

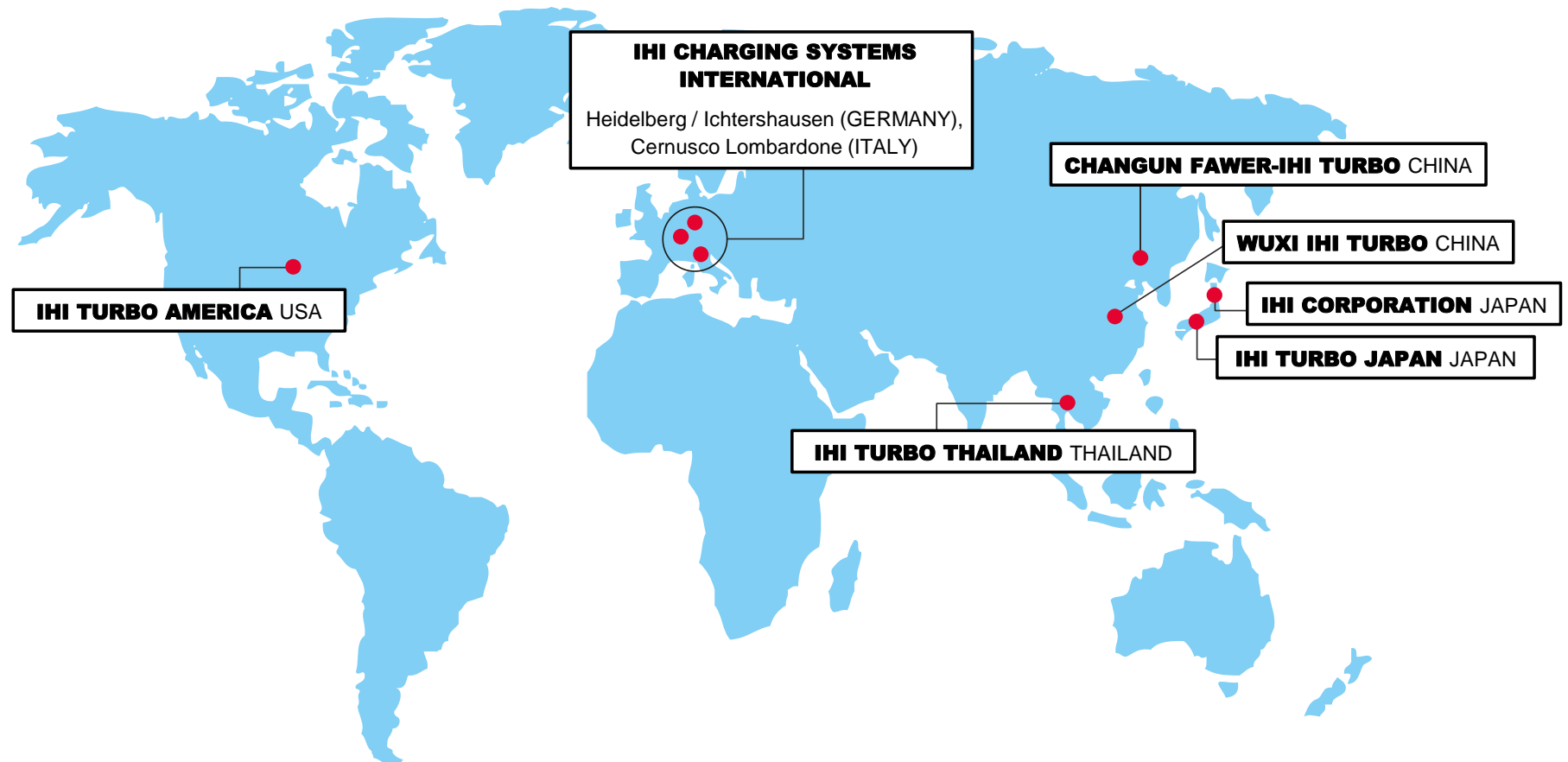
High Speed Balancing Mark Optimization

IHI

25.06.2020

I H I Corporation

IHI Charging Systems International GmbH
Martin Kreschel, Roberto DeSantis



ICSI is an established provider of charging systems in the European turbocharger segment. Millions of our systems are successfully used by almost all European automotive manufacturers.

With fast processing time and sophisticated production technologies, even during the prototype phase, we contribute to shortening development time and increasing project effectiveness.

Future orientation and the associated responsibility are an integral part of our corporate philosophy and are implemented every day by our employees across the entire process chain.

LOCATION

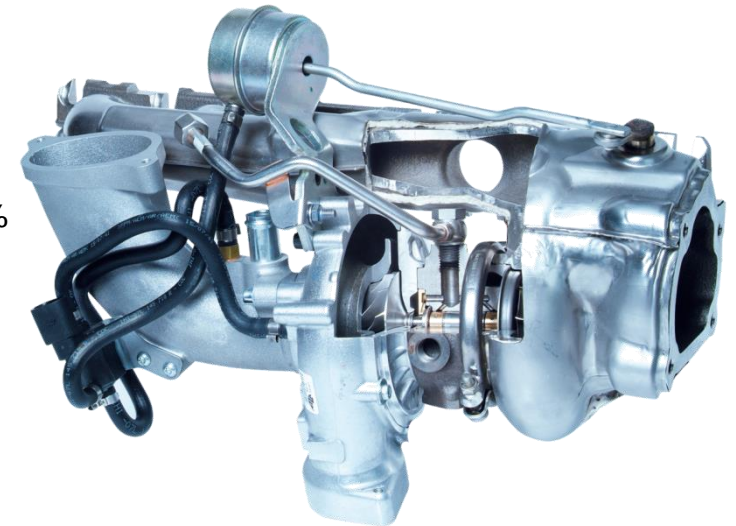
Heidelberg, Germany

ESTABLISHED

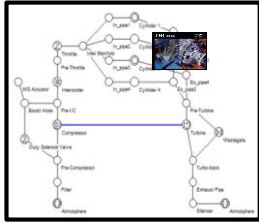
IHI Charging Systems International GmbH was founded in 2001 as a joint venture company between Daimler AG and IHI Corporation (Japan). Since March 2013, ICSI is a 100% subsidiary of IHI Corporation.

