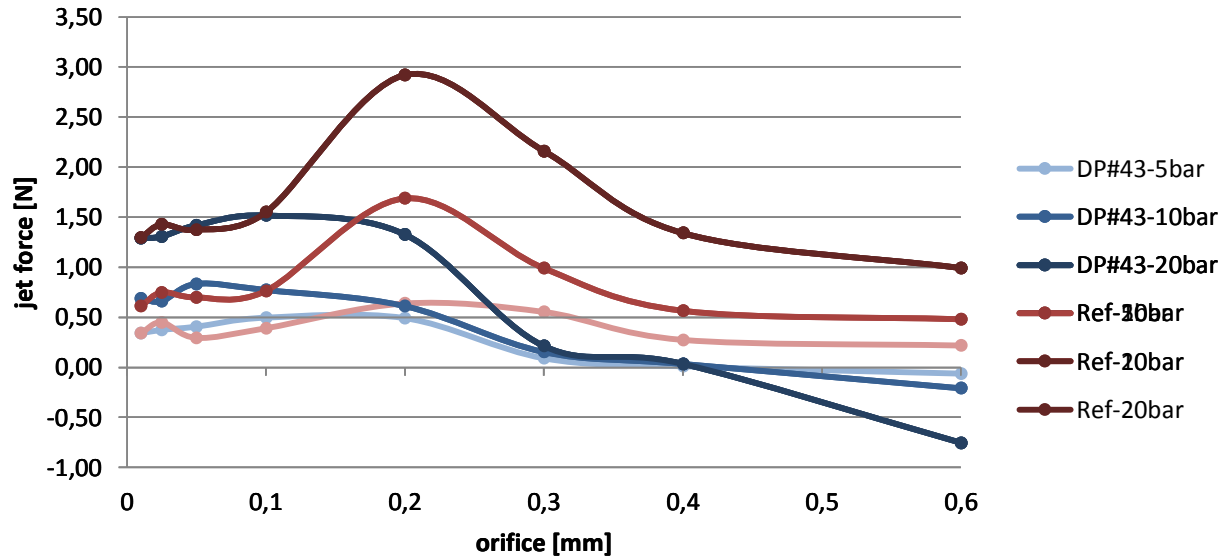
chamfer at P-Port control edge

chamfer at "Mittelsteg"

"Mittelsteg"

Results - Optimization

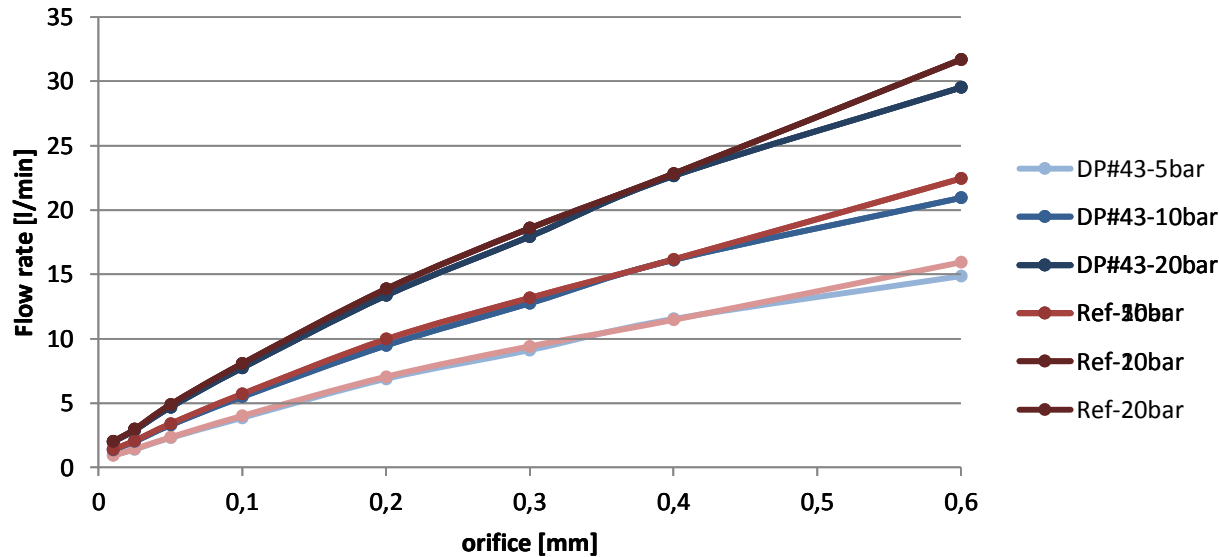
Jet force



- Detailed verification computation to show the behavior of the **jet force vs. spool position** (P-Port gap) for a pressure jump of 5bar, 10bar and 20bar
- **Comparison** of the **optimized design** and the **reference design** of the spool
- **Reduction of the jet force of 50%** around the peak value
- **Shift of the maximum position to a smaller gap**

