

Optimization of a Radial Pump using CFturbo, TCFD & optiSLang

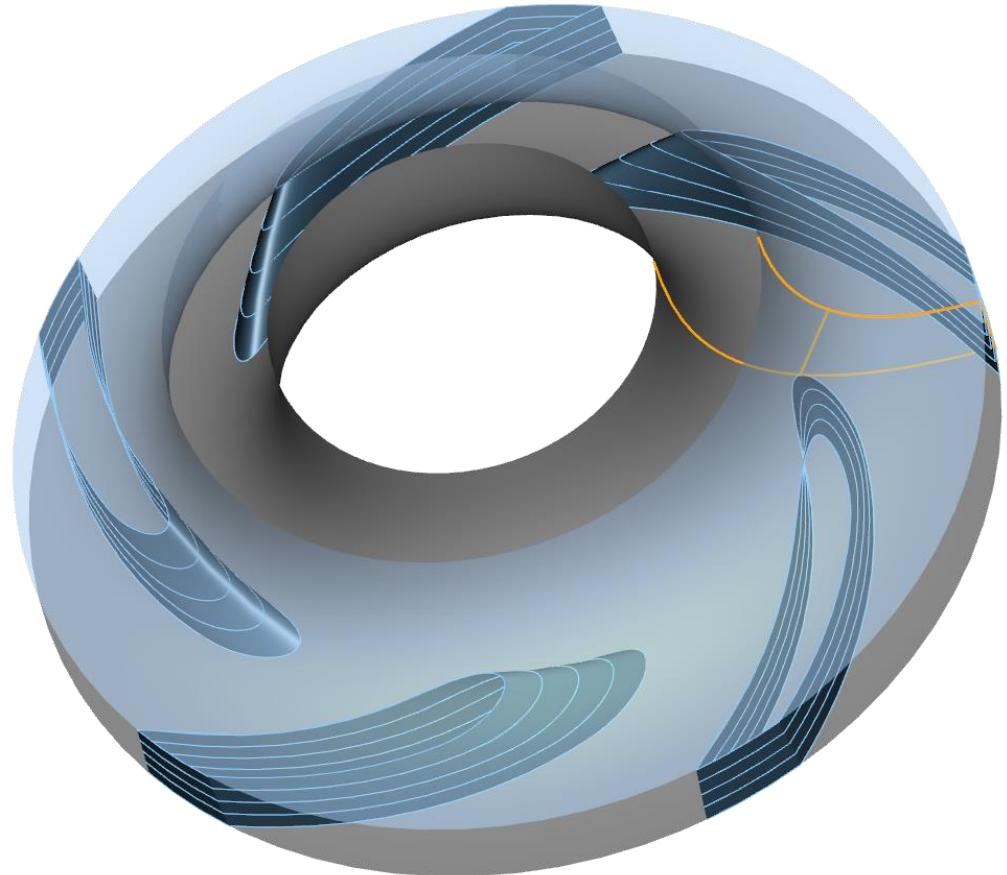
M.Sc. Daniel Pöscha, Chris Kemmerzell

CFturbo GmbH



Content

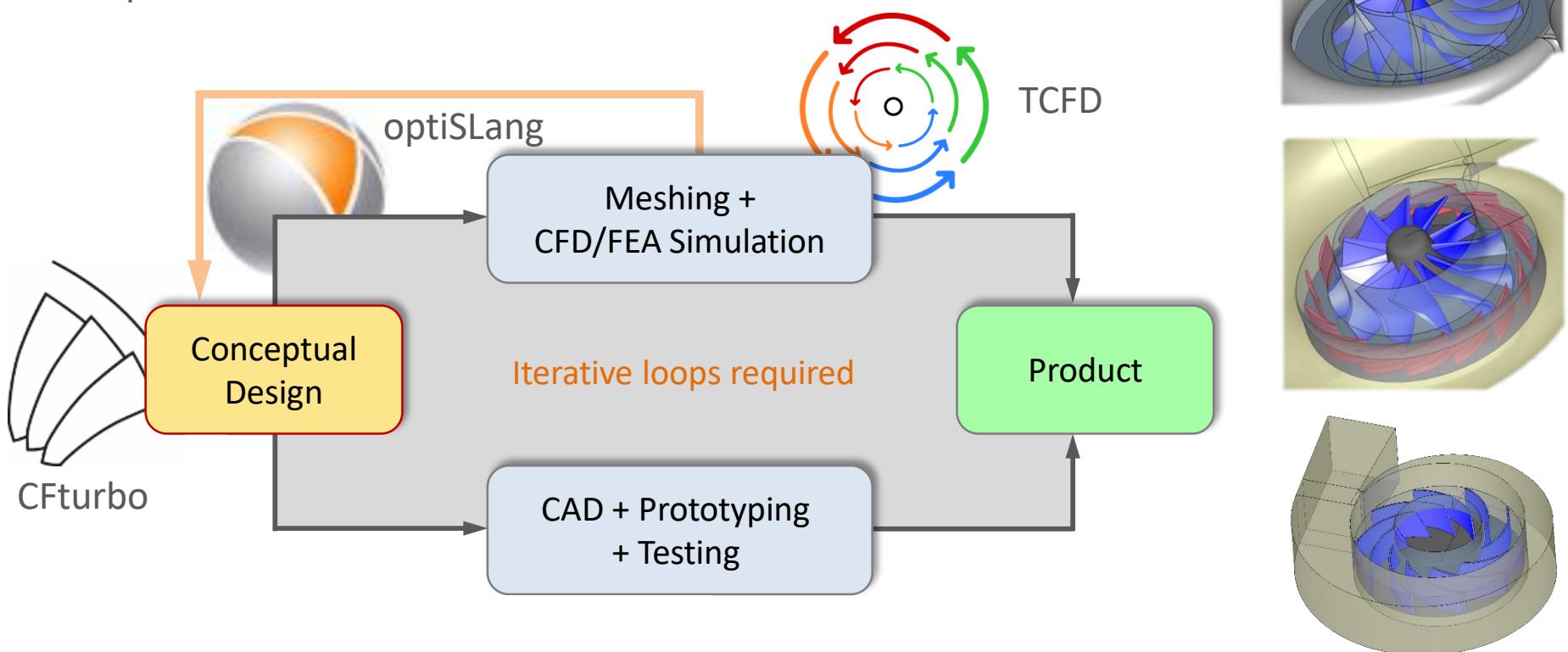
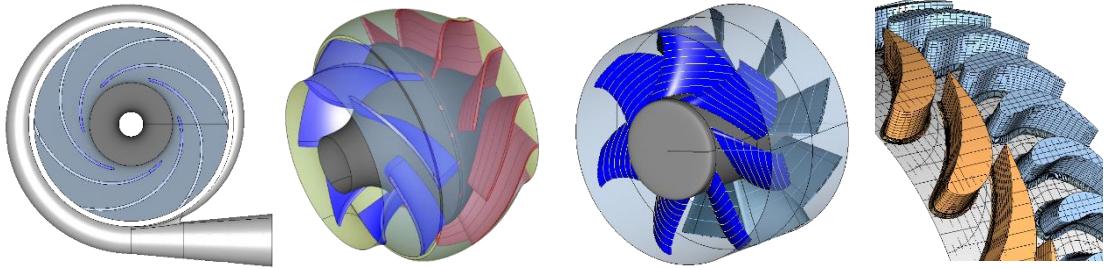
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1. Motivation