FUNCTIONS

Engineering center
Engine & T/C test center
Prototype shop
Project management of European customers

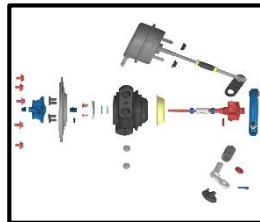


LOCATION	Ichtershausen (Thuringia), Germany - Headquarter	Cernusco Lombardone (Lecco), Italy
COMPANY	IHI Charging Systems Germany International GmbH 100% shares owned by ICSI GmbH	IHI Charging Systems International S.p.A. 100% shares owned by ICSI GmbH
ESTABLISHED	2008, opening ceremony in April 2009 Second production hall completed in October 2012	1995 as WARNER-ISHI Turbo 1998 IHI Turbo Italy S.p.A. 2001 IHI Charging Systems International S.p.A.
CAPACITY	> 2 million units/year	> 2 million units/year
FUNCTIONS	<ul style="list-style-type: none">• Machining bearing housing• Machining compressor wheel• Welding turbine wheel & shaft• CS assembly and balancing• TC assembly• Testing• Prototype shop	<ul style="list-style-type: none">• Machining bearing housing• CS assembly and balancing• TC assembly• Prototype shop



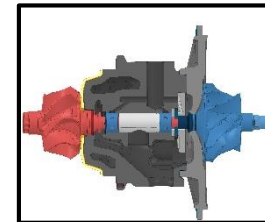
- 0D simulation (ITES in-house tool)
- TC frame size selection
- Turbine & compressor specification
- 1D simulation (GT-Power tool)

Matching TC/ICE



- Standard and customized components
- 3D design
- 2D drawings
- Materials

Design



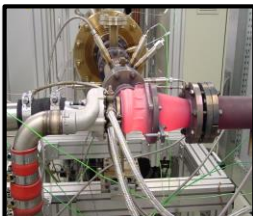
- FEA compressor & turbine wheel
- CW-shaft connection
- Rotordynamics
- Thrust force
- Oil & water core

Verification of center section



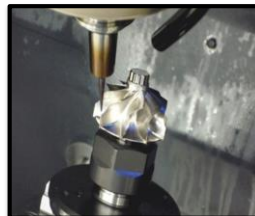
Validation/ Testing

- Standard test procedures
- Cold-gas test benches
- Hot-gas test benches
- Engine test bench



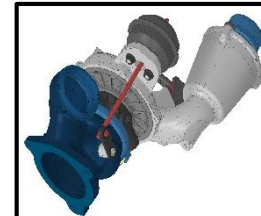
Prototyping

- Machining center
- Inspection and metrology lab
- Prototyping/ assembly

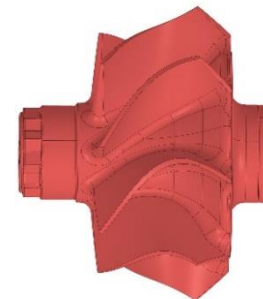
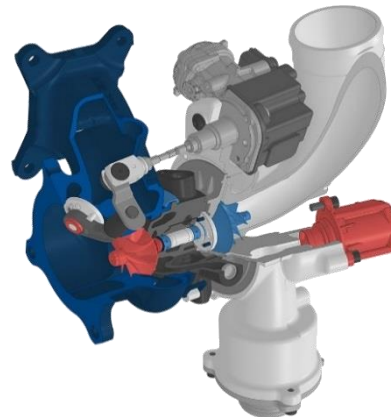
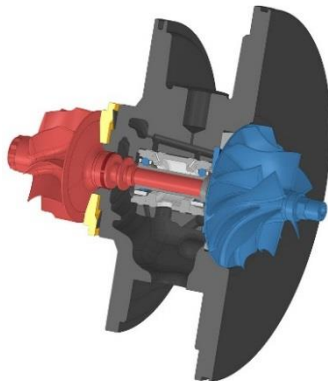
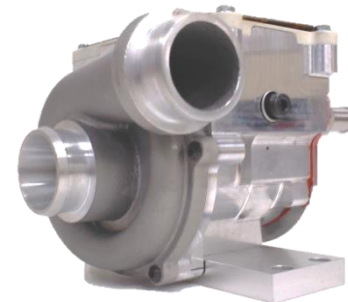
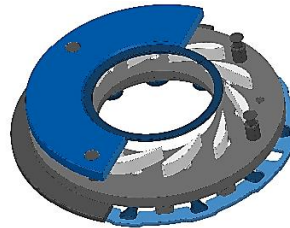


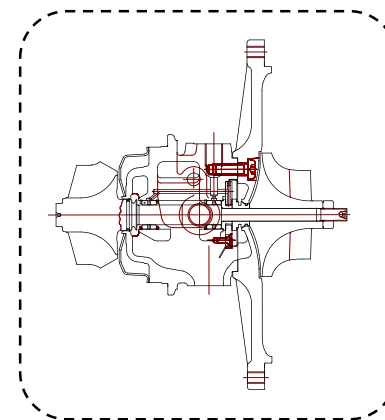
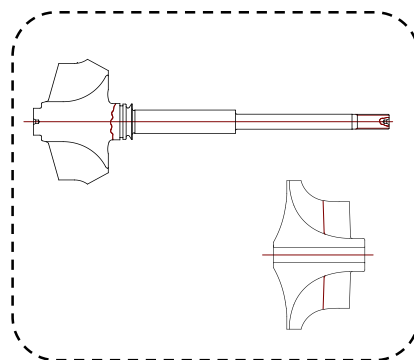
Verification of assembly

- CFD compressor & turbine stage
- FEA housings
- Thermo-mechanical fatigue of turbine housing
- Modal and Forced Response Analysis