Results - Optimization

Flow rate

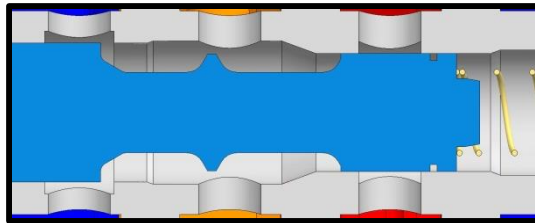


- Detailed verification computation to show the behavior of the **flow rate vs. spool position** (P-Port gap) for a pressure jump of 5bar, 10bar and 20bar
- **Comparison** of the valve between the **optimized** and the **reference spool**
- **No increase** of the **flow rate** with a reduced jet force **possible**
- **Low decrease** of the flow rate (maximum at **7%**) with a significant reduction of the jet force (50%)

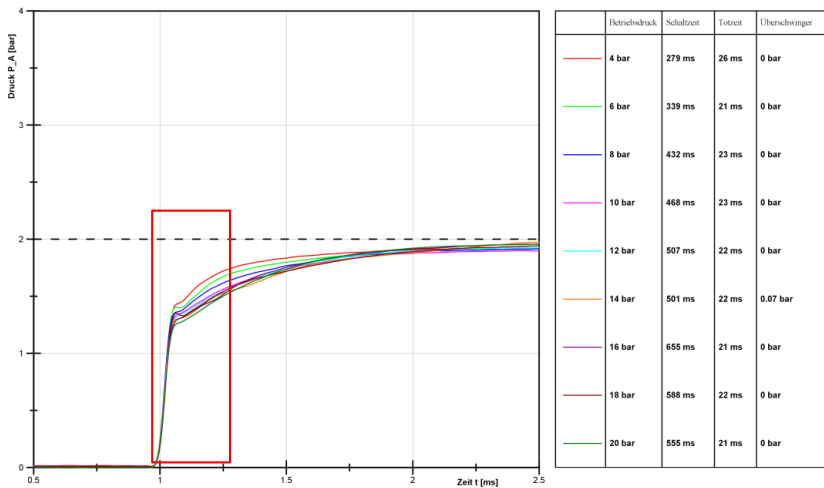
Results - Optimization

Response time

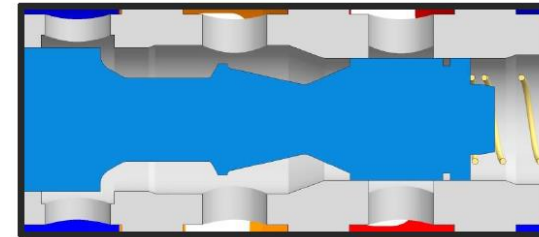
p-t-curve without jet force compensated spool (reference)



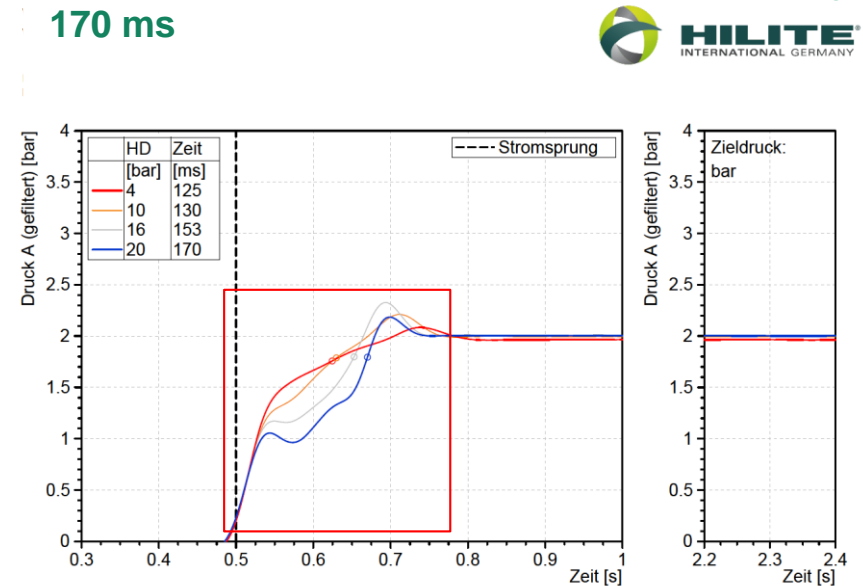
Max. response time at 20bar (90% pressure target):
550 ms



p-t-curve with jet force compensated spool (optimized)



