

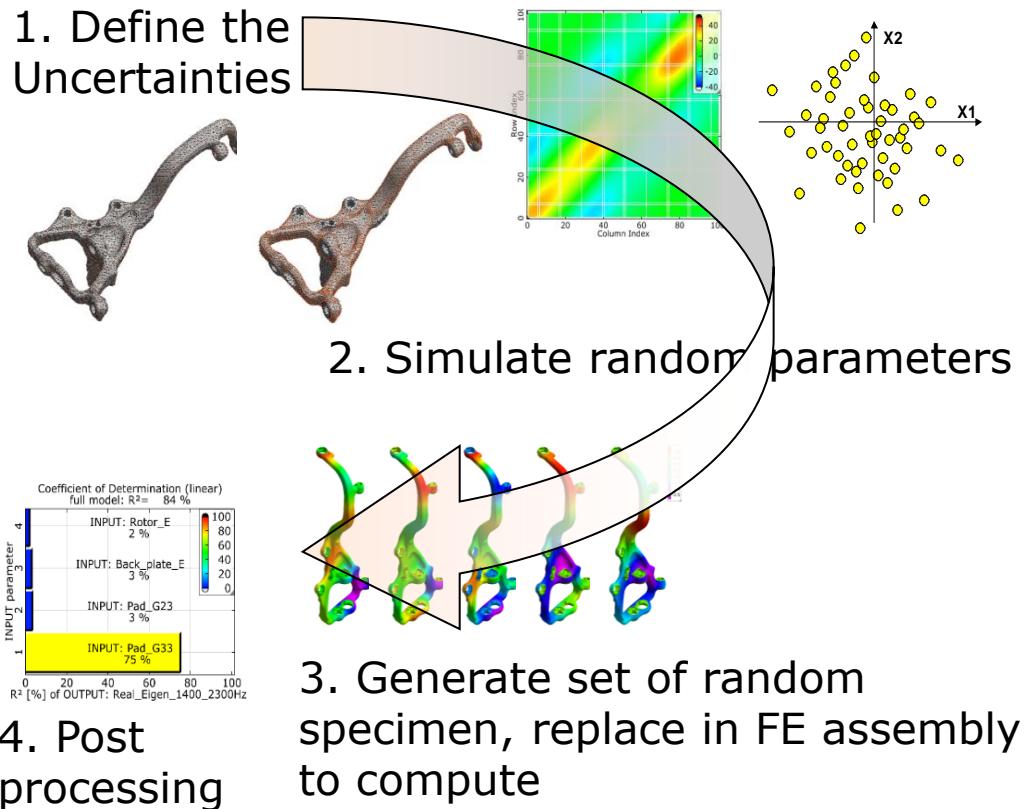
Robust Design Optimization examples automotive industry



RDO of brake systems

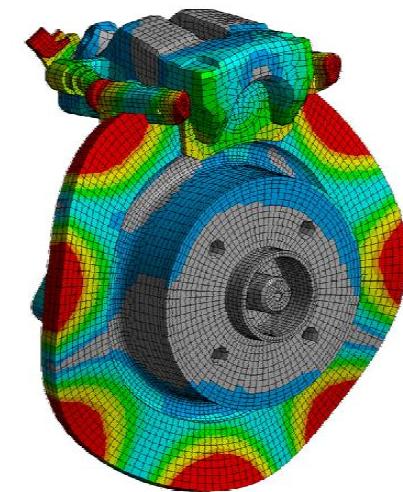
Investigating and optimizing the NVH performance of brake systems answering the question: How does material and geometric scatter influence the brake noise performance ?

- Consideration of material and geometry scatter
- CAE-Solver: NASTRAN, ABAQUS, ANSYS
- Up-to-date robustness evaluation have 20 ..30 scattering variables
- Started 2007 with Robustness evaluations
- Since 2010 extension to automatic robust design optimization using shape function for geometry optimization

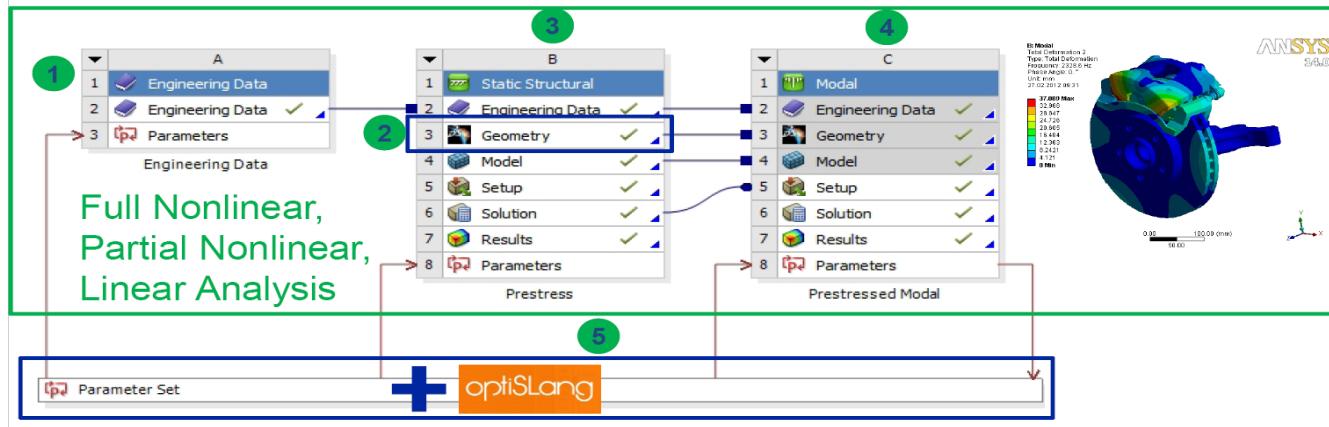


Modern Parametric Process for Brake-Squeal Simulation

- Brake is an **important and complex, safety and performance** component in automobile.
- NVH field **complaints and warranty costs** as well as permanently **increase in the customer requirements and targets**
- ANSYS Workbench parametric **geometry** modeling functionality helps in establishing a **Simulation Driven Product Development for robust Brakes**



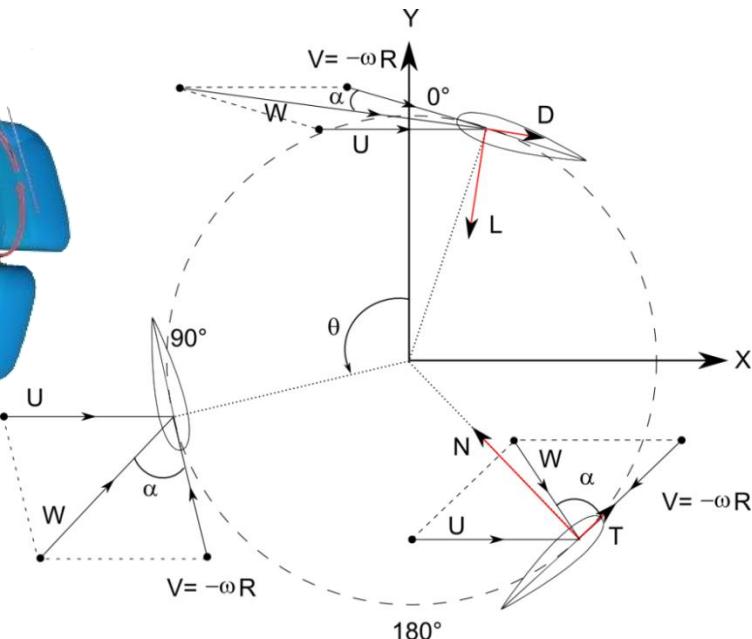
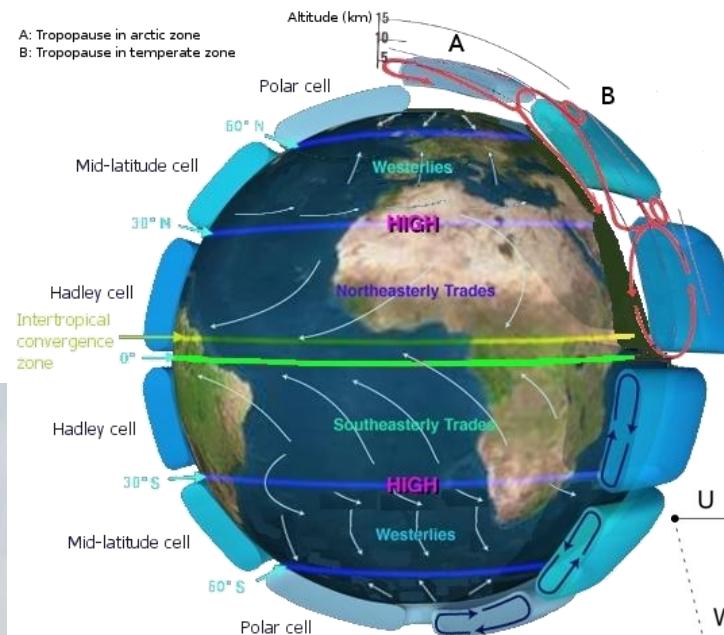
ANSYS Workflow for Parameterized Brake Squeal Simulation