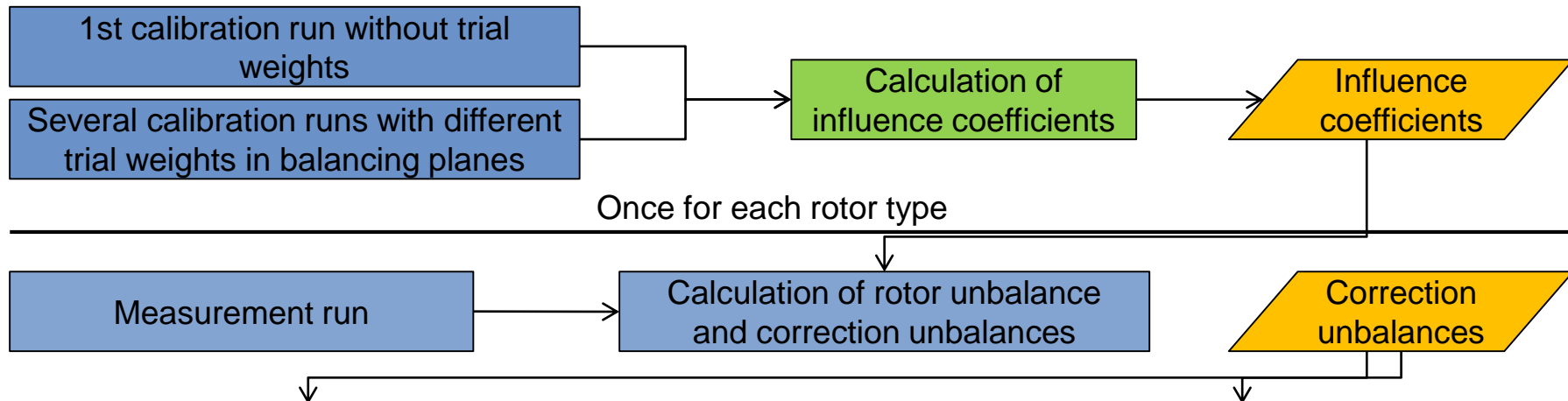


- Turbine housing with integrated manifold
- Turbine housing adapted to manifold integrated into cylinder head
- Ball bearing
- E-compressor
- E-assisted turbo
- Mixed-flow turbine
- Twin scroll turbine
- Double scroll turbine
- Variable turbine for gasoline engine application
- Electric waste gate regulation





	Single component balancing	High Speed Balancing (HSB)
What	Turbine Wheel & Shaft Compressor Wheel	Centersection
Method	Rigid rotor	Flexible rotor
Aim	Reduction of initial unbalance of single components	Reduction of unbalance of rotor assembly
Measurement Speed	Single Speed between 4000 – 5000 RPM	Single speed or speed range within T/C speed range



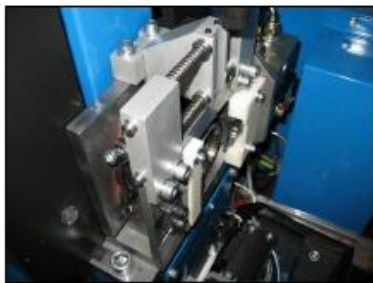
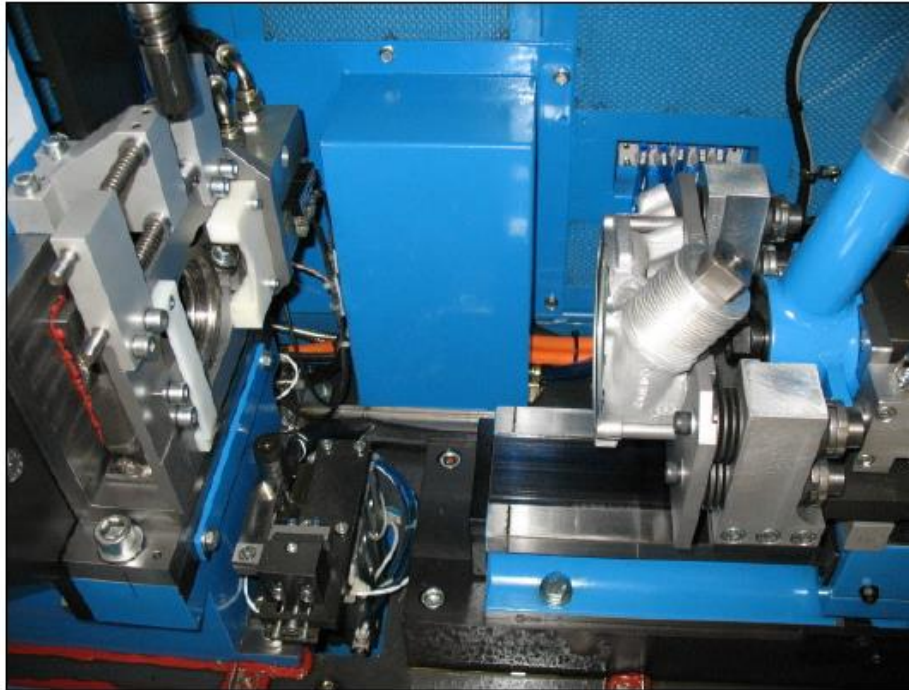
Lock Nut Plane

Sufficient balancing mass
for a robust process



Compressor Wheel Plane

Target conflict between production
requirements and stress reduction →
Optimization needed