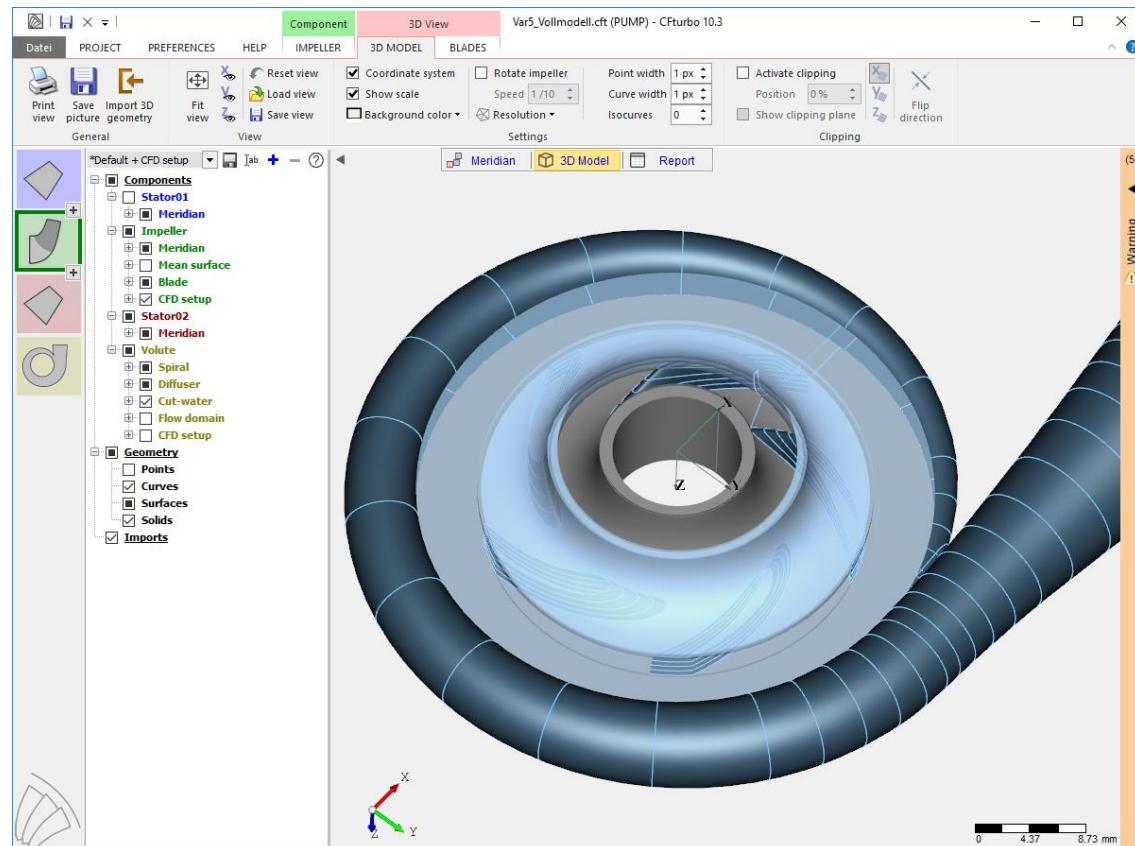
Turbomachinery design is

- Complex
- Time-consuming
- Expensive



2. Software tools a) CFturbo

- Modern turbomachinery design software for
 - ✓ Pumps
 - ✓ Ventilators
 - ✓ Compressors
 - ✓ Turbines
- Covers all major types:
 - ✓ Radial
 - ✓ Mixed-flow
 - ✓ Axial
- Available components:
 - ✓ Impeller
 - ✓ Volute
 - ✓ Vaned/vaneless stators
- Analytical design theory & empirical correlations

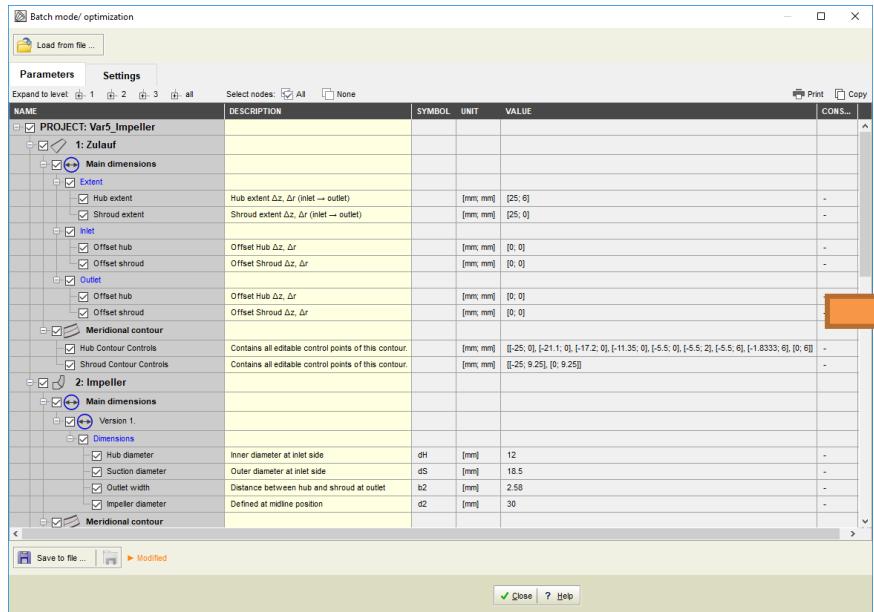
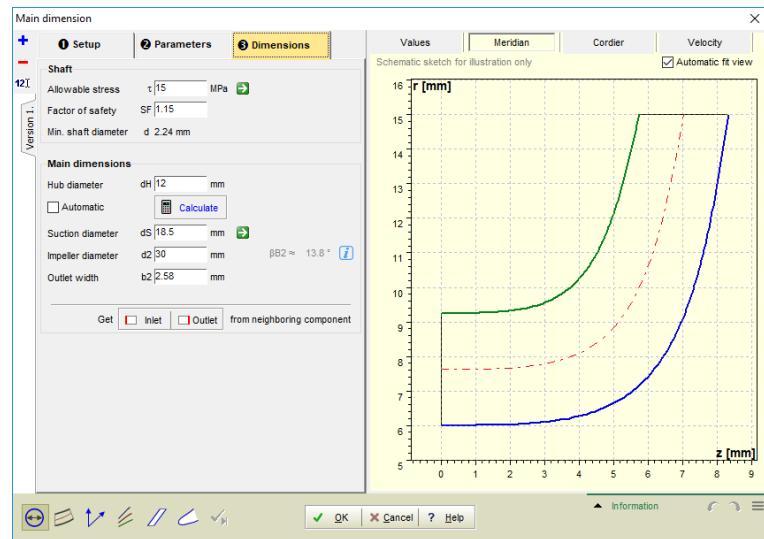