Robust Design Optimization Turbomachinery



Introduction

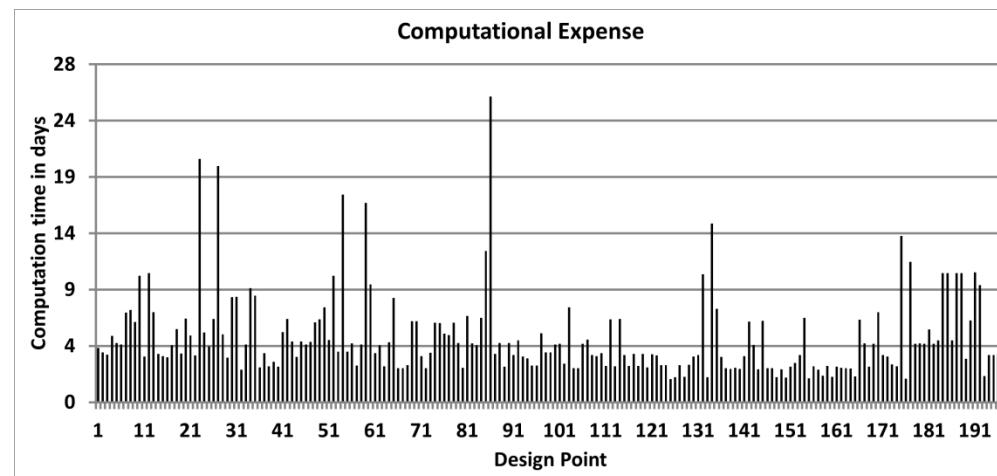


Advantages of Vertical Axis Wind Turbines (VAWT):

- Lower cost compared to horizontal axis wind turbines
- Independent of wind direction
- Good efficiency for lower wind speeds

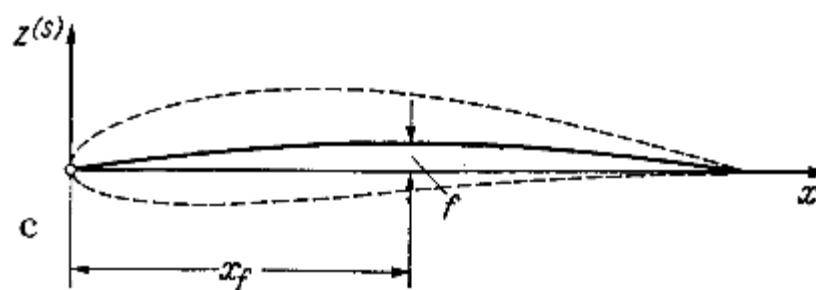
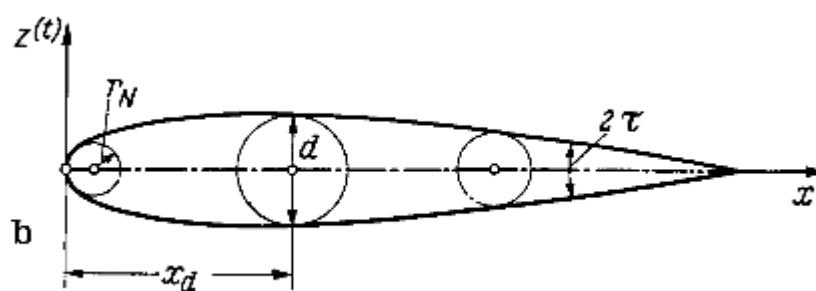
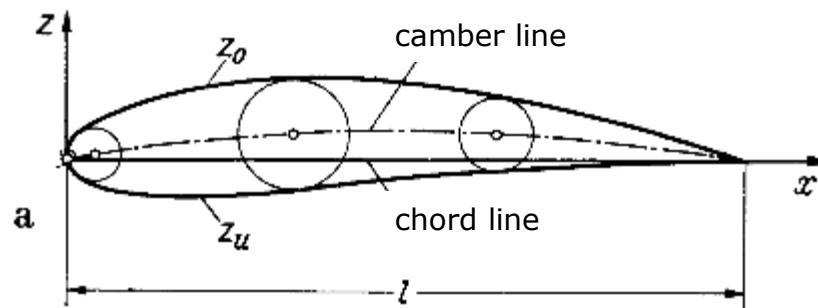
Motivation

- Prediction with analytical methods is very vague
→ **CFD approach needed**
- Unsteady flow → Transient simulations are required
- Large scale machinery → High grid size
→ **High computational expense**



Solution: **A high-quality metamodel admits efficient optimizations!**

Modelling

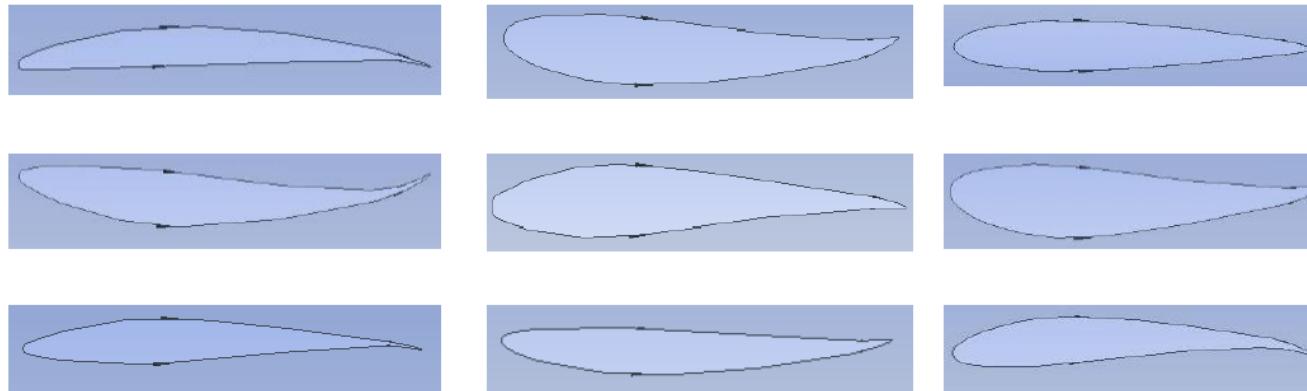


Parameterization according to
Aerofoil Theory:

- Relative thickness $\frac{d}{l}$
- Relative camber $\frac{f}{l}$
- Relative distance of maximum thickness from the leading edge $\frac{x_d}{l}$
- Relative distance of maximum camber from the leading edge $\frac{x_f}{l}$
- Relative leading edge radius $\frac{r_n}{l}$
- Relative trailing edge radius $\frac{r_h}{l}$