Mounting jig



TC clamped in jig

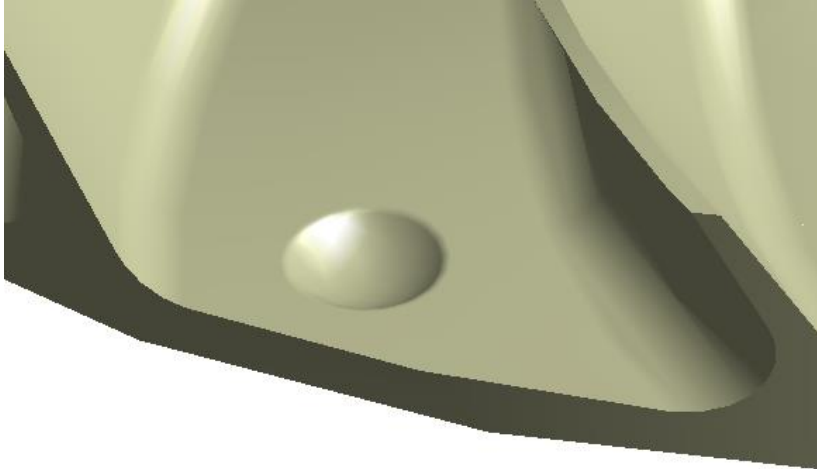


Acceleration sensor

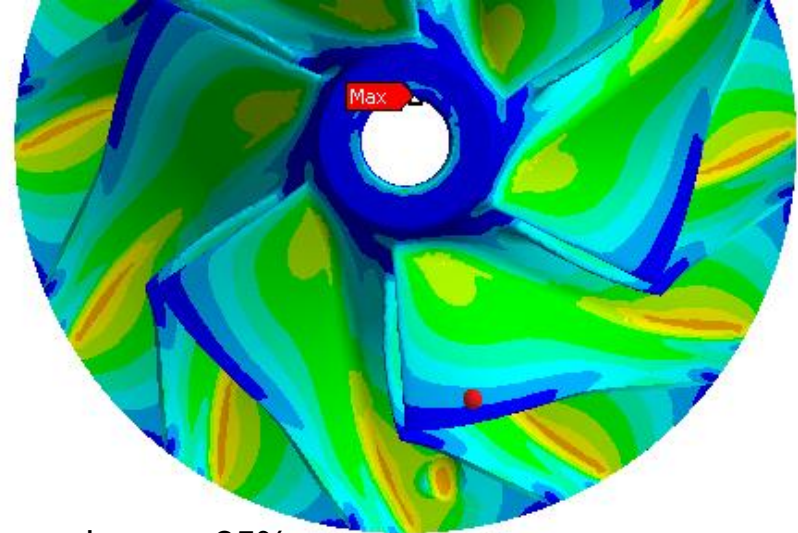


Speed sensor

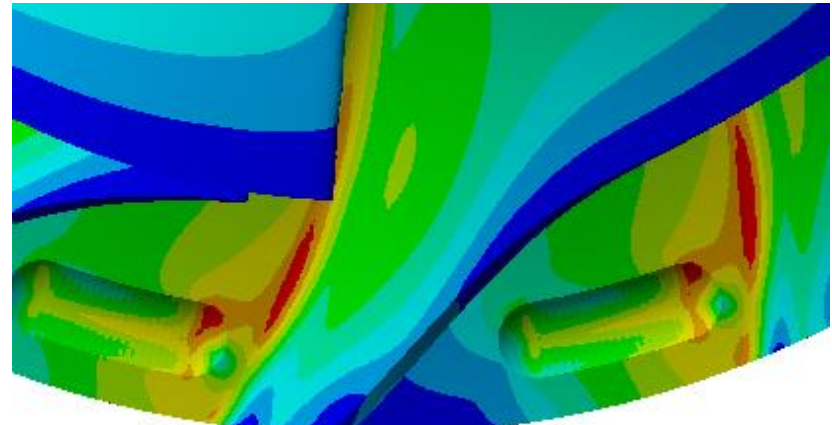
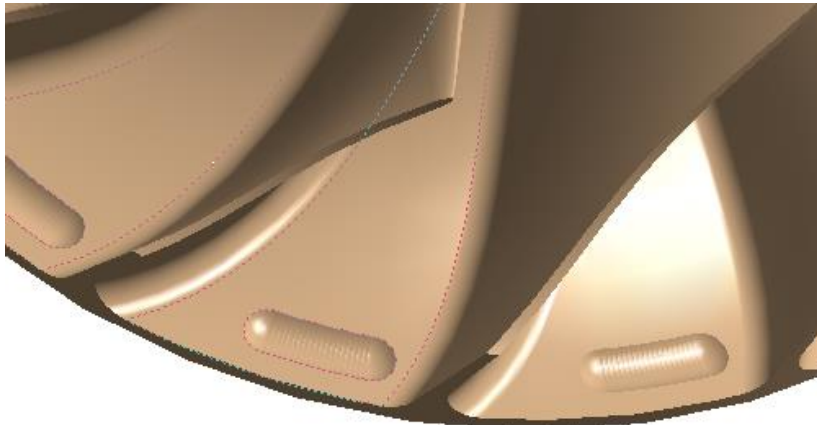
Balancing geometries



Stresses under centrifugal forces



Thick Backdisc: Balancing removal mass: 25%



Thick Backdisc: Balancing removal mass: 100%

Thin Backdisc: Balancing removal mass: **15%**

New balancing mark design required!

Shape optimization in 2017:

New compressor wheel geometry

- Optimized stresses
- Inertia reduction with thin backdisc → too small balancing mass removal capacity

New High Speed Balancing (HSB) mark required

- Development aim: maximizing balancing mark removal mass
- Side criteria: **Stress criteria**, production restrictions