→ Fully parametric geometry model of machines

2. Software tools

- Several design steps from design point to 3D geometry
- All parameters can be accessed via batch mode per XML file
- optiSLang integration available

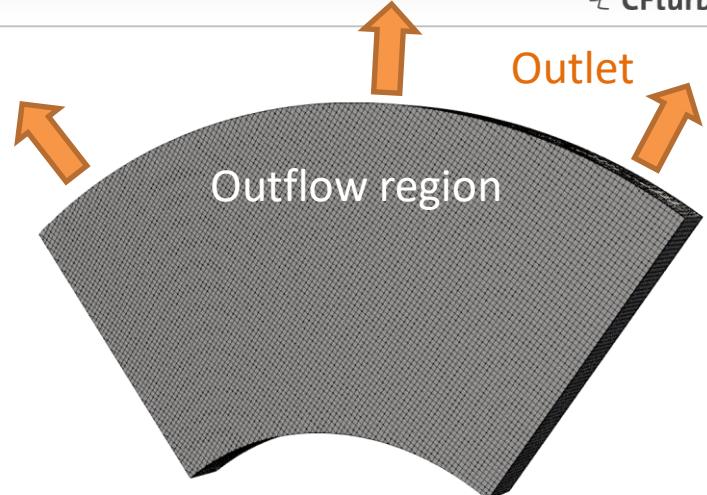
a) CFturbo



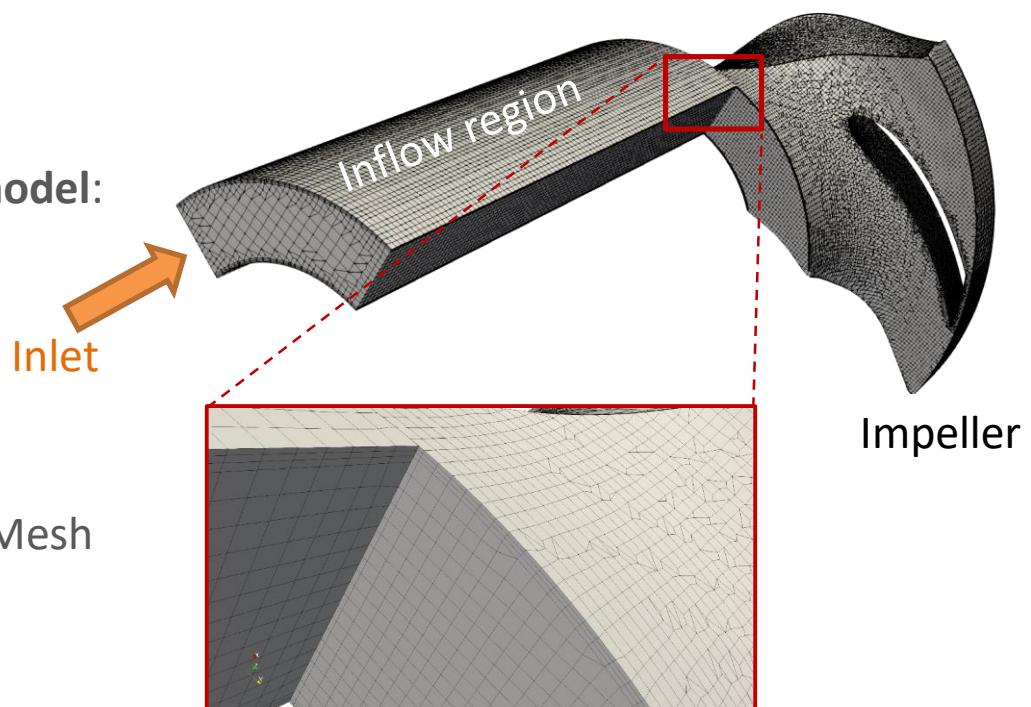
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```

2. Software tools b) TCFD

- **TCFD® = Turbomachinery CFD**
- CFD workflow for turbomachinery simulations
- Commercial software based on OpenFOAM®
- Graphical user interface (GUI) in ParaView
- Developed by CFD Support s.r.o.



Geometry model:



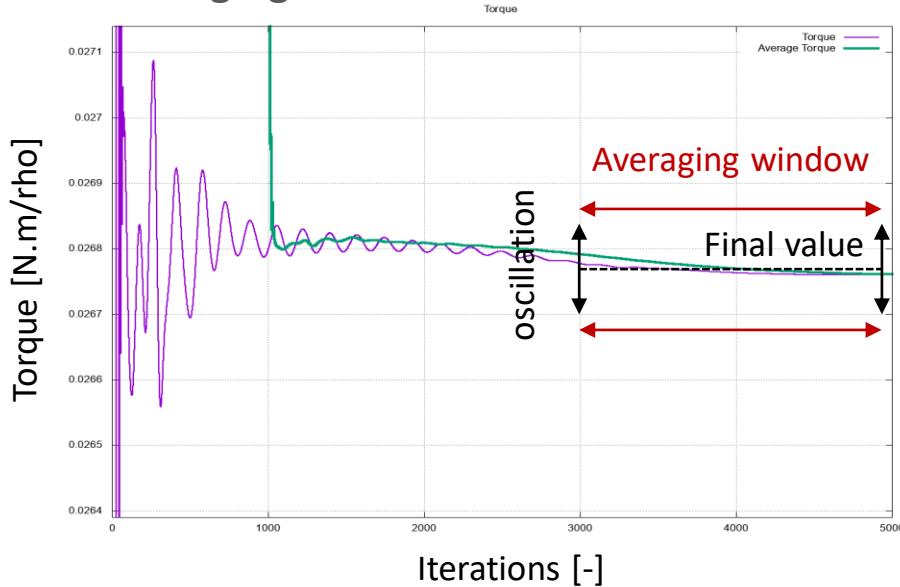
Mesh density:

- 270 000 cells
- Created with snappyHexMesh
- Solver speed \Leftarrow accuracy

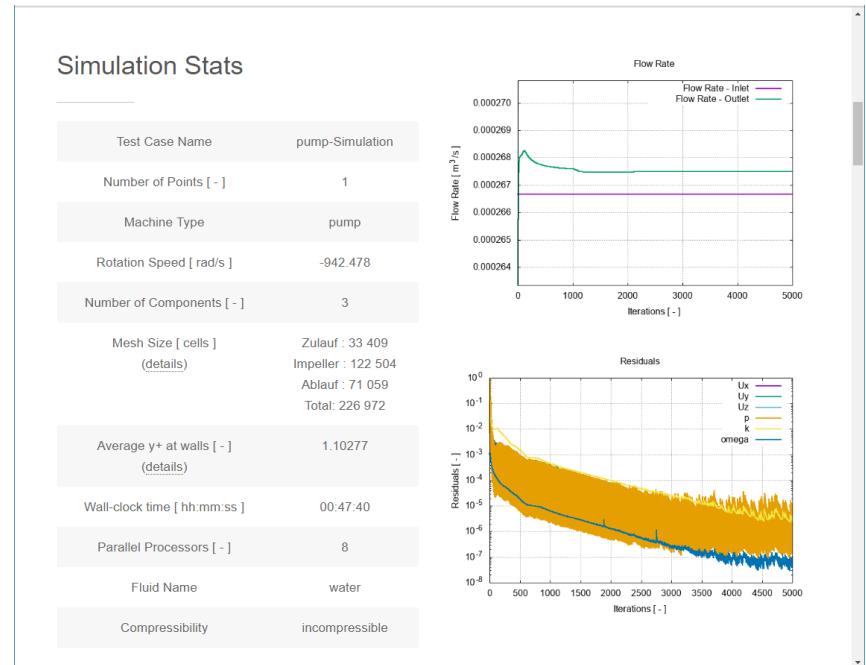
2. Software tools

b) TCFD

Averaging window:



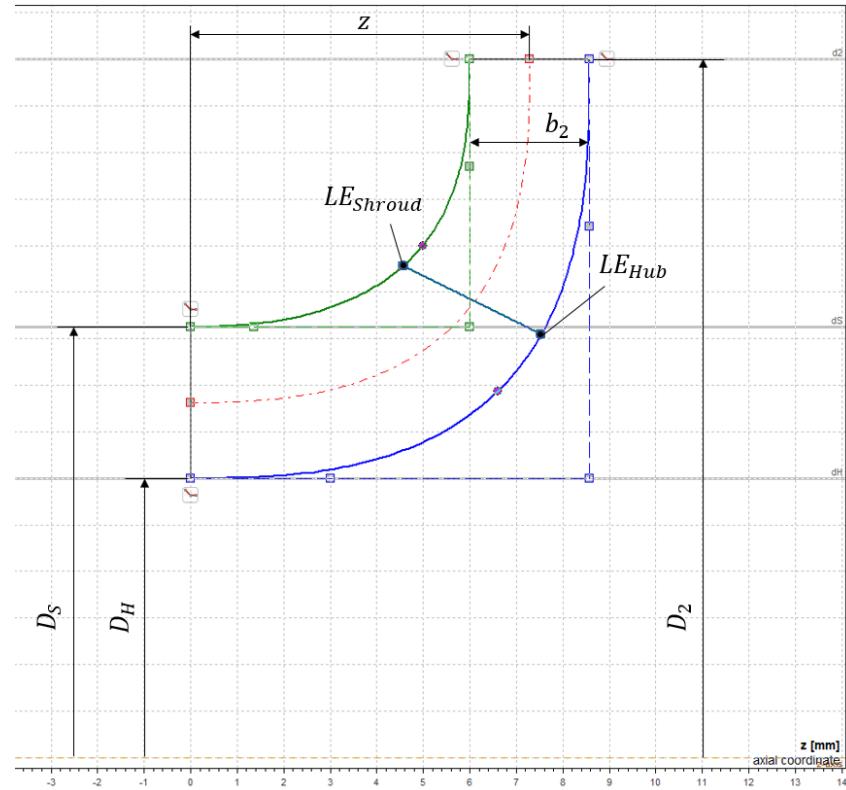
HTML Report:



3. Parameters

- Overall 12 parameters are used

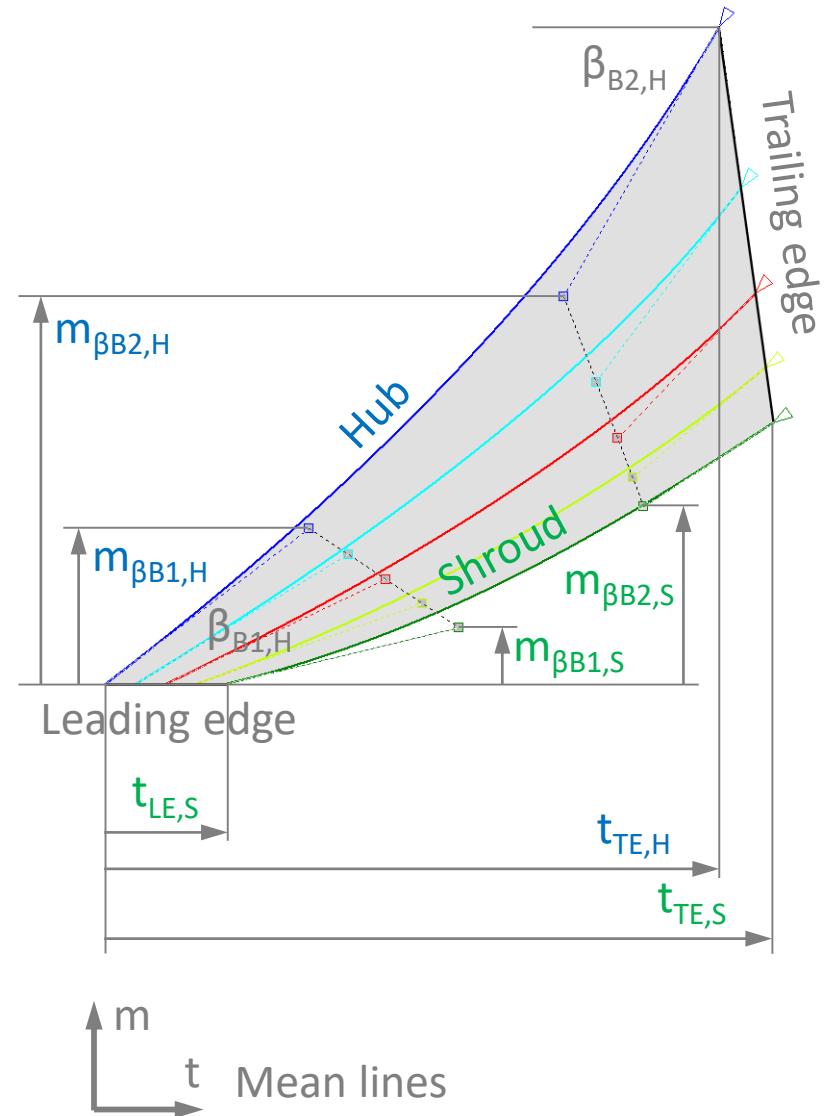
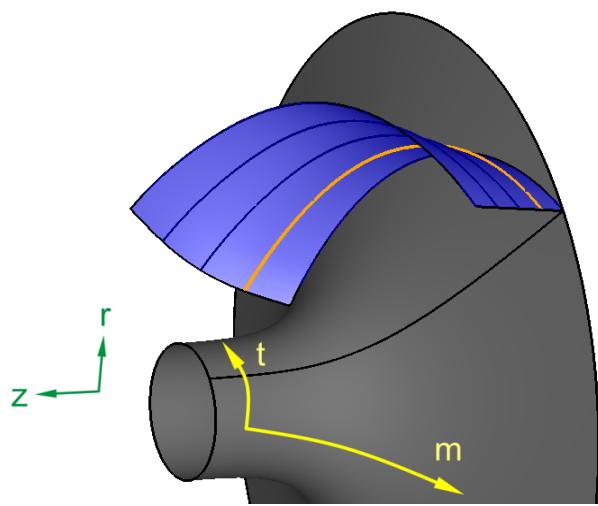
	#	Parameter	Reference	Minimum	Maximum
Main dimensions	1	D_H [mm]	12.0	9.6	14.4
	2	D_S [mm]	18.5	14.8	22.2
	3	b_2 [mm]	2.58	2.064	3.096
	4	D_2 [mm]	30	24	32
Meridional contour	5	z [mm]	7.29	5.832	8.748
	6	LE_{Hub} [%]	58.63	53.1	64.9
	7	LE_{Shroud} [%]	50.16	45	55