Modelling

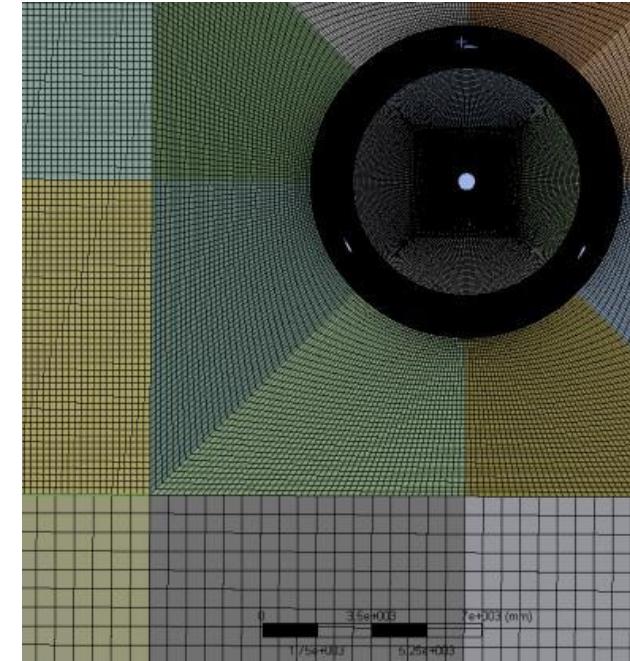
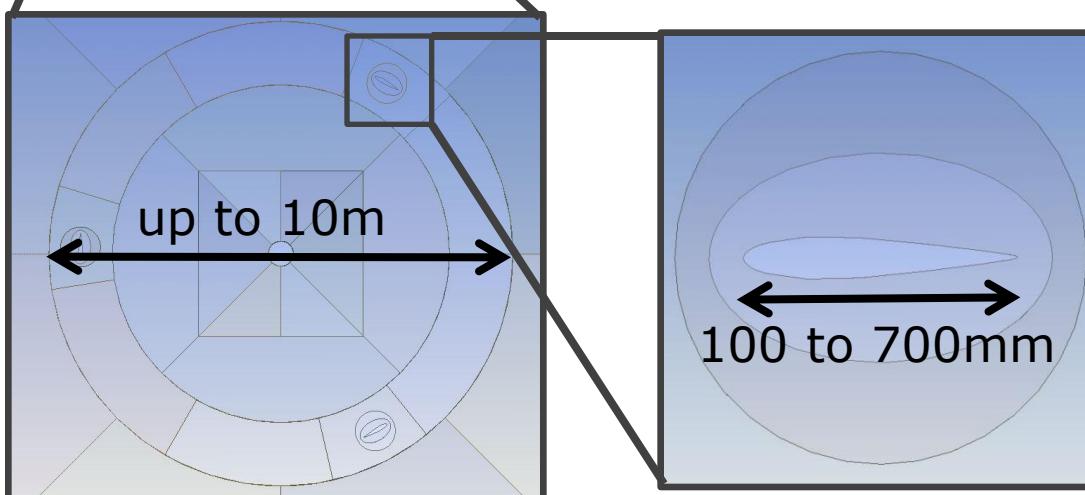
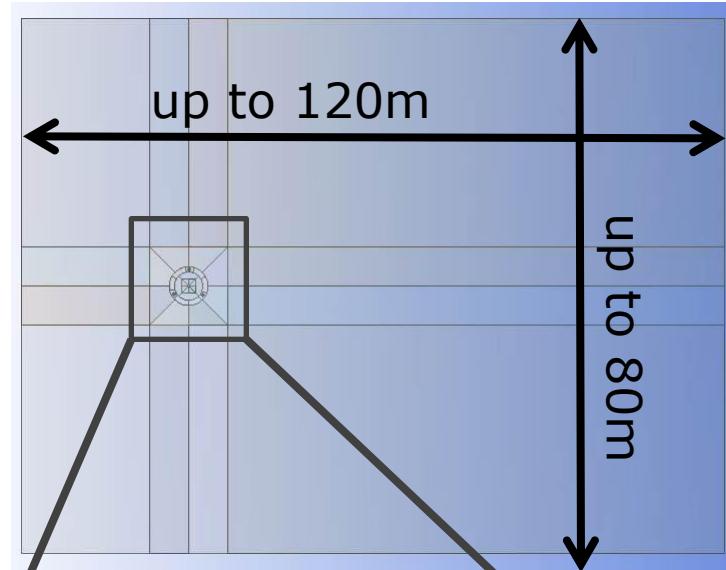
Variation of the aerofoil geometry



Further parameters:

- Wind turbine diameter
- Aerofoil size ratio
- Wind speed

Modelling



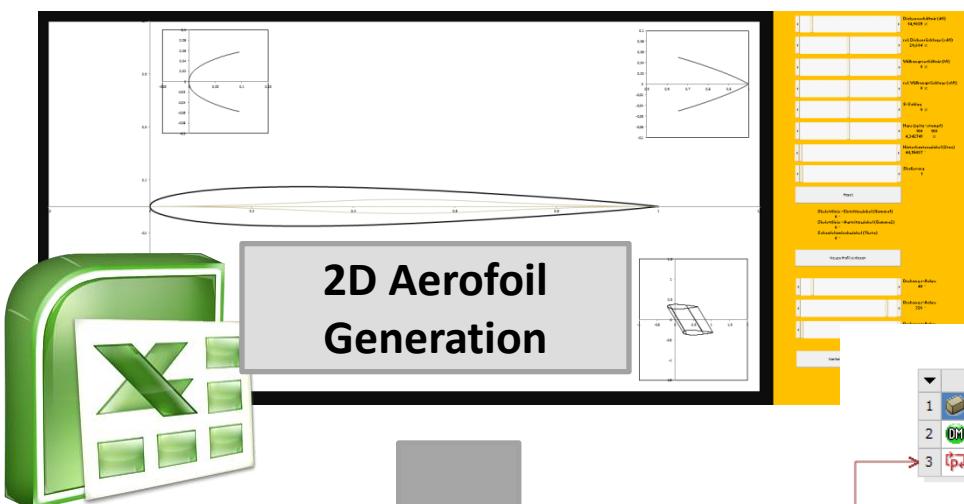
Different geometric scales have to be represented by the grid
→ High grid size

Process

Design Parameters



2D Aerofoil
Generation



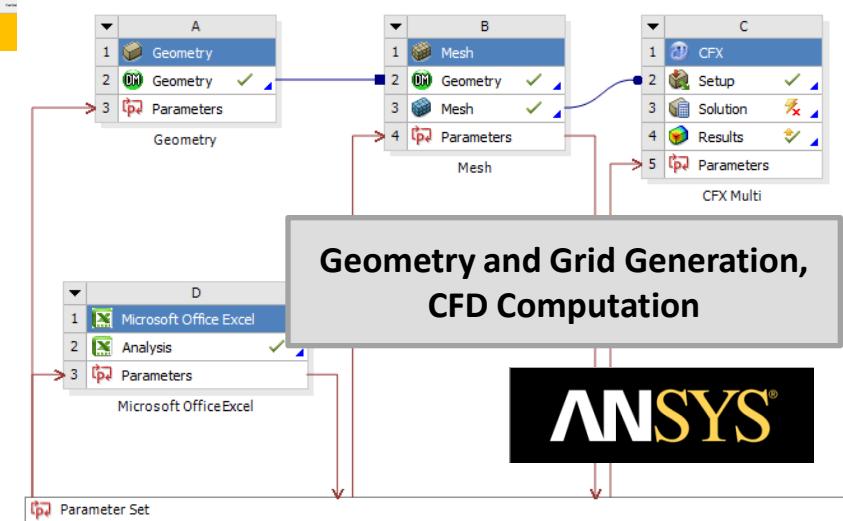
2D Aerofoil

Simulation computationally very expensive: 200 design at 25 workstations running 3 weeks.



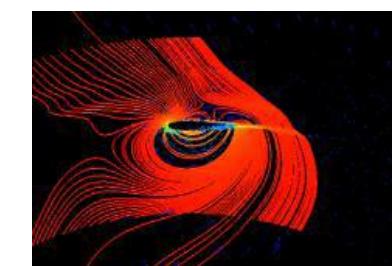
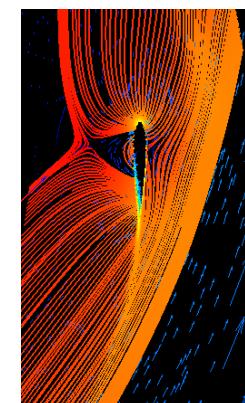
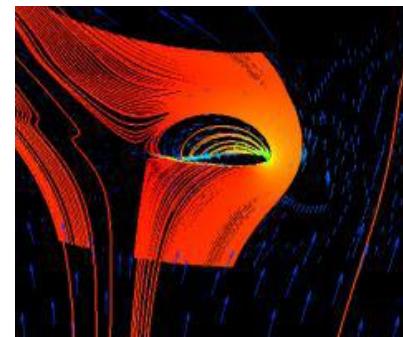
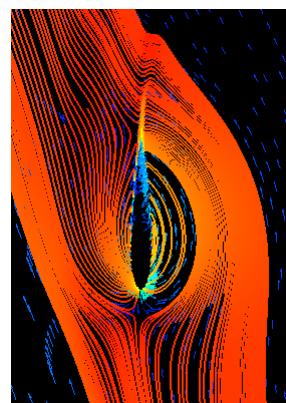
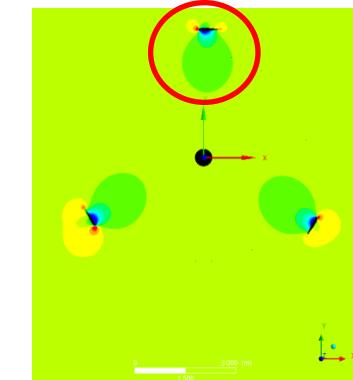
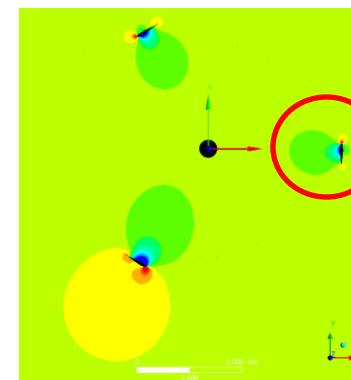
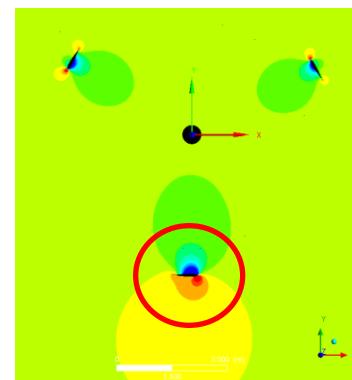
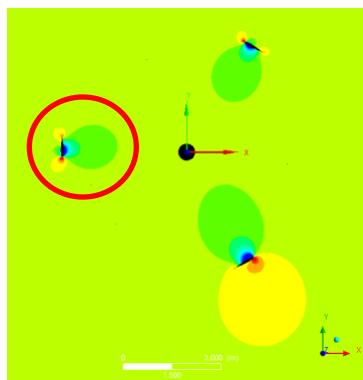
optiSLang®