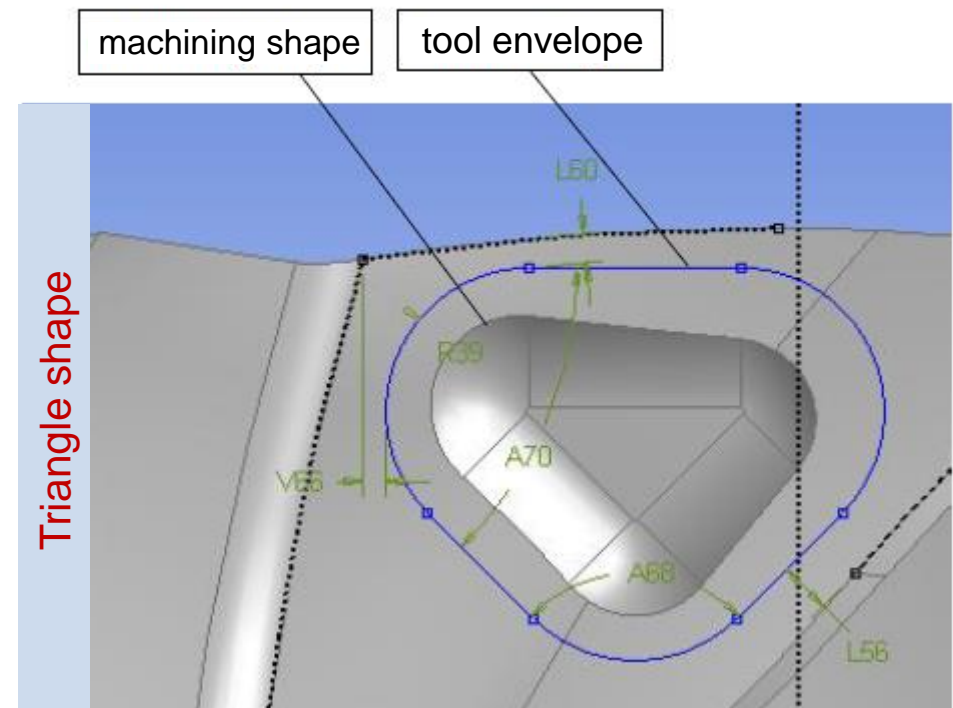
Boundary conditions:

Minimum wall thickness	
Milling tool distance to blade	
Distance milling geometry to outer radius	
Milling tool diameter minimum	
Realization of geometry by HSB machine	Radially symmetrical
Stress borders	To be fulfilled

Realizable mass removal approx 62,5%

Low acoustic borders on HSB → higher HSB scrap rate

→ **increase of balancing mass** required from production



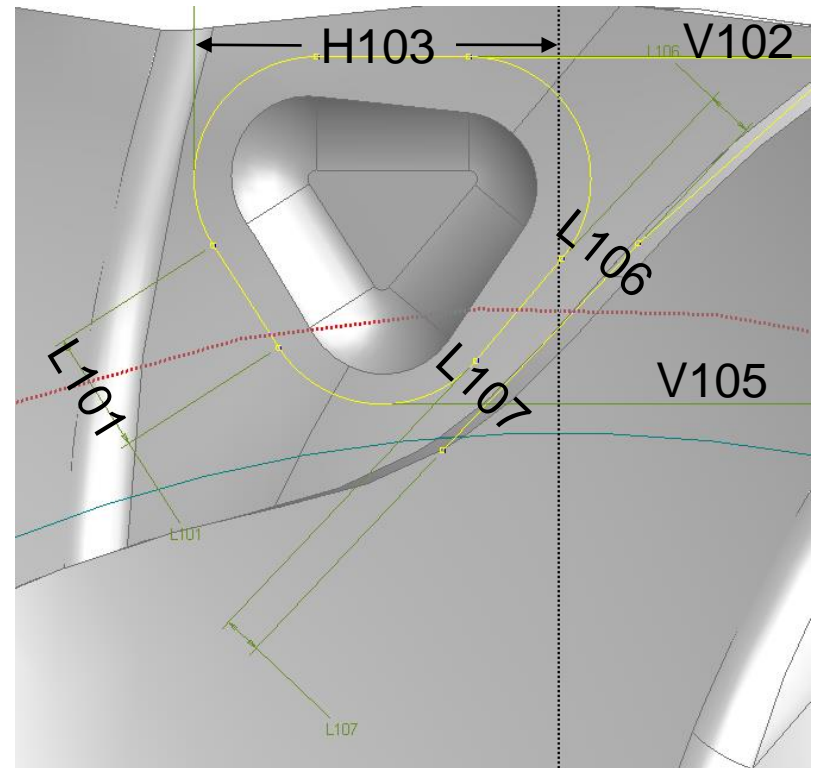
New optimization in 2019/2020:

Change of boundary conditions

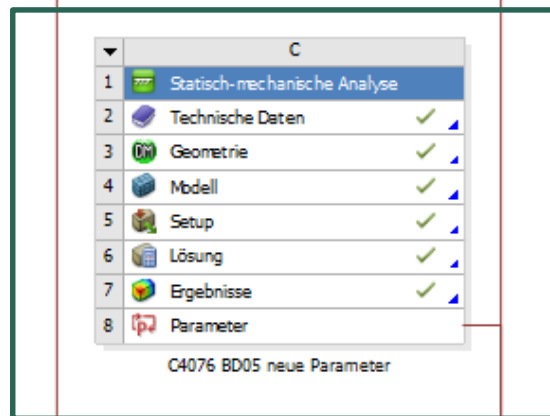
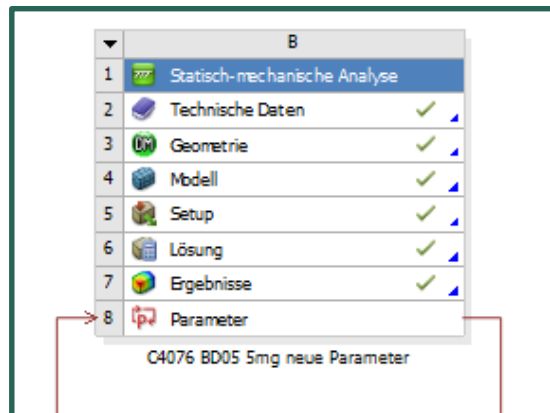
- Stronger acoustic HSB borders require higher HSB mass removal to keep scrap rate low
- Radially symmetrical geometry no longer a requirement → more DOF for shape of balancing mark

New parametrization:

Parameter	restriction
Depth milling tool	maximum
H103 – upper width	maximum
L101 – left length	
L106 – upper distance tool to blade	minimum
L107 – lower distance tool to blade	minimum
V102 – max radius balancing mark	
V105 – min radius balancing mark	



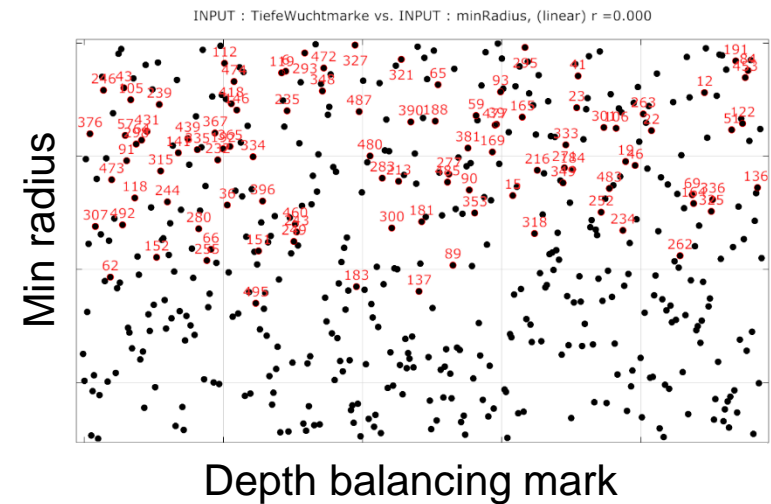
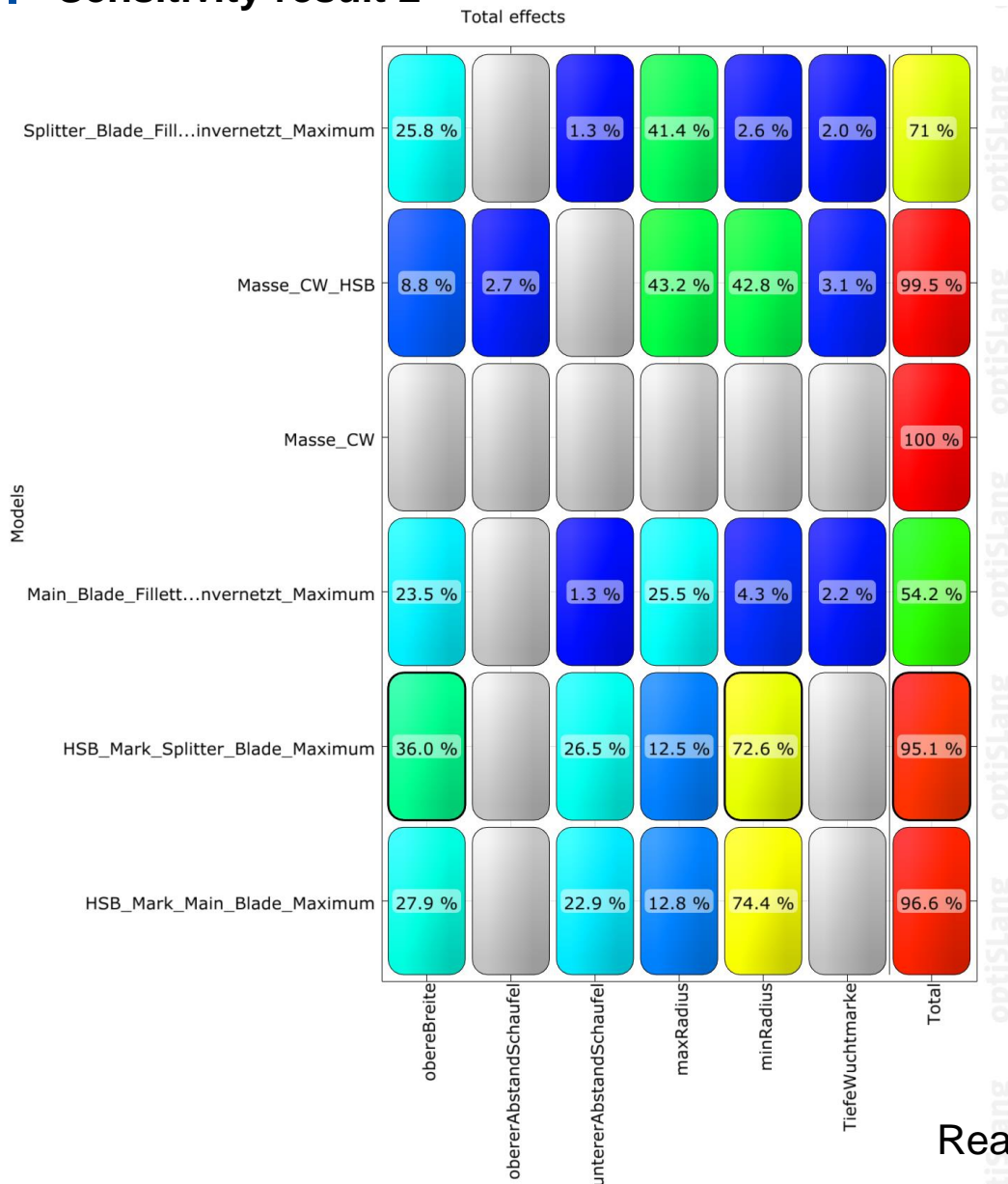
Workbench model:



Parametersatz

Eingabeparameter	Input parameter
P9	minDurchmesser
P10	maxDurchmesser
P11	obereBreite
P13	linkeLänge
P14	obererAbstandSchaufel
P15	untererAbstandSchaufel
P16	TiefeWuchtmarke
Neuer Eingabeparameter	Neuer Name
Ausgabeparameter	Output parameter
P4	
P17	HSB Mark 2 Maximum
P18	Fillet 1 Maximum
P19	Fillet 2 Maximum
P8 Reference mass	
Neuer Ausgabeparameter	
Diagramme	