- Minimum / Maximum values: $\pm 10 \dots 20 \%$
- Maximum D_2 limited by available space

3. Parameters

	#	Parameter	Reference	Minimum	Maximum
Blade properties	8	n_{BI} [-]	5	3	7
	9	$\beta_{B1,H}$ [°]	19.65	15.72	23.58
	10	$\beta_{B2,H}$ [°]	28.0	22.40	33.60
Mean lines	11	$m_{\beta B1,H}$ [%]	36.15	20	40
	12	$m_{\beta B2,H}$ [%]	66.62	60	70



4. Optimization workflow

Design point

- Flow rate $Q = 0.96 \text{ m}^3/\text{h}$
- Total pressure difference $\Delta p_t = 0.87 \text{ bar}$ ($H = 8.885 \text{ m}$)
- Rotational speed = 9000 /min
- Water, no pre-swirl

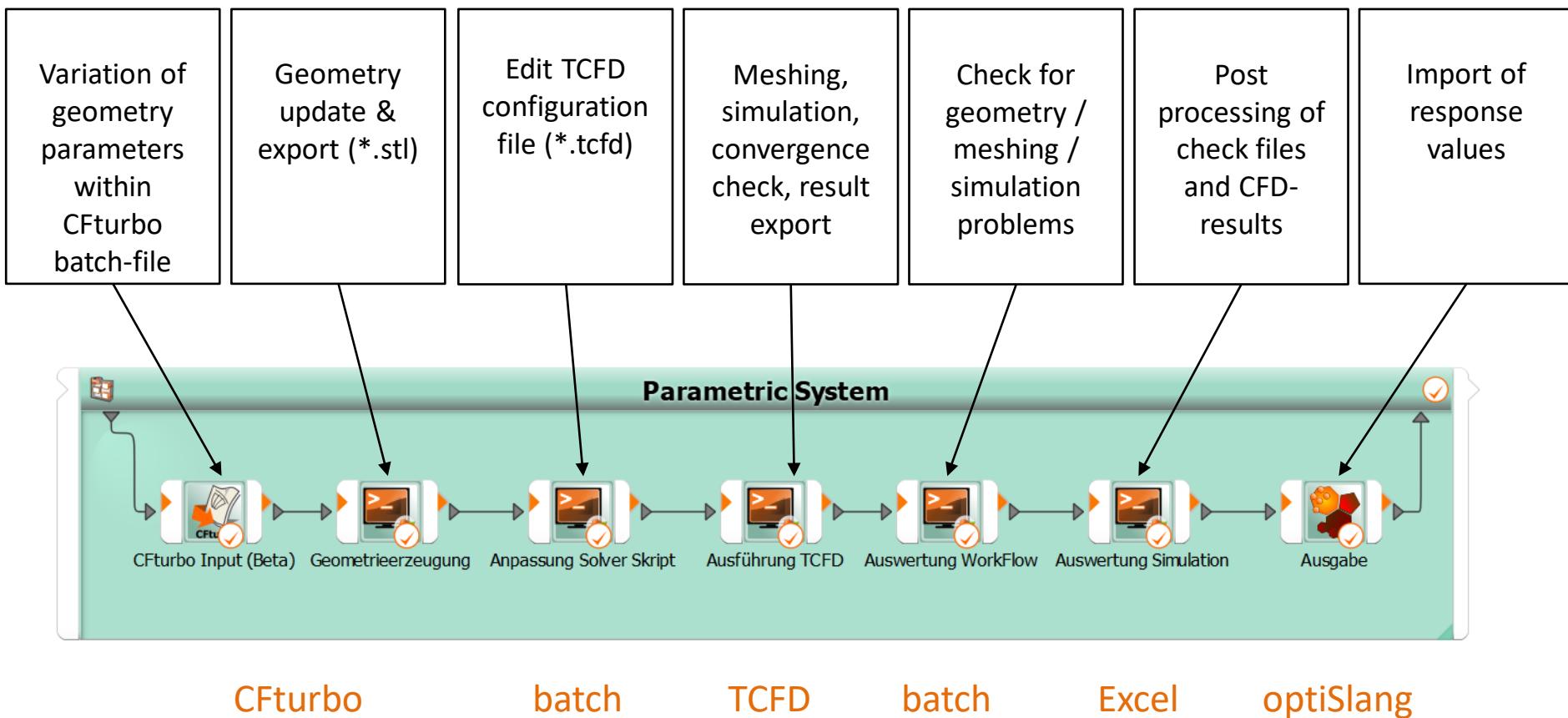
Objective

- Max. hydraulic efficiency η_{hydr}

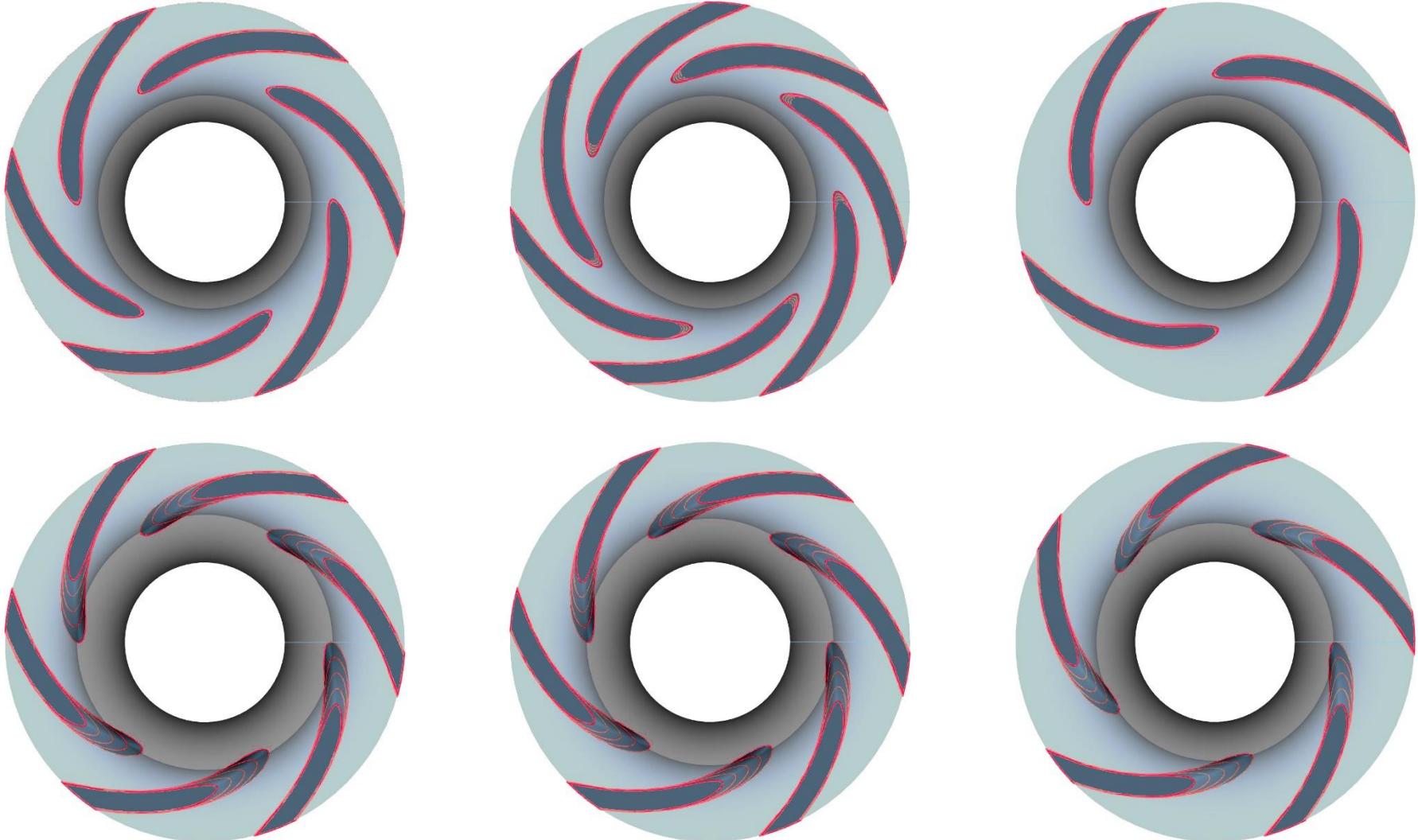
Constraints

- $P_{\text{hydr}} < 30 \text{ W}$
- Variation of total pressure difference $\Delta p_t \pm 10\%$

4. Optimization workflow



5. Results – Some geometry examples



5. Results – Sensitivity analysis / Number of samples

Advanced Latin Hypercube Sampling

		Samples
		100
Failed	Geometry	10
	Meshing	7
	Simulation	20
Valid	samples	63
Constraints	Valid	19
	Not valid	44
CoP	Δp_t	95.7 %
	P_{hydr}	96.6 %
	η_{hydr}	94.2 %

High variation of results

- $61,27\% \leq \eta_{hydr} \leq 91,15\%$
- 36 samples with low η_{hydr}
- Reduction of samples to calculate CoP not possible



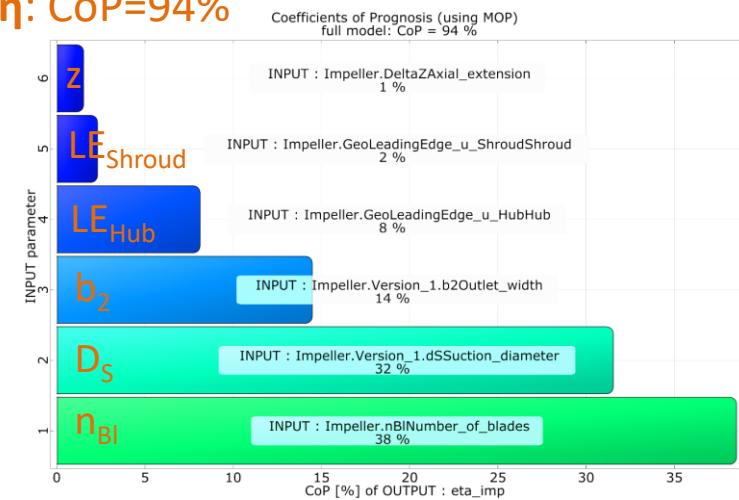
5. Results – Sensitivity analysis / Coefficient of Prognosis CoP

η hydraulic efficiency (objective)