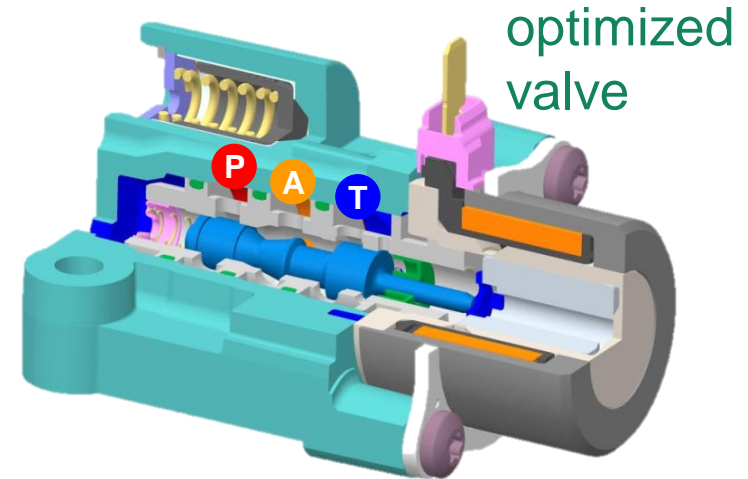
Max. response time at 20bar (90% pressure target):
170 ms



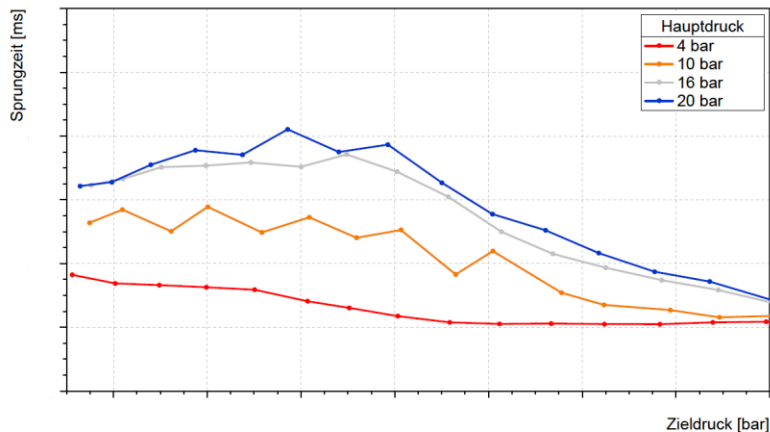
Results - Optimization

Step response

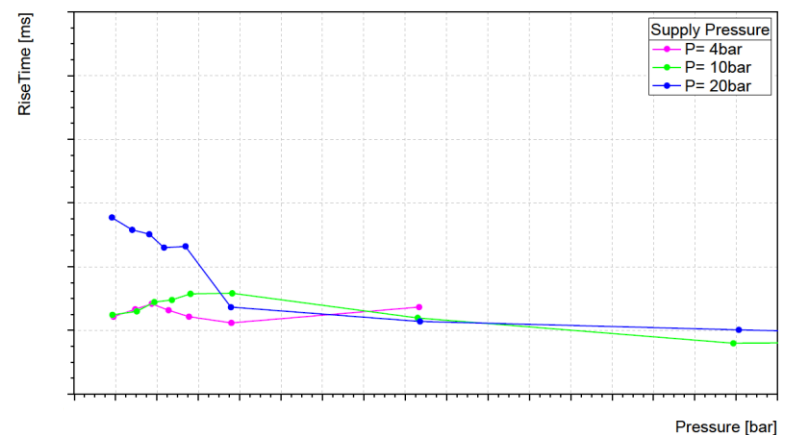
- max. step response at 20bar supply pressure of 280ms (reference 420ms)
- pressure dependence within a band of <200ms (reference 250-300ms)



Reference valve



Optimized valve



Summary

- Using the given parameters, a reduction of the jet force at small gaps is very difficult
- Due to the optimized spool the maximum jet force decreased by 50%
- The critical peak of the jet force (reference design) was removed
- Reduction of the flow rate decreased by 7% at maximum gap
- Tests with the real optimized design show the improvement of the valve`s response time
- No cogging during gear change with the optimized valve

➔ Optimization target of a faster valve fulfilled



Danke für Ihre
Aufmerksamkeit.

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