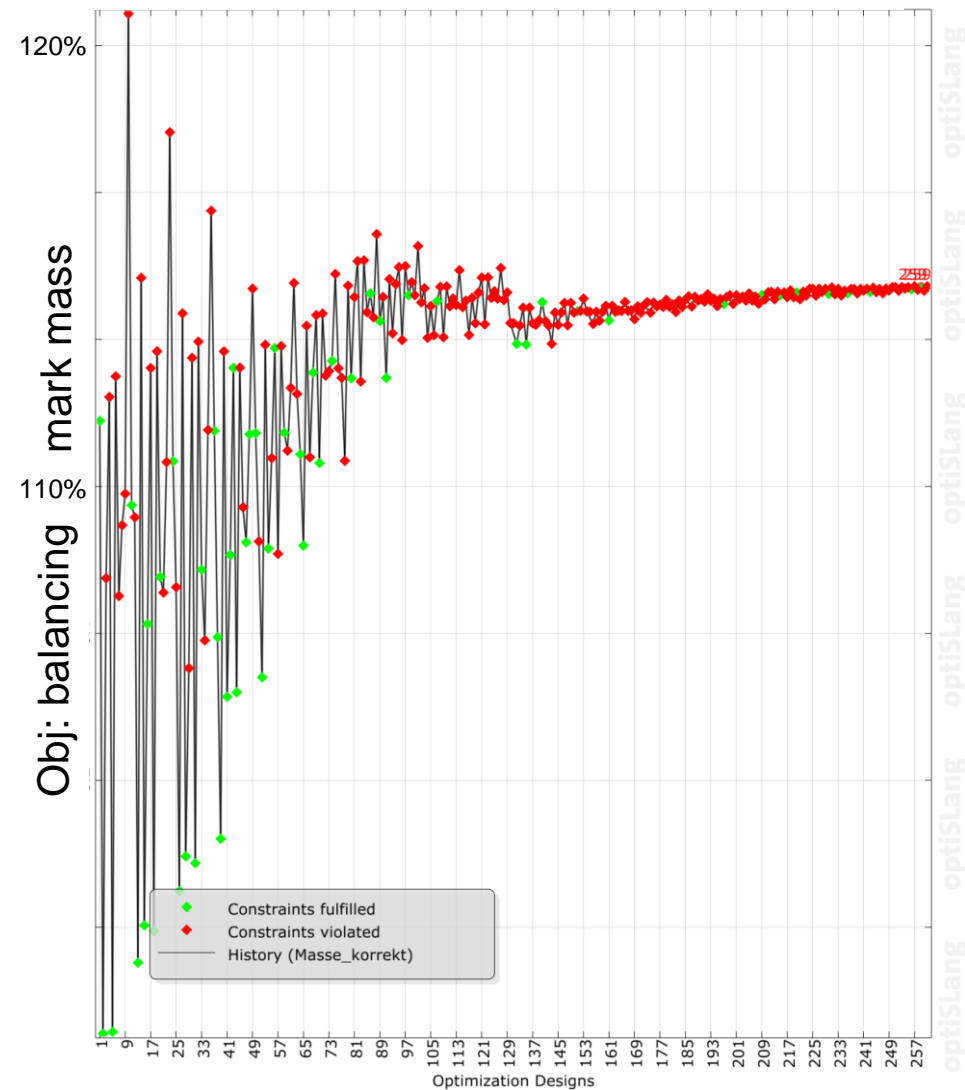
Sensitivity result 2



Realizable mass removal approx 105%.

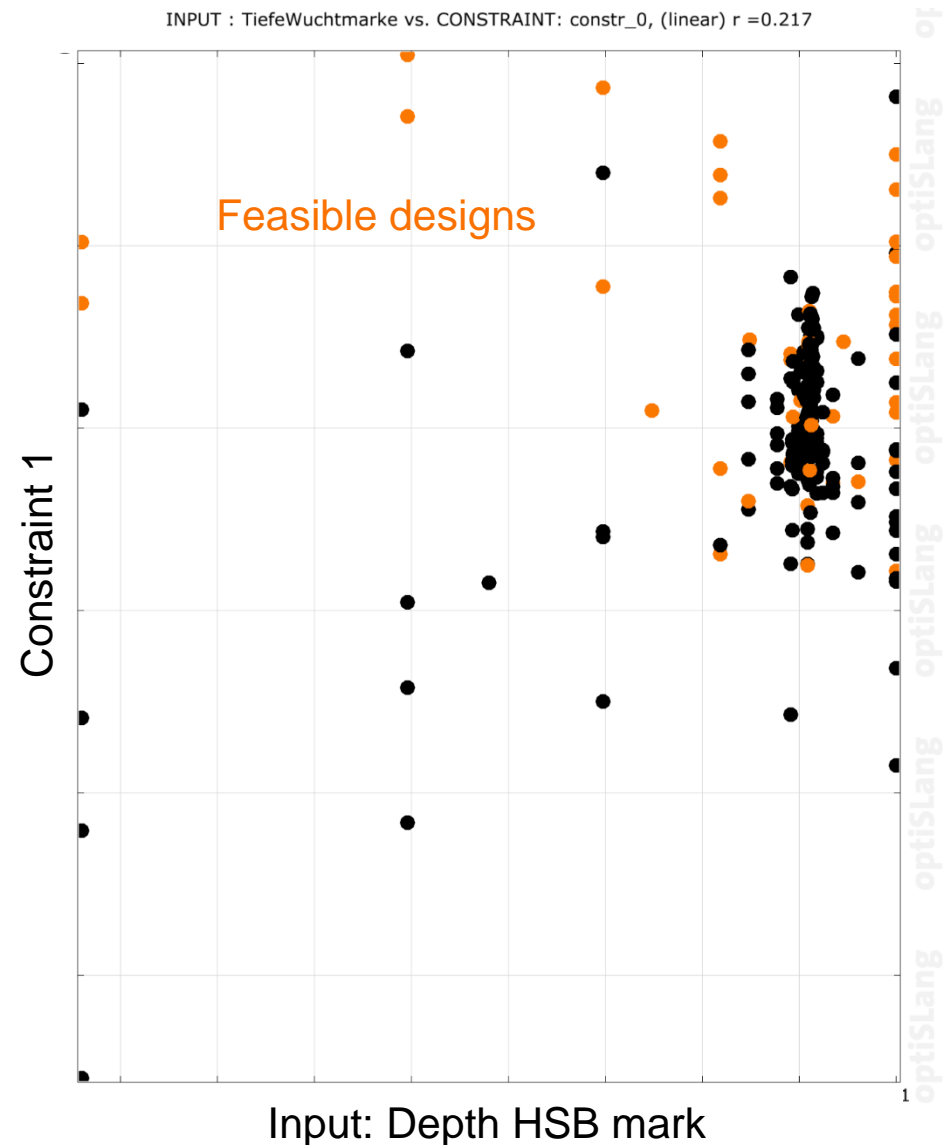
Direct optimization (ARSM):