Results



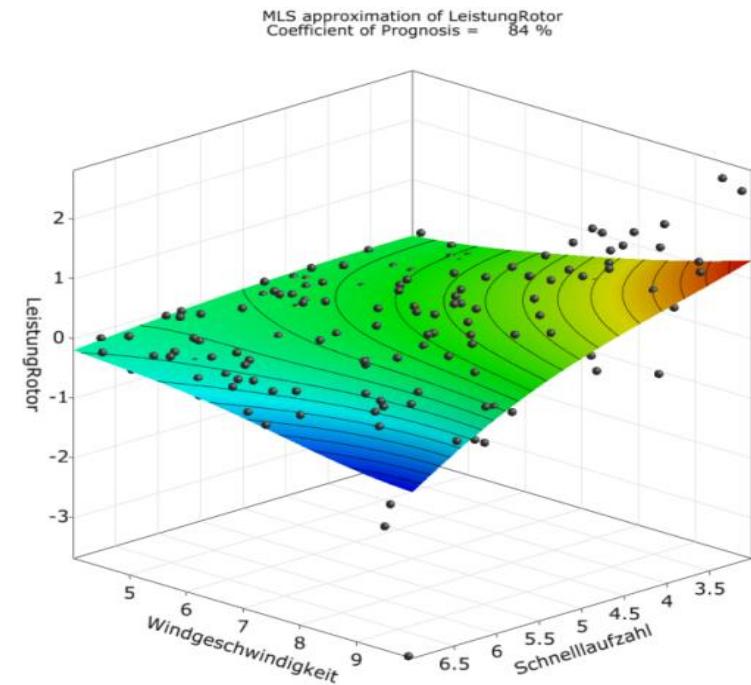
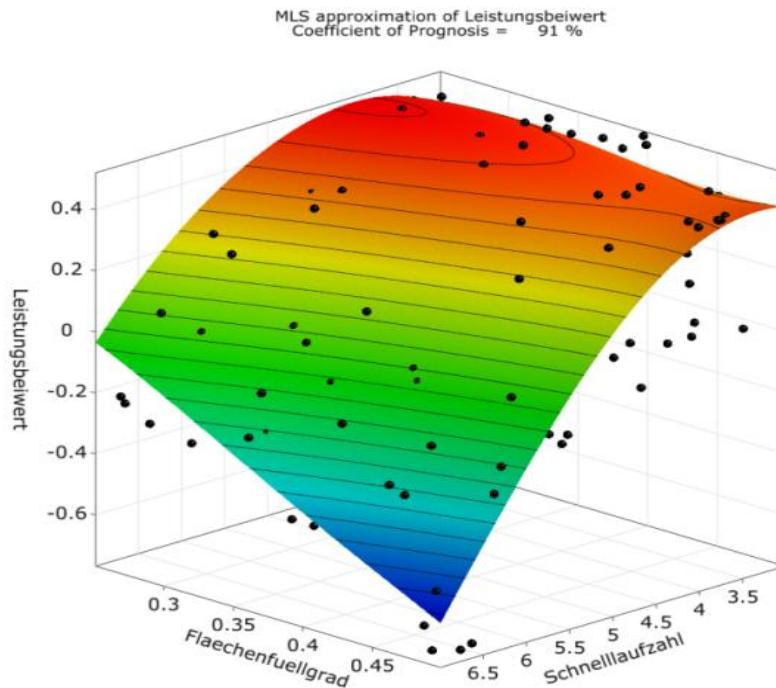
Results

Flow field for one rotor revolution



Results

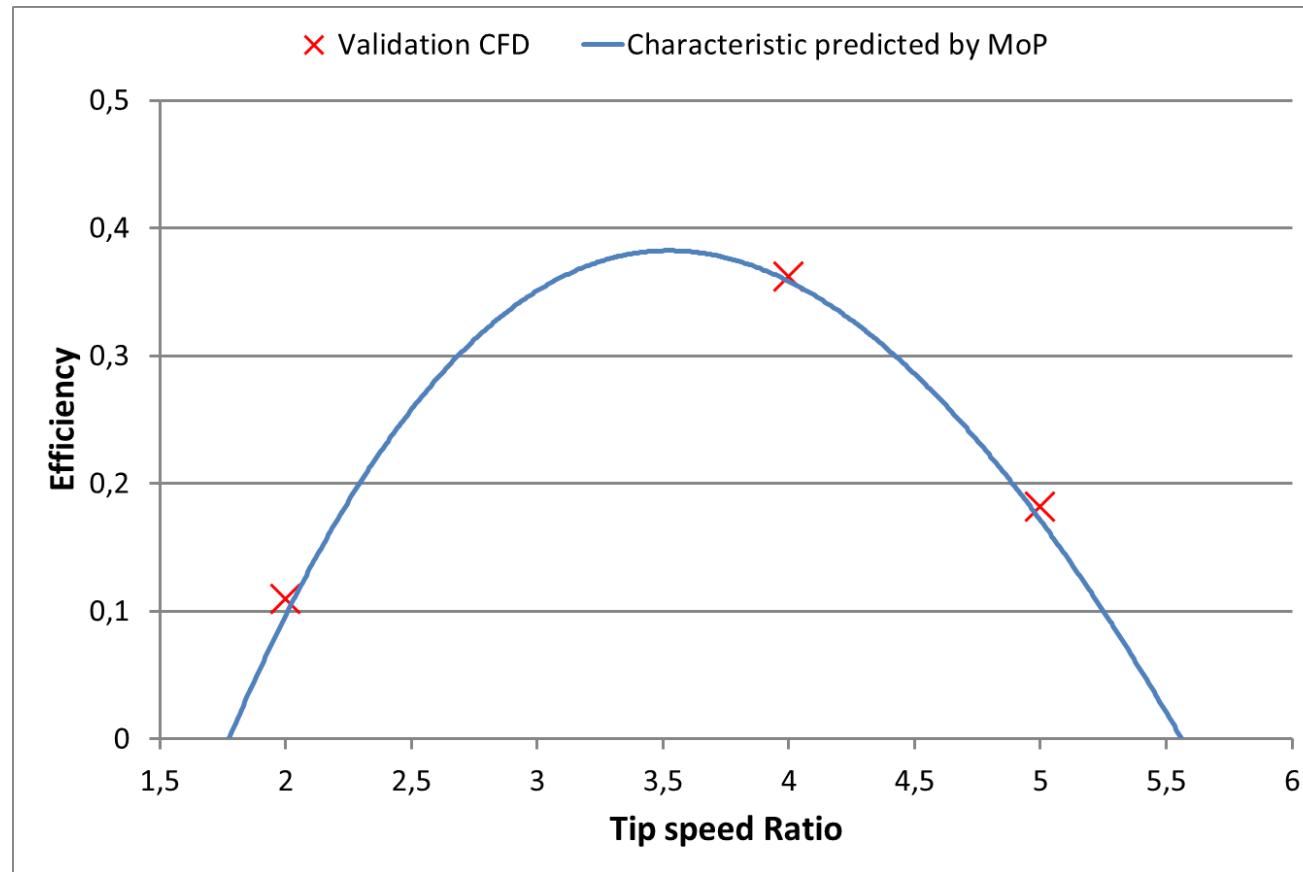
MLS – Approximation of the Efficiency / Power Output based on the Meta Model of optimal Prognosis (MoP)