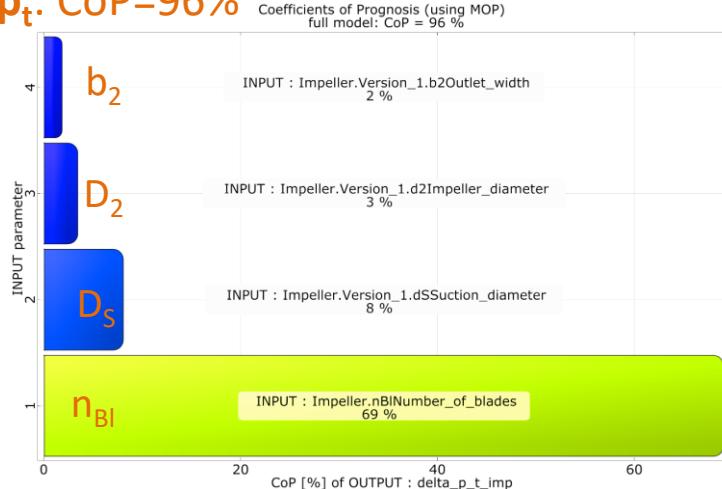
$$\eta = \frac{Q \cdot \Delta p_t}{P}$$

Q Flow rate
 Δp_t Total pressure difference
P Power consumption

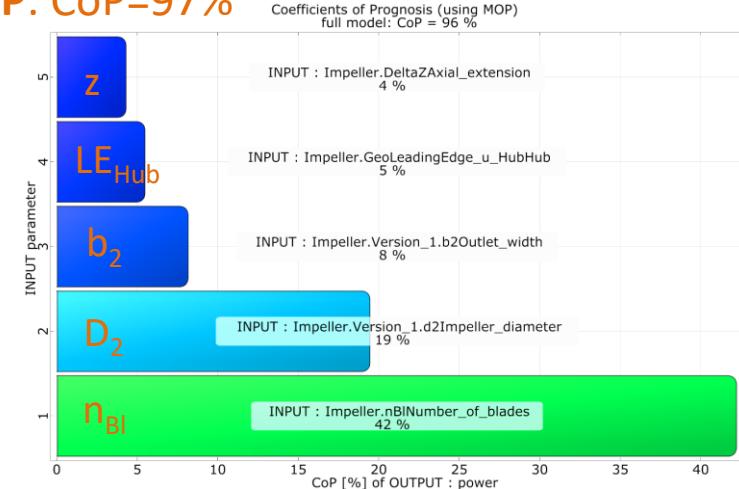
$\eta: \text{CoP}=94\%$



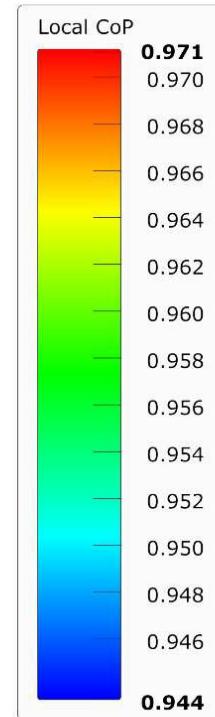
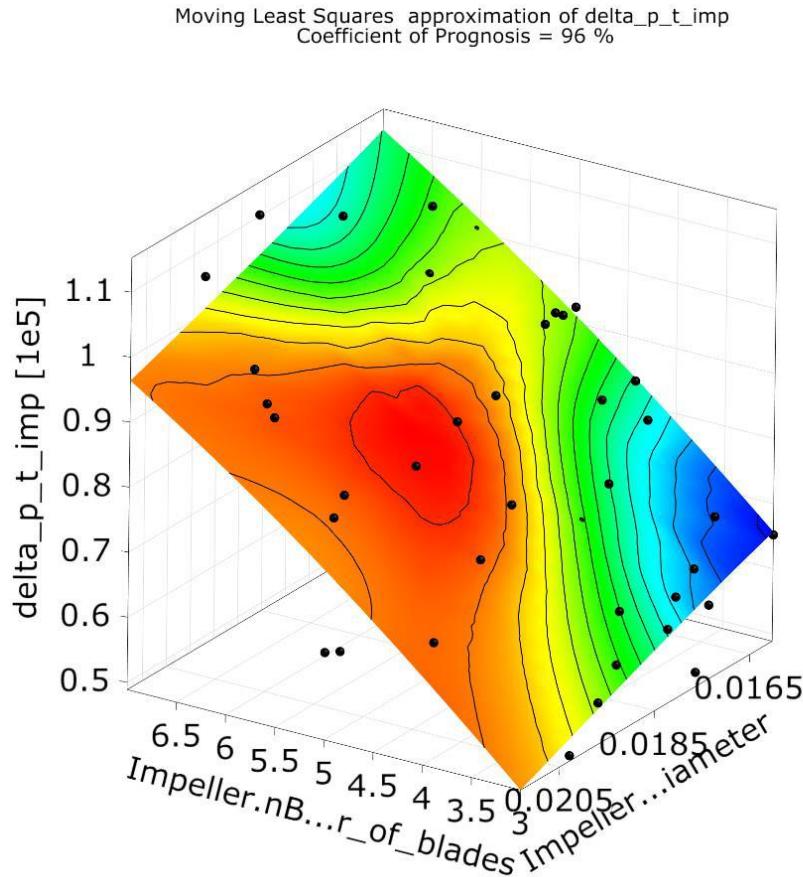
$\Delta p_t: \text{CoP}=96\%$



$P: \text{CoP}=97\%$

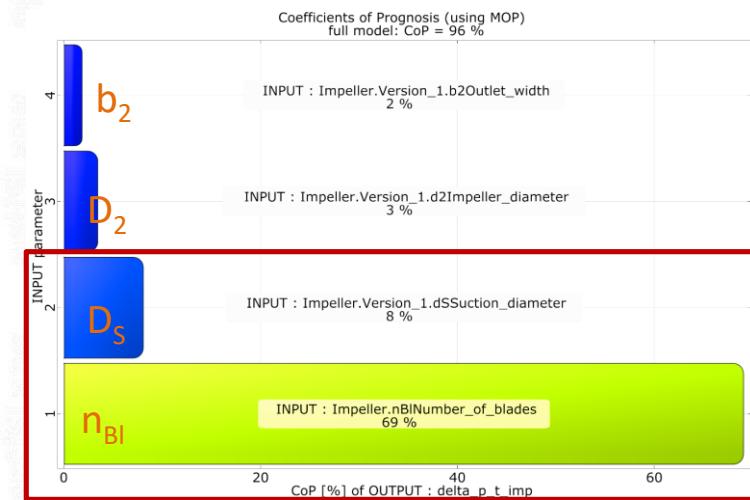


5. Results – Sensitivity analysis / Metamodel of Optimal Prognosis MOP