Obj: balancing mark mass



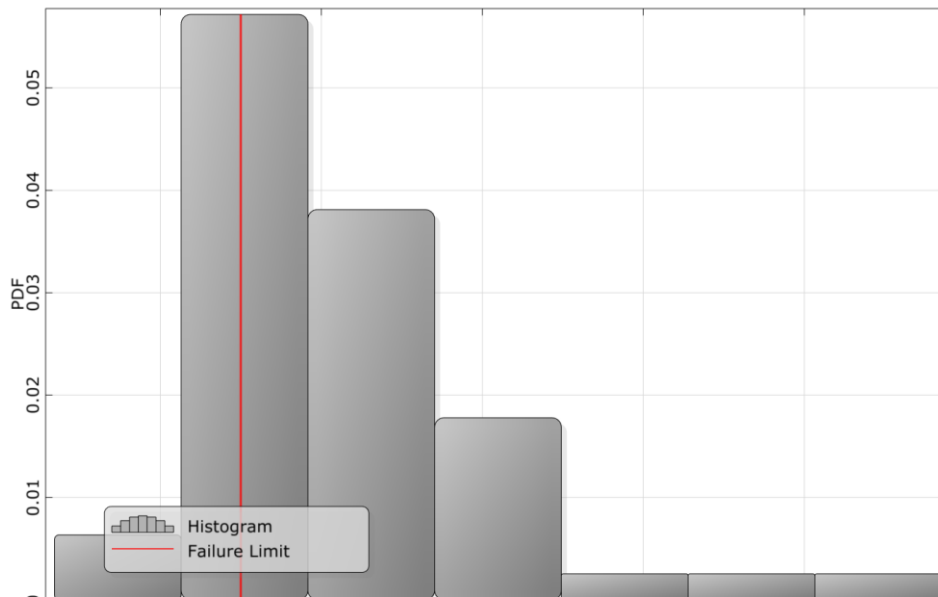
Realizable mass removal approx 114%.

INPUT : TiefeWuchtmarke vs. CONSTRAINT: constr_0, (linear) $r = 0.217$

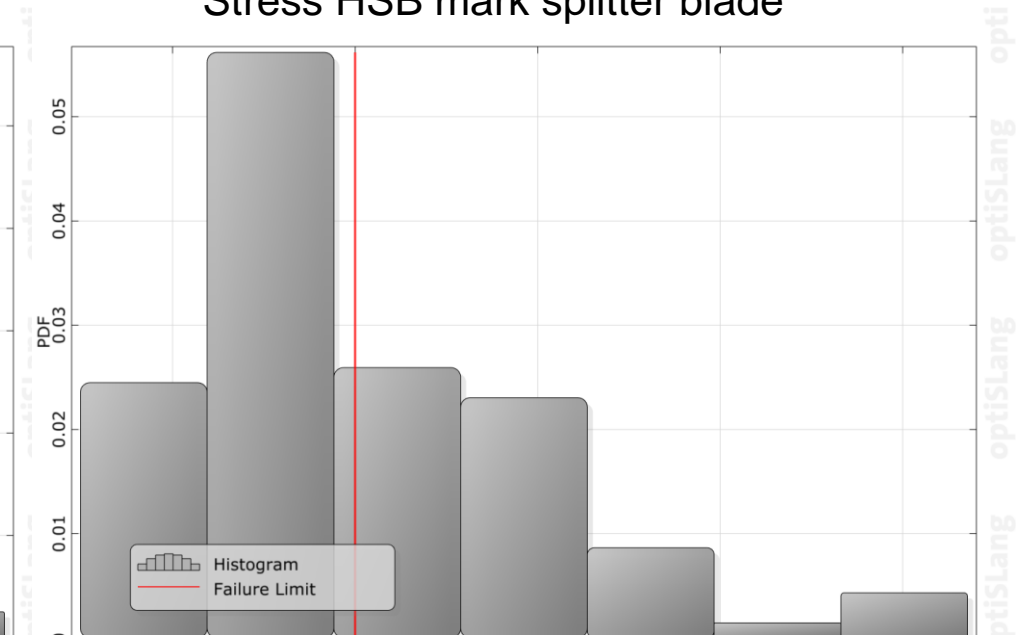


Robustness:

Stress HSB mark main blade



Stress HSB mark splitter blade



With normal distributed input values,
too many objectives above criterion.

Robustness shows increase in mass removal up to 120%, but with improvement potential for robustness.

→ Lower stress limits for optimization to find more robust optimized design.

Low balancing mass removal yields in higher balancing scrap rate

→ Increase in balancing mass removal required

Sensitivity, Optimization & Robustness study directly integrated in Workbench model

- Sensitivity shows increase in mass removal up to 105%
- Optimization shows increase in mass removal up to 114%
- Robustness shows increase in mass removal up to 120%, but with improvement potential – with loss of balancing mass removal