High CoP for both Efficiency and Power Output
(CoP = 91%) (CoP = 84%)

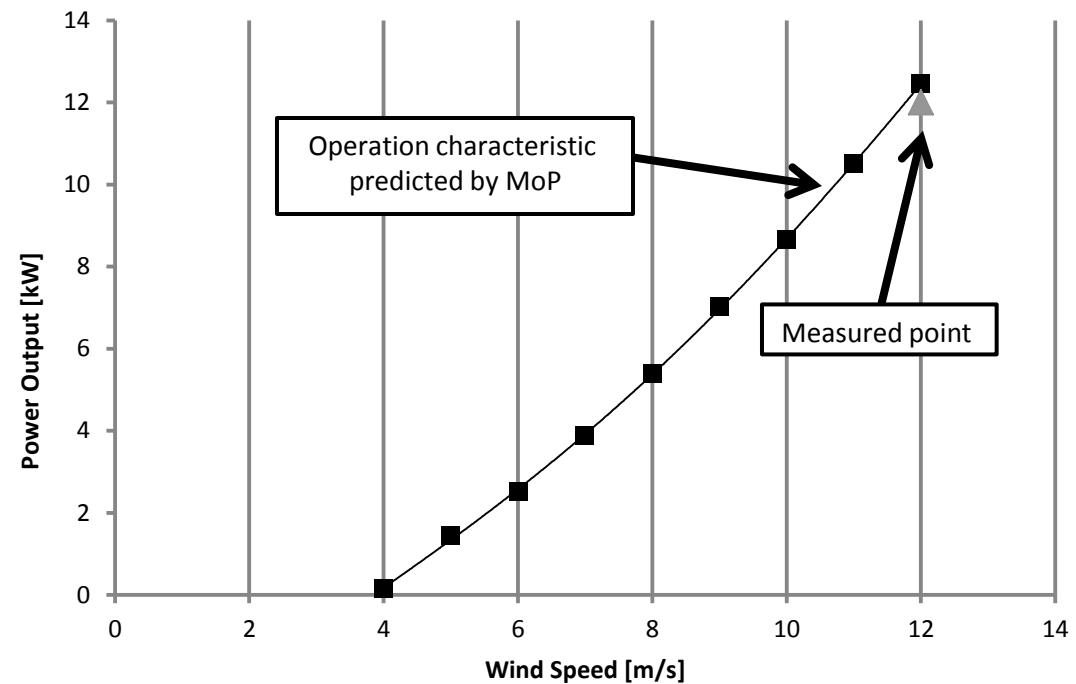
Results

Validation of the metamodel against CFD calculations



Results

Validation of the metamodel against an existing Machine



Marsta-VAWT – University of Uppsala

Summary

- 200 CFD Designs calculated
- Metamodel with high CoP
- Metamodel validated with additional CFD caluclations
- Metamodel validated against existing machines
- Operation characteristics of arbitrary machines can be predicted natively in optiSLang
without further CFD Simulations!

Multi-Physics Design Optimization of an Axial Compressor

Application and Best-Practice Guide-Lines



Fluid Dynamics

Structural Mechanics



Electromagnetics



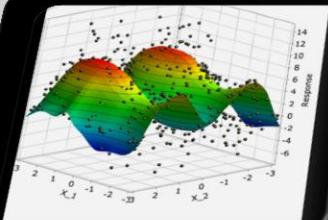
Systems and Multiphysics

Johannes Einzinger, ANSYS

Overview



Simulation Model, Best-Practice CFD and FEM

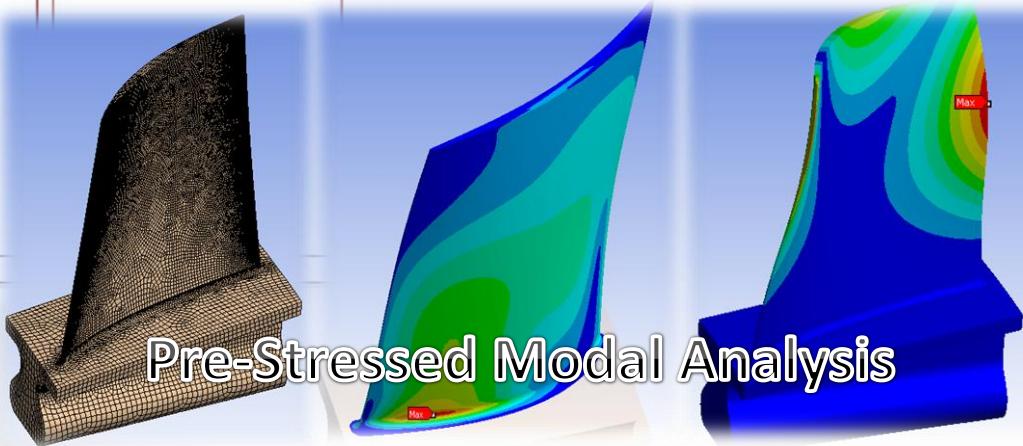
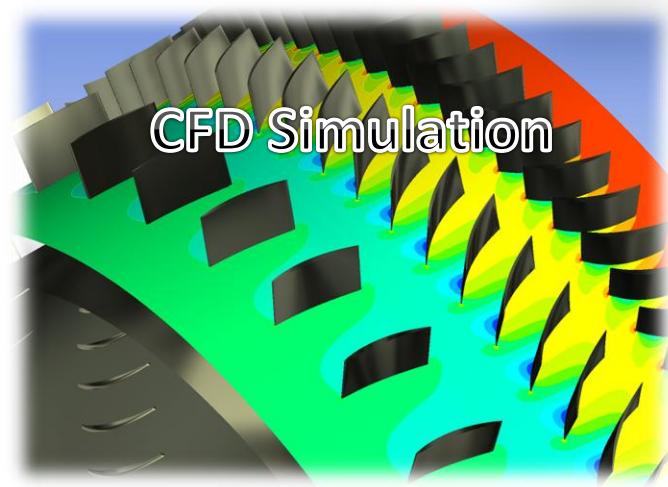
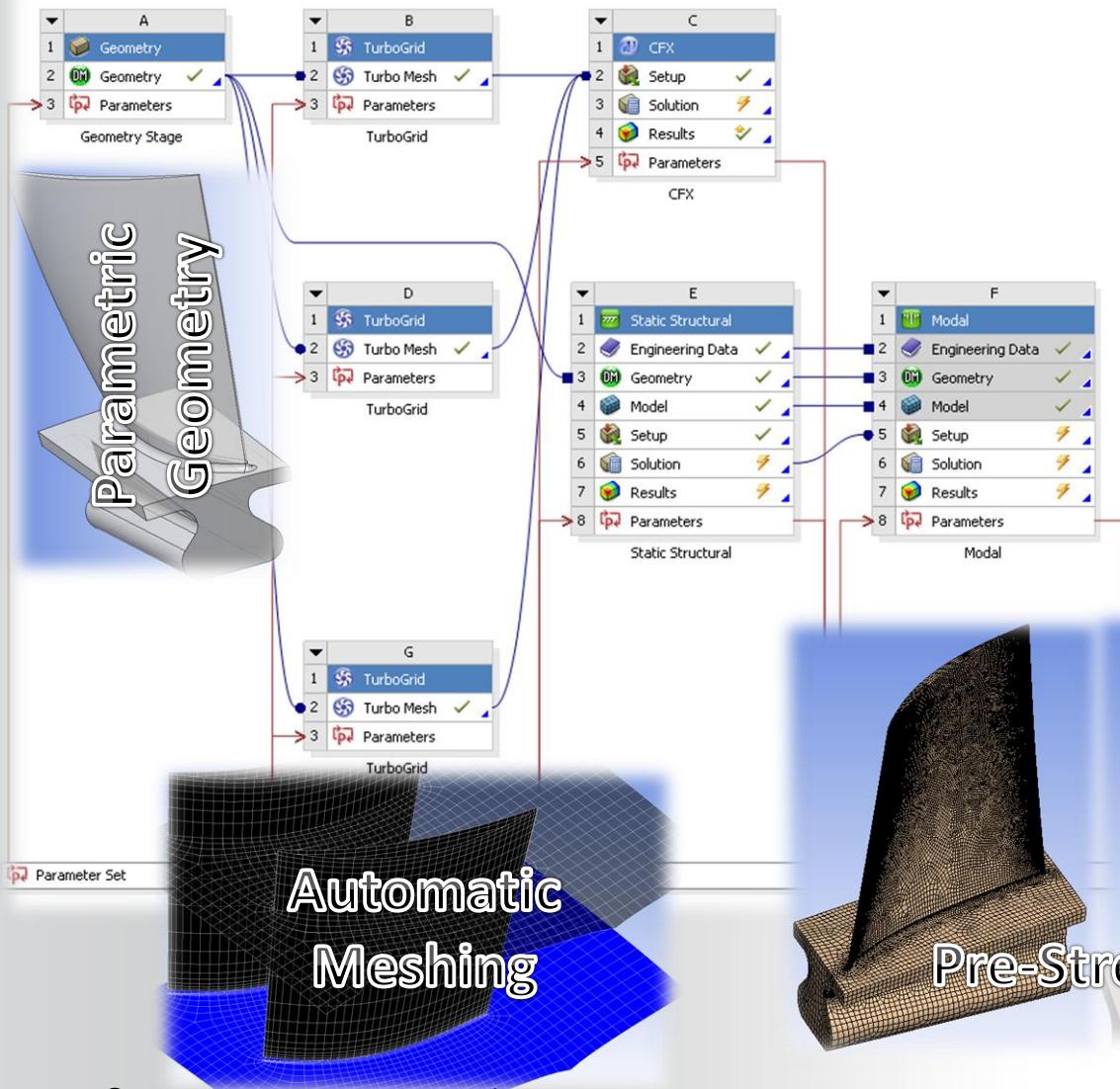