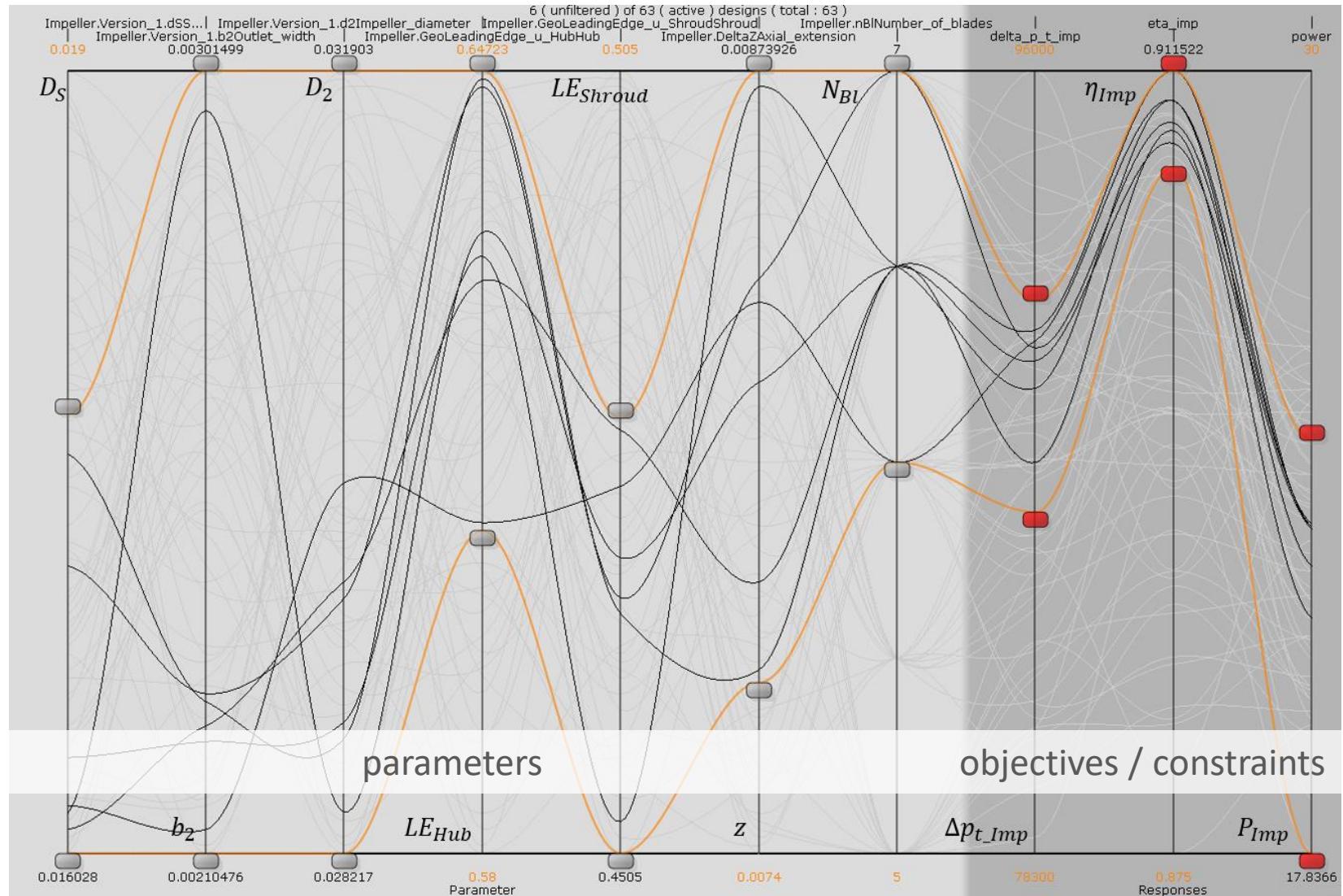


Response surface for
 $\Delta p_t = f(n_{BL}, D_s)$

Δp_t : CoP=96%



5. Results – Sensitivity analysis / Parallel coordinate plot



5. Results – Optimization

Algorithm	Samples	Simulation time	Simulation time/ sample	η_{hydr}
EA Evolutionary Algorithm	400	$\approx 235 \text{ h}^*$	$\approx 35 \text{ min}$	92.2 % + 10.2 %
ARSM Adaptive Response Surface Method	260	$\approx 255 \text{ h}^{**}$	$\approx 60 \text{ min}$	92.2 % + 10.2 %