Meta-Model of optimal Prognosis



Best-Practice for MoP

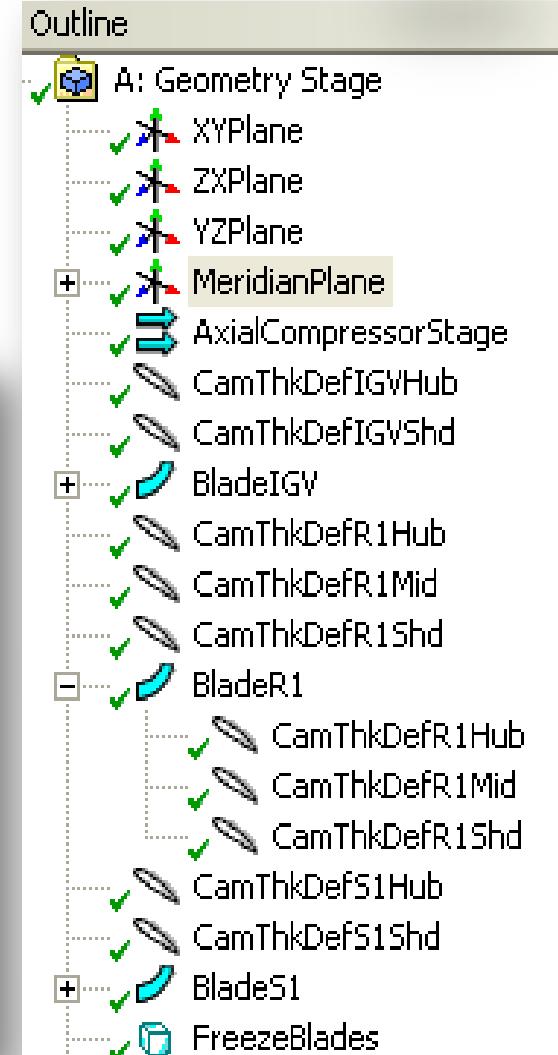
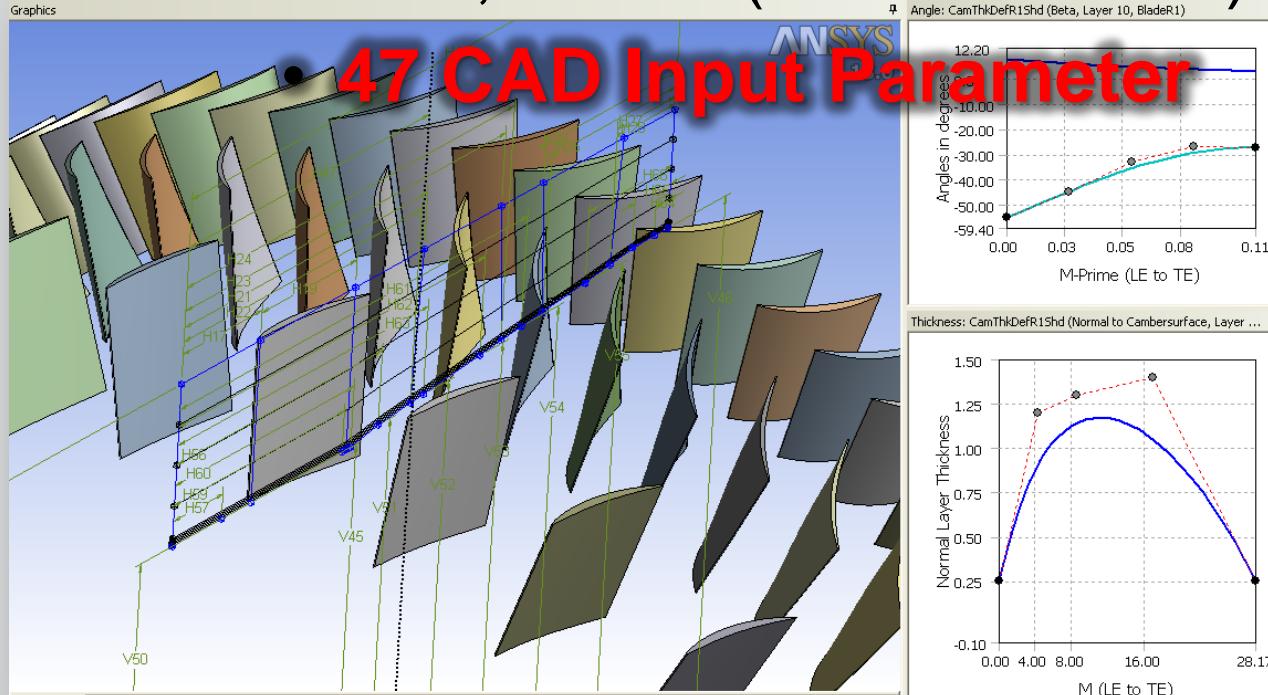
Application Overview



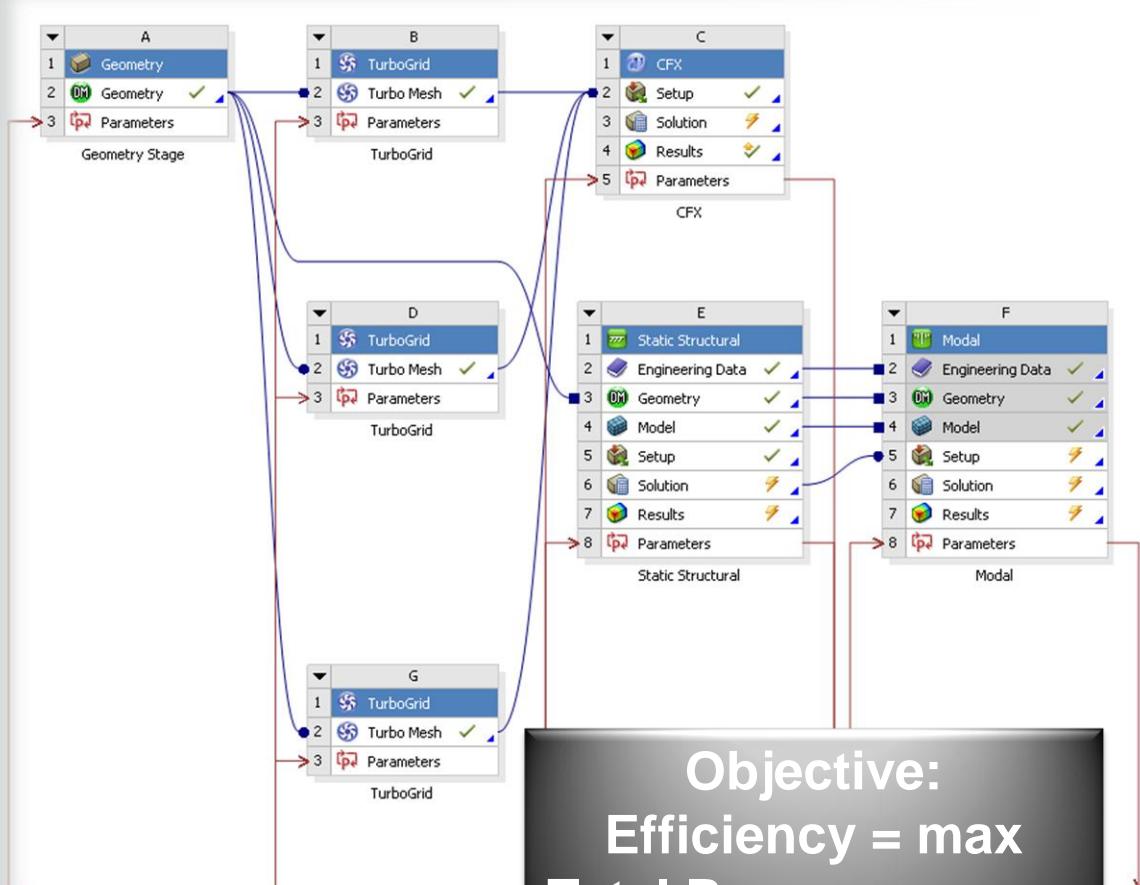
Geometry, Aero Dynamic



- Camber/Thickness for
 - IGV, R1, S1; 2-3 Layers
 - 5 β_i per Layer, 3xThk
 - Hub, 8 radii (const. Shroud)