* Desktop PC

- 2 x Intel Xeon 3.30 GHz, 8 cores
- 128 GB RAM

compared to
initial design

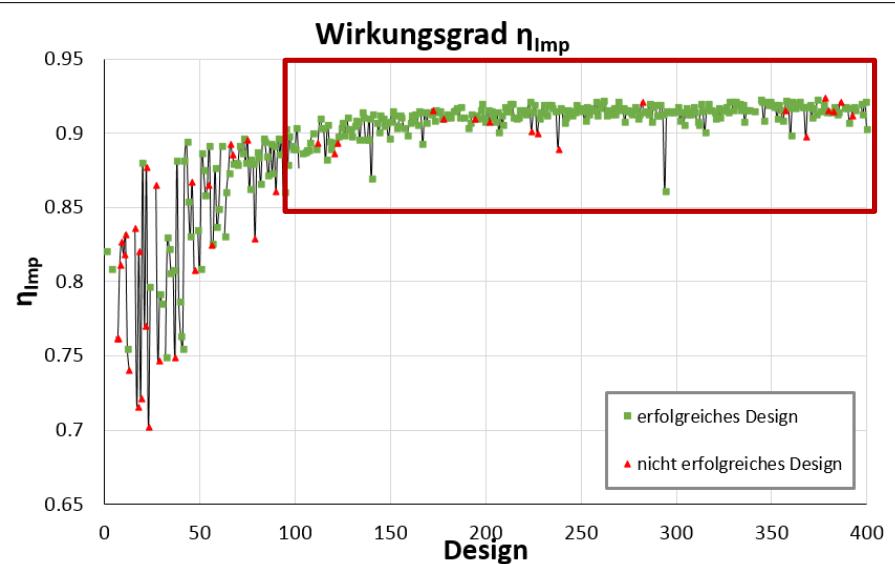
** Desktop PC

- 2 x Intel Xeon 3.07 GHz, 6 cores
- 32 GB RAM

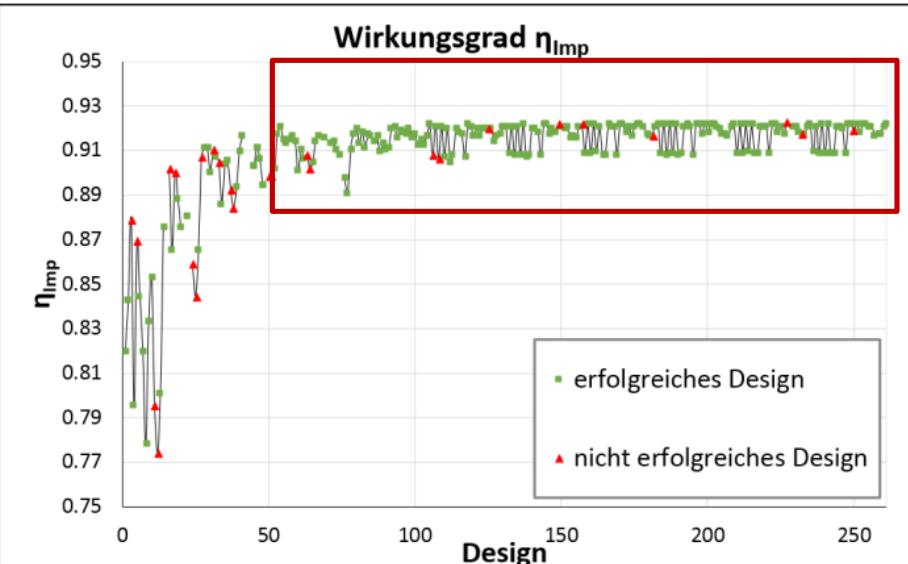
5. Results – Optimization

	D_H [mm]	D_S [mm]	D_2 [mm]	n_{BL} [mm]	z [mm]	$\beta_{B1,H}$ [°]	$\beta_{B2,H}$ [°]	Δp_t [bar]	P [W]	η_{hydr} [%]
EA	12.12	15.98	29.71	6	7.28	18.51	23.73	0.897	25.94	92.2
ARSM	12.00	15.95	30.00	5	7.09	19.65	28.00	0.904	26.15	92.2

EA:



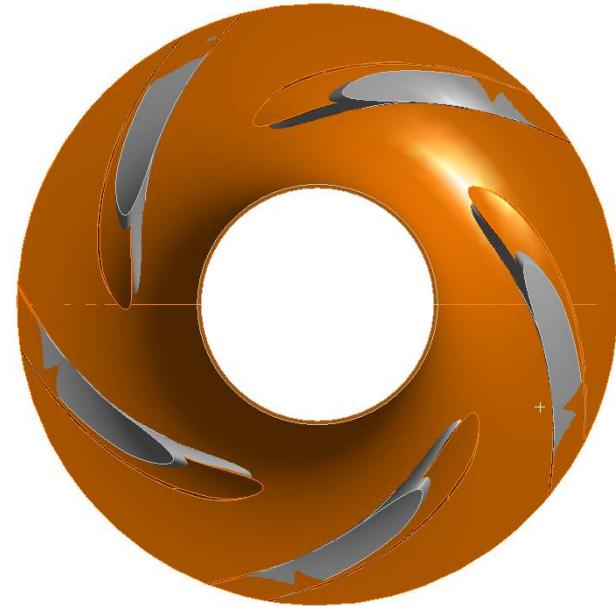
ARSM:



5. Results – Optimization

Reference

ARSM



Reference

EA



ARSM

EA



6. Conclusion, outlook

- Optimization process CFturbo + TCFD + optiSLang successfully tested
- CFturbo initial design can be used as very reasonable starting point („pre-optimized“) to save optimization time
- Geometry simplifications necessary to save optimization time
→ segment model
- High CoP of efficiency due to high variation of results
- Different geometric designs as result of EA and ARSM
→ more than 1 optimum with similar efficiency
- Next steps:
 - Use metamodel for indirect optimization
 - Optimization of other components (volute, diffusor) of pump with optimized impeller

Thanks to Dynardo (optiSLang) and CFD Support (TCFD) for licenses and support!