Process and Objectives



Objective:
 $\text{Efficiency} = \max$
 $\text{Total Pressure} = \max$
 $\text{Stress} < \text{Limit}$
No Resonance

47 (59) Input Parameter

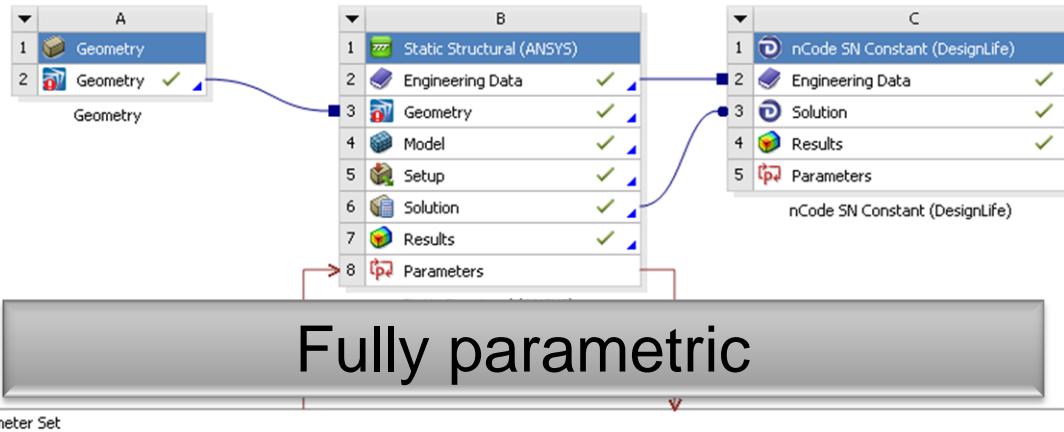
+	Geometry Stage (A1)	nPitchS1
+	TurboGrid (B1)	nPitchR1
+	TurboGrid (D1)	nPitchIGV
+	TurboGrid (G1)	myAirCP
+	CFX (C1)	myAirR
+	P16	myomega
+	P15	mymass
+	P14	Ttin
+	P17	P22
+	P18	ptin
+	P19	Face Sizing Element Size
+	P20	Mesh Max Size
+	P21	Mesh Min Size
+	P22	Mesh Max Face Size
+	Static Structural (E1)	Rotational Velocity Z Component
+	P89	ViewExpand ARG1
+	P90	Density
+	P91	Young's Modulus
+	P92	Poisson's Ratio
+	P93	New input parameter
+	P94	New Name
+	P111	Output Parameters
+	P112	Charts
+	P113	
+	Modal (F1)	

11 Input Constraints

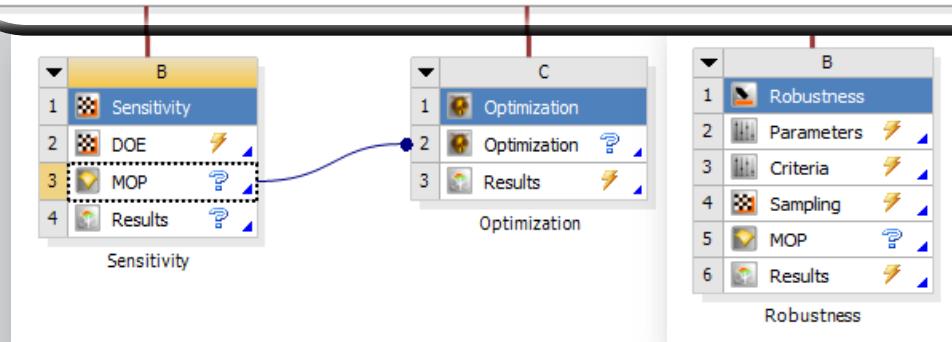
24 Output Parameter

optiSLang inside Workbench

The Workbench Effect – easier to use



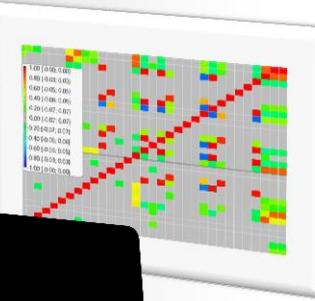
Easy parametric
set up of complex
simulations



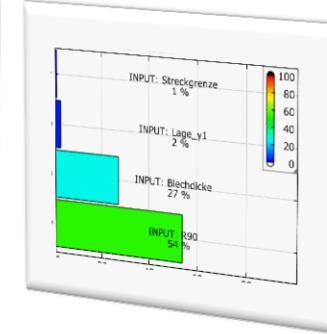
easy use of best praxis automated
flows inside Workbench

Meta-Model of Optimal Prognosis, MoP

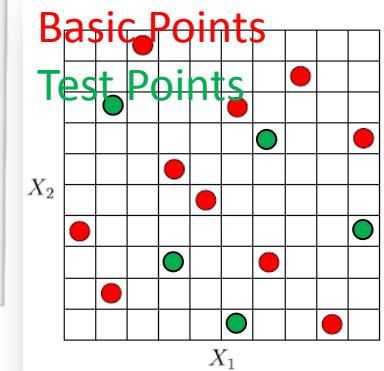
Significance Filter



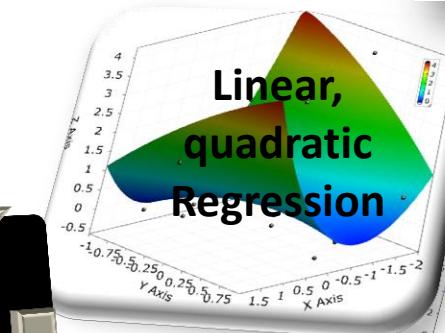
Importance Filter



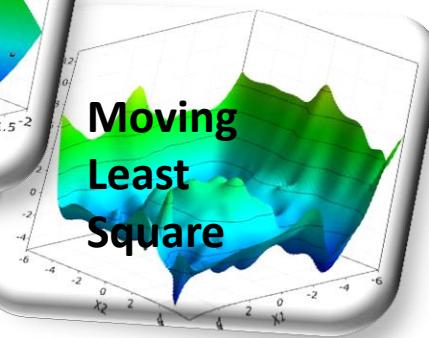
Test-Data Point Split



Response Surface



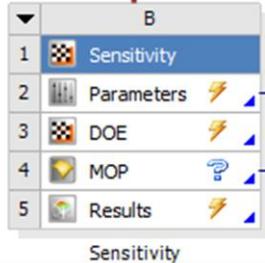
Coefficient of Prognosis



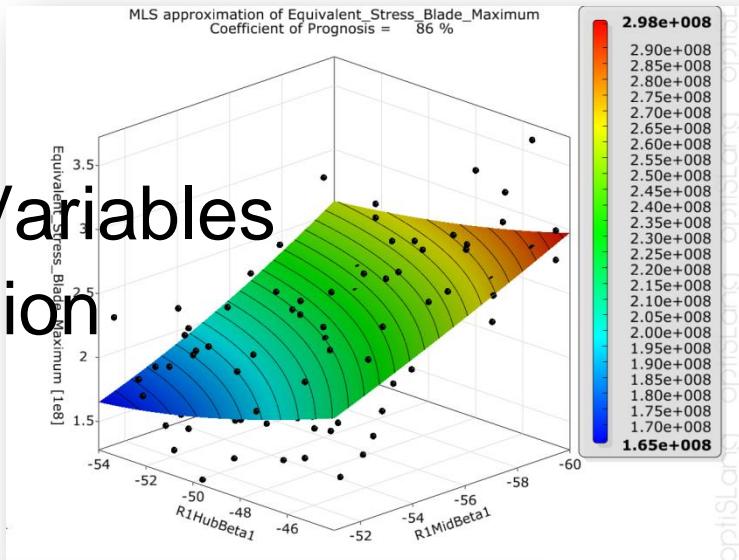
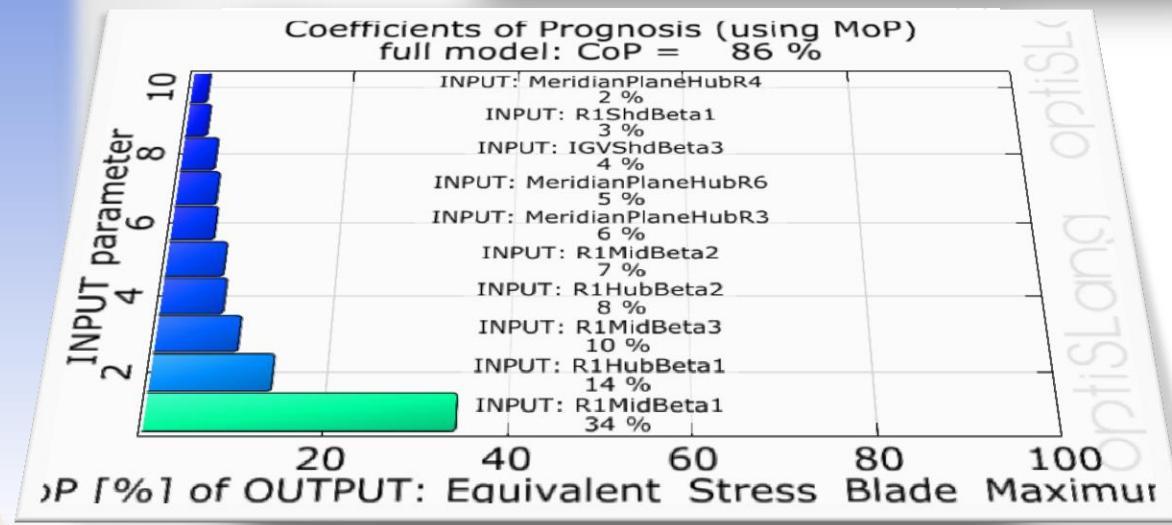
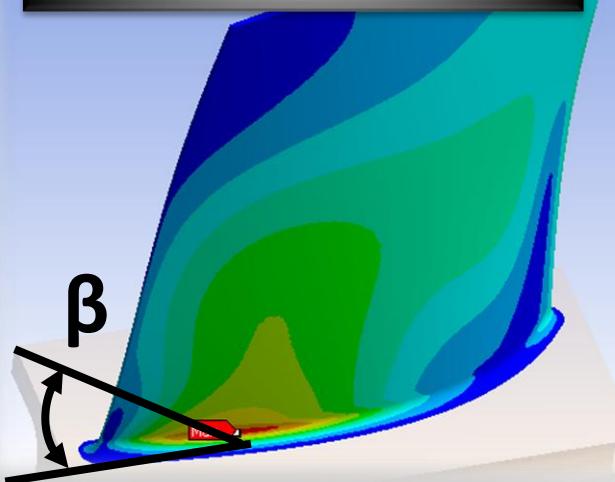
$$CoP = \left(\frac{E(Y \cdot \hat{Y})}{\sigma_Y \cdot \sigma_{\hat{Y}}} \right)^2 = \left(\frac{\sum_{k=1}^N (y^{(k)} - \mu_y) \cdot (\hat{y}^{(k)} - \mu_{\hat{y}})}{(N-1) \cdot \sigma_Y \cdot \sigma_{\hat{Y}}} \right)^2$$

Maximal Stress

- CoP=86%
 - Statistic is reliable
 - Detect important Variables
 - Parameter Reduction
- MoP is plausible

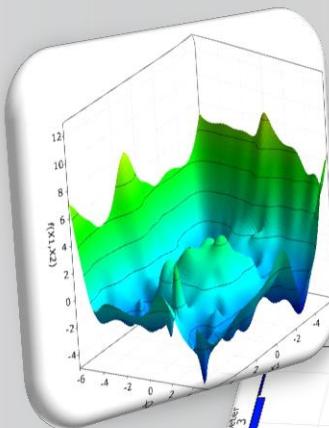


Blade Angle: Hub,
Mid Leading Edge

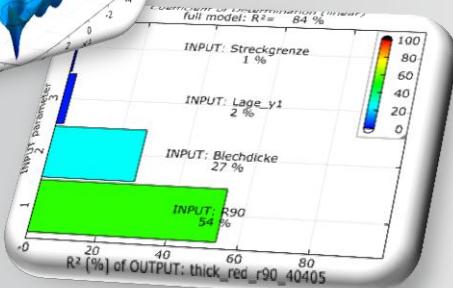


optiSLang Strategy

optiSLang
optimizing structural language



Quality of Response Surface Approximation



100%

Coefficient of Prognosis

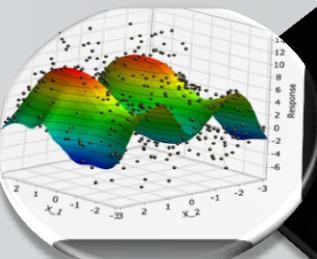
0%

Optimization on Response Surface

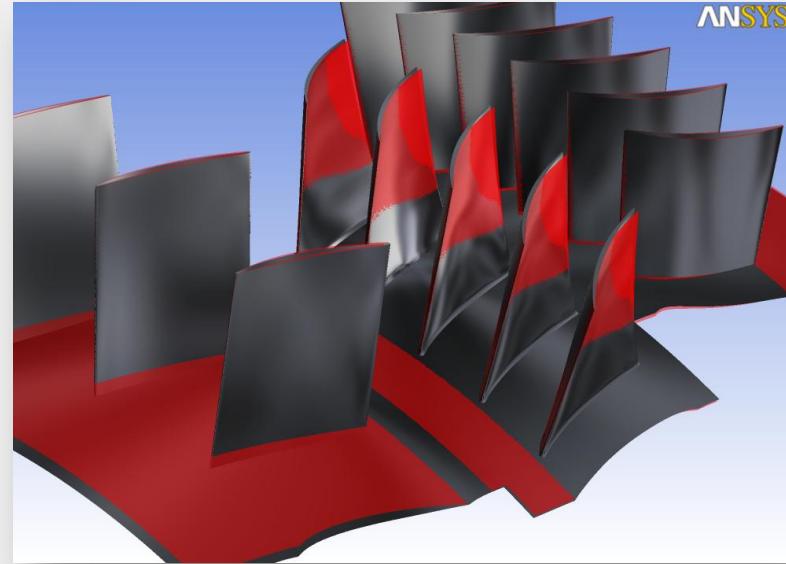
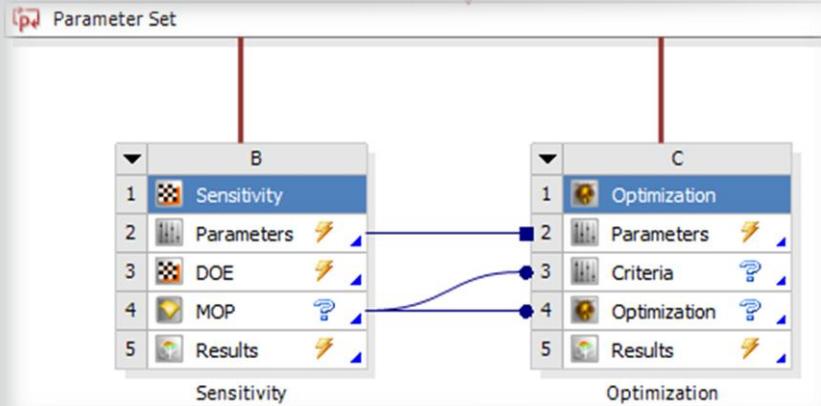
Design Optimization

New MoP in reduced Parameter Sub-Space

Meta-Model of optimal Prognosis



Design Optimization, Summary



	Initial Design	Best Design SA	Best Design Solved (MoP)	Best Design ARSM
Efficiency [%]	87.0	88.0	88.9 (91.0)	88.9
p _{tot} Ratio [-]	1.41	1.41	1.41 (1.44)	1.41
Max. Stress [MPa]	219	235	232 (230)	239
#Designs	1	150	1 (0)